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Gamero, P.; Cantero-Chinchilla, FN.; Bergillos, RJ.; Castro-Orgaz, O.; Dey, S. (2022). Shallow-water lee-side waves at obstacles: Experimental characterization and turbulent non-hydrostatic modeling using weighted-averaged residual equations. *Environmental Modelling & Software*. 155:1-16. <https://doi.org/10.1016/j.envsoft.2022.105422>



The final publication is available at

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Additional Information

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# Shallow-water lee-side waves at obstacles: Experimental characterization and turbulent non-hydrostatic modeling using weighted- averaged residual equations

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Preprint submitted to Environmental Modelling & Software

1       1    **Abstract**

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3       2    The flow over uneven topography is a problem of interest in environmental fluid flow  
4  
5       3    modeling, including flows over river bedforms, exchange flows over oceanic sills or  
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7       4    the airflow over mountains. The common experimental procedure to investigate these  
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9       5    flows, moving a small obstacle in a laboratory flume, yields experimental difficulties,  
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11      6    whereas modeling using non-linear shallow flow equations does not explain all the  
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13      7    flow phenomena. Novel alternative procedures are presented for the experimentation  
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15      8    and shallow water representation of flow interaction with obstacles. A large-scale  
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17      9    obstacle model is constructed in a dam-break set-up, and used to generate flow  
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19     10    phenomena over topography, including dispersive and broken surges, wave reflection,  
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21     11    hydraulic jumps and non-hydrostatic sill overflows. Simulations are conducted with a  
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23     12    shallow-water weighted-averaged residual flow software for turbulent flows. The  
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25     13    proposed software reproduces the experiments satisfactorily, supporting its use in  
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27     14    modeling, whereas the new experimental database can be used by modelers to test their  
28  
29     15    software.

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31  
32     17    *Keywords:* non-hydrostatic flows; laboratory experiments; weighted-averaged  
33  
34     18    residual equations; lee-side waves; obstacles

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## 19 **Highlights**

- 20 • A novel experimental procedure to study wave interaction with obstacles is  
21 presented
- 22 • A new experimental database of utility for software validation by  
23 environmental fluid flow modelers is generated
- 24 • A new weighted-averaged residual flow software for turbulent flow over  
25 obstacles is presented

## 26 **Software availability**

- 27 - Name of software: Waves Transformation Model Software
- 28 - Contact emails: [ag2gaojp@uco.es](mailto:ag2gaojp@uco.es), [z12cachf@uco.es](mailto:z12cachf@uco.es), [ag2caoro@uco.es](mailto:ag2caoro@uco.es)
- 29 - Requirements: MATLAB
- 30 - Availability: The software platform is freely available at  
31 [https://github.com/Frnch/Waves\\_Transformation\\_Model\\_Software](https://github.com/Frnch/Waves_Transformation_Model_Software)

## 32 **1. Introduction**

33 The study of two-dimensional shallow-water flows over obstacles is relevant in  
34 several branches of environmental mechanics, as in the river flow over bedforms, like  
35 dunes and antidunes, in oceanographic exchange flows over a seamount, or in the  
36 mesoscale atmospheric flow past a steep mountain (Nadiga et al., 1996; Zhu and  
37 Lawrence, 1998). A common mathematical procedure to study these flows consists in  
38 set instantaneously an obstacle into an initially steady and uniform stream, and then  
39 study the time-dependent flow adjustment that takes place and the ensuing asymptotic  
40 steady-state (Long, 1954; 1970; Pratt, 1984; Nadiga et al., 1996), typically using either  
41 the shallow-water dispersionless Saint-Venant equations (Houghton and Kasahara  
42 1968, Pratt 1984; Cea et al., 2011) or the dispersive Serre–Green–Naghdi (SGN)  
43 equations (Nadiga et al. 1996). The experimental procedure to generate these flows  
44 consists in rapidly accelerate up to a target constant velocity an obstacle initially at rest  
45 in a flume filled with water (Long, 1954; 1970). In Long’s (1970) experiments, the

1 46 obstacle was moved in the flume by a thin line wrapped around a cylindrical winder  
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3 47 driven by a motor. The velocity of obstacle displacement was experimentally  
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5 48 determined by counting the revolutions, and then this velocity was used to deduce the  
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7 49 flow patterns observed moving with the obstacle, e.g., the equivalent flow with a static  
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9 50 obstacle (Long, 1970). This experimental procedure is however not simple, and is  
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11 51 prone to large number of problems. First, the experimental design is only economical  
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13 52 thus feasible using small obstacles, e.g., in Long's (1970) experiments rather small  
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15 53 ones, only of 9.1 and 2 cm high, were installed. Second, the wave-generation around  
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17 54 the obstacle is induced by moving it, thereby producing leakage at the joins with the  
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19 55 flume sidewalls and at the obstacle bottom. These problems are hardly solvable,  
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21 56 especially for high Froude numbers involving supercritical flows (Long, 1970). In  
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23 57 addition to the experimental configuration problems, the quality and type of  
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25 58 experimental data that can be extracted from this set-up are limited. For example,  
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27 59 instantaneous free surface profile measurements of these flows are not available in the  
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29 60 literature, which would be especially desirable to compare with the theoretical  
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31 61 predictions of shallow-water models. The typical data extracted from Long's  
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33 62 experiments consists only in the upstream and crest flow depths, as well as the Froude  
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35 63 number (Long 1970). Further, the flow over a curved obstacle involves vertical  
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37 64 accelerations and thus non-hydrostatic bed pressures (Nadiga et al., 1996; Zhu and  
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39 65 Lawrence, 1998; Gamero et al., 2020), which should be experimentally determined.  
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41 66 However, in Long's (1970) set-up, the obstacle was moving, thus it would have been  
42  
43 67 a very challenging task to take bed-pressure measurements for the asymptotic steady  
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45 68 flow over the obstacle by installing pressure taps at the obstacle surface. Note that the  
46  
47 69 instantaneous appearance of an obstacle in a stream is not a realistic mechanism of  
48  
49 70 forcing in geophysical fluid flows, despite its wide use by environmental flow  
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51 71 modelers (Pratt, 1984). Thus, an alternative method to study wave interaction with  
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53 72 obstacles in environmental flows was considered, as commented below.

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73 During the time-dependent flow adjustment towards a steady flow over an obstacle,  
74 shocks are formed moving upstream and downstream of it (Long, 1954; 1970;

1 75 Houghton and Kasahara, 1968; Pratt, 1984; Nadiga et al., 1996), with a transition from  
2 76 subcritical to supercritical flow conditions occurring in the vicinity of the obstacle crest  
3 77 (Naghdi and Vongsarnpigoon, 1986; Zhu and Lawrence, 1998). At the lee-side of the  
4 78 obstacle the flow typically changes from supercritical to subcritical flow conditions  
5 79 through a moving shock displacing away from the obstacle. The shocks at the lee-side  
6 80 of the obstacle may be either undular or broken, depending on the Froude number.  
7 81 These instantaneous waves are hardly characterized experimentally in the literature,  
8 82 and simulations using SGN models (Nadiga et al., 1996) showed inability of this  
9 83 shallow-water representation to mimic the turbulent flow processes occurring, given  
10 84 that wave breaking (Bayon et al., 2016; Gualtieri and Chanson, 2021) is not modeled.  
11 85 If the Froude number at the lee-side face of the obstacle is high, turbulent breaking  
12 86 occurs, and the undular waves predicted by SGN models become unrealistic (Nadiga  
13 87 et al., 1996; Castro-Orgaz and Chanson, 2017). Turbulent breaking at the lee-side of  
14 88 obstacles is important, as for example in a oceanographic flow of salt water moving  
15 89 over a sill in a fresh water environment (Farmer and Denton, 1985; Denton, 1987). In  
16 90 these flows, turbulent breaking of the lee-side waves induces mixing between layers  
17 91 and, thus, provides nutrients and dissolved oxygen for the deep water; it further affects  
18 92 the dispersion of any pollutant. Therefore, a shallow-water turbulent flow model with  
19 93 ability to simulate both undular and broken waves is needed to simulate flow over  
20 94 obstacles. The main aim of this research is to characterize the complex features of  
21 95 open-channel flows downstream of a bottom hump, which cannot be simulated by the  
22 96 standard shallow water equations.

23 97 Given that the lee-side obstacle waves are neither characterized with detailed  
24 98 experiments nor simulated with shallow-water turbulent non-hydrostatic models, the  
25 99 experimental and numerical modeling of these flows are the two major objectives of  
26 100 this work.

27 101

1 102 As described in section (ii) of this work, lee-side obstacle waves are generated  
2  
3 103 experimentally in a large-scale obstacle model with a new and entirely different  
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5 104 procedure from that of Long's: A dam-break-like set-up was installed in a long  
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7 105 experimental flume, consisting in a gate installed downstream of the obstacle and  
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9 106 equipped with an instantaneous opening mechanism, thereby permitting to generate  
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11 107 lee-side waves once the gate was opened instantaneously. The type of waves generated  
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13 108 at the lee face of the obstacle depends on the upstream water depth at the gate, which  
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15 109 was an experimental parameter to generate the flows. This new experimental  
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17 110 procedure is different from that of Long, but it permits to generate waves at the lee-  
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19 111 side of obstacles and study their interaction with it, which was the major objective in  
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21 112 Long's set-up. This new experimental set-up has the major advantage of permitting to  
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23 113 execute accurate experiments at a large-scale obstacle model and collecting high-  
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25 114 quality data, including the instantaneous free surface profiles and the steady bed-  
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27 115 pressures over the obstacle. Note that the instantaneous appearance of an obstacle in a  
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29 116 stream as in Long's approach is not a realistic flow; however, this experimental  
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31 117 procedure, in addition to serve to study the flow adjustment over an obstacle in  
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33 118 response to the generated waves, conceptually represents an abrupt drop in the water  
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35 119 levels around the obstacle in response to a downstream forcing. The value of the new  
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37 120 experimental dataset generated in this work relies on two major aspects: (1) In Long's  
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39 121 (1970) experiments, the free surface profiles are presented only qualitatively using  
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41 122 photos, but measurements are not available, given the complications to take such  
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43 123 readings with a moving obstacle; and (2) there is no previous work in the literature to  
44  
45 124 the authors' knowledge, where waves over a curved obstacle are generated with a dam-  
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47 125 break like setting.

48 126 Shallow-water modeling is considered in section (iii); A shallow-water non-  
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50 127 hydrostatic flow model was constructed from the Reynolds-Averaged Navier-Stokes  
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52 128 equations by using a weighted-averaged residual method of Galerkin type. The model  
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54 129 accounts for a non-hydrostatic fluid pressure, which has the ability to model undular  
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56 130 waves. Further, the turbulent velocity profile and turbulent stresses are modeled in the  
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1 131 weighted-averaged equations, permitting to simulate broken waves during the flow  
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3 132 adjustment over the obstacle. A high-resolution finite volume-finite difference  
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5 133 numerical solver is developed and extensively verified in section (iv) using both  
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7 134 benchmark numerical tests and the new experimental measurements conducted in this  
8  
9 135 work. Conclusions are presented in section (v). The model software is freely available  
10  
11 136 on GitHub ([https://github.com/Frnch/Waves\\_Transformation\\_Model\\_Software](https://github.com/Frnch/Waves_Transformation_Model_Software)),  
12  
13 137 whereas the experimental database is provided as supplementary material in the file  
14  
15 138 “Experiments\_EMS2022.xls”.

## 17 139 **2. Experimental characterization**

### 20 140 *2.1 Experimental flume and equips*

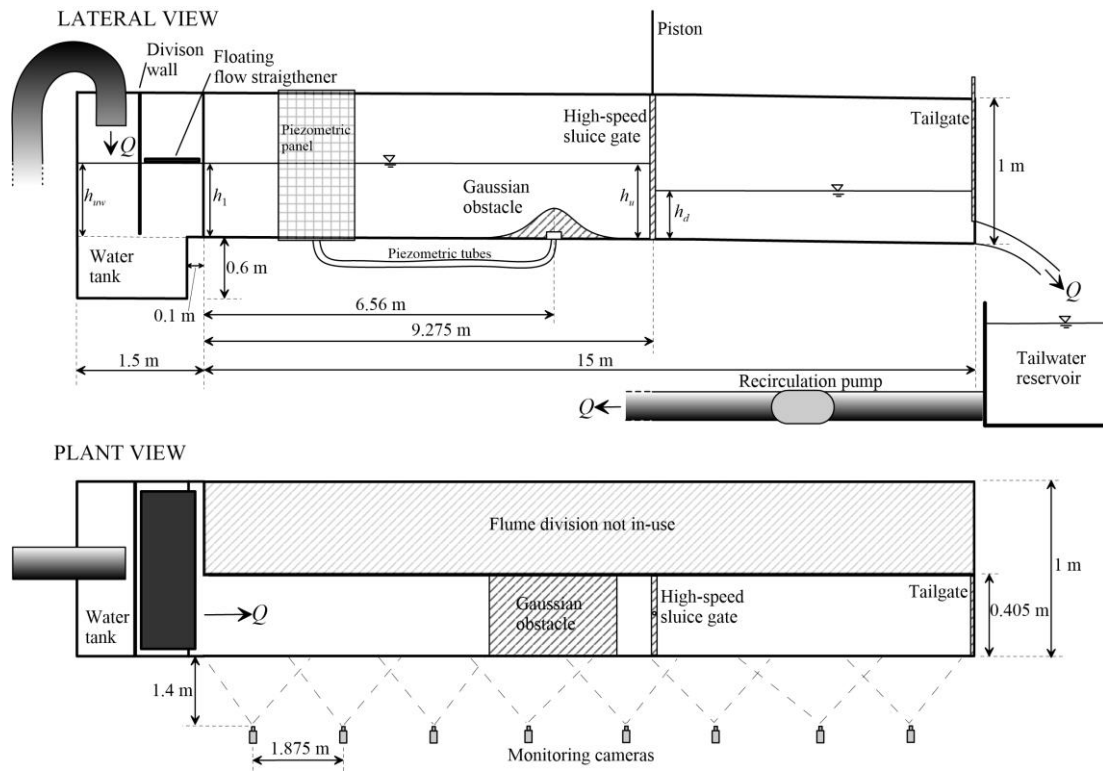
21  
22 141 The experiments were conducted in a 15-m-long, 1-m-high, 1-m-width tilting  
23  
24 142 experimental flume in the Hydraulics laboratory at the University of Córdoba, Spain.  
25  
26 143 A reduction of the flume width to 0.405 m was accomplished by a moving division  
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28 144 wall (Fig. 1), and the flume slope for the experimental series conducted was 0.0015  
29  
30 145 m/m. The tailwater portion of the flume from 9.634 m to 15 m downstream the inlet  
31  
32 146 section was structurally a cantilever, and the beam deformation, though small, was  
33  
34 147 considered in the simulations to accurately define the actual bed profile of the flume.  
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37 148 The flume was equipped with a recirculation pump of 0.078 m<sup>3</sup>/s maximum  
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39 149 discharge connected to a downstream water tank, allowing to work in closed-circuit.  
40  
41 150 A water tank with flow straightener was located at the flume inlet to reduce flow  
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43 151 disturbances. The tailgate of the flume was fully open or closed, depending on the type  
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45 152 of experiment, e.g., with or without reflection at the flume end. A large-scale obstacle  
46  
47 153 of Gaussian profile  $z_{bG} = 0.209 \cdot \exp[-1/2 \cdot ((x - x_{crest})/0.254)^2]$ , where  $z_{bG}$  is the local  
48  
49 154 obstacle height above the flume bed and  $x_{crest}$  the longitudinal location of the crest, was  
50  
51 155 installed at  $x_{crest} = 6.565$  m. A Gaussian obstacle shape was selected because it permits  
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53 156 to mathematically adjust their parameters in the design phase prior to construction. In  
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157 this way, it was possible to adjust the crest curvature until producing the desired degree  
 158 of non-hydrostaticity in the flow over the obstacle. Further, this shape is easy to  
 159 construct in metal by a manufacturer. Along the longitudinal symmetry axis of the  
 160 obstacle 17 piezometric tapings were installed to take bottom pressure head reading in  
 161 a piezometric panel.

162 The flume was equipped with a dam-break like set-up consisting in a sluice gate of  
 163 high-speed release induced by a pneumatic drive system. The gate opening time was  
 164 less than 0.15 s in all the experiments conducted in this work, thus the opening  
 165 operation can be considered instantaneous. A high-speed camera Fastec Ts5 with 50  
 166 mm focal length lens to avoid image distortion capturing at up to 253 fps at maximum  
 167 resolution was used to characterize the movement of the gate and ensure that the  
 168 operation can be considered fast enough to reproduce dam-break like waves.



169

170 **Fig. 1.** Sketch of experimental set-up, showing static water levels at initiation of  
 171 unsteady flow experiments

1 172 Flow visualization during the experiments was accomplished through the eight  
2  
3 173 lateral crystal windows, 1.875-m-wide by 0.975-m-high, of the flume. Each window  
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5 174 was monitored by a camera perpendicularly installed in front of the flume (Fig. 1). The  
6  
7 175 monitoring video system comprises eight Basler Ace acA1920-40uc cameras, with 6  
8  
9 176 mm focal length lens to allow capturing the whole width of each lateral crystal  
10  
11 177 window, recording 40 frames-per-second (fps) maximum at full resolution, and a  
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13 178 laptop Intel® Core™ i7-9750H with software for image capture, synchronization,  
14  
15 179 assembling and processing. The system automatically assembles the images collected  
16  
17 180 by the 8 cameras in a synchronized way, correcting distortion errors and thereby  
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19 181 providing instantaneous experimental images of the 15 meters of flume.

20 182 The novelty of this experimental research is in the experimental procedure and set-  
21  
22 183 up used to generate waves evolving over obstacles. In Long's (1970) classical  
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24 184 experiments, an obstacle is moved in a flume filled with initially still water (Pratt and  
25  
26 185 Whitehead 2007), whereas in this work we have generated the unsteady waves over a  
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28 186 fixed obstacle using a dam-break like set-up.

## 31 187 *2.2 Experimental series*

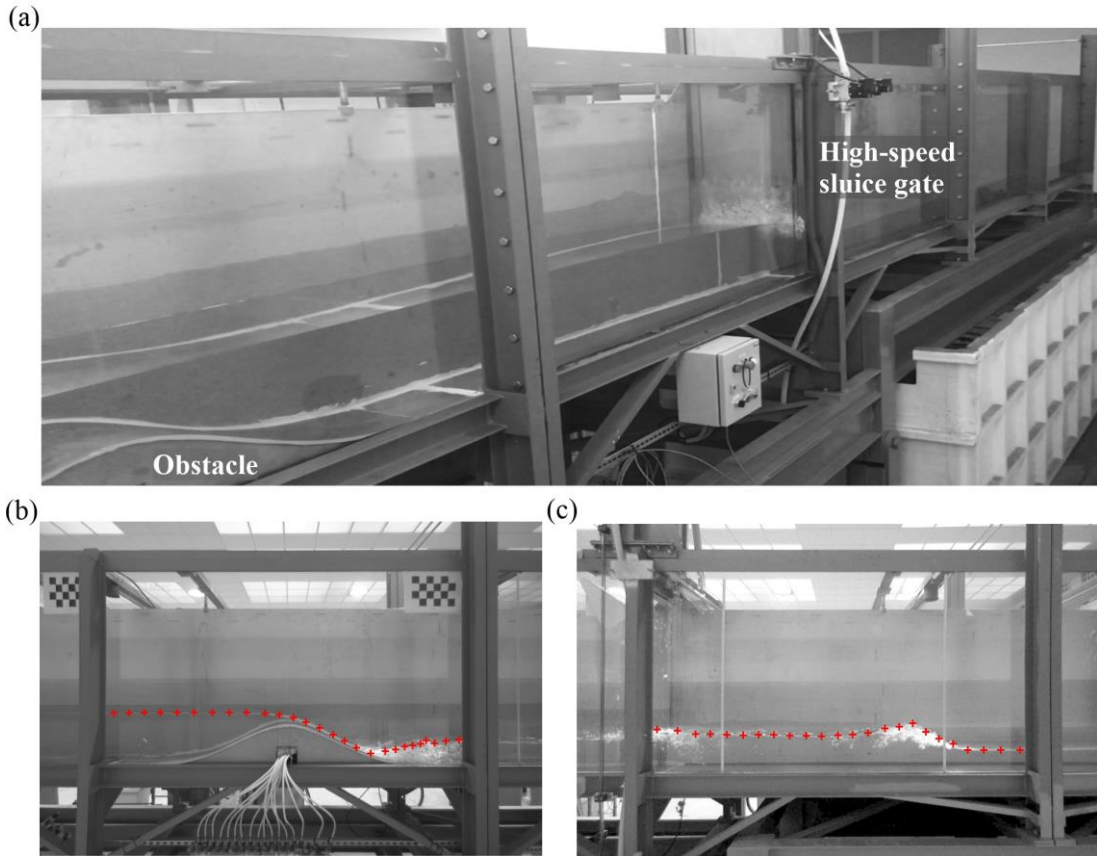
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34 188 Two kinds of experimental series were produced. The first series consisted of dam-  
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36 189 break wave experiments generated by using the high-speed sluice gate with no inlet  
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38 190 flow ( $Q = 0$ ), and the tailgate fully closed to allow wave reflection. The second series  
39  
40 191 consisted of steady flow experiments with various  $Q$  up to the maximum discharge,  
41  
42 192 with the high-speed gate deactivated (no operation, positioned above the flume) and  
43  
44 193 the tailgate fully opened.

198 **Table 1.** Test series characteristics for dam-break experiments

Test	$h_u$ (m)	$h_d$ (m)	$r$ (-)
1	0.302	0.12	0.397
2	0.3	0.18	0.6
3	0.3	0.24	0.8
4	0.3	0	0

199

200 The dam break experiments were designed using different downstream to upstream  
 201 water depth ratios,  $r = h_d/h_u$ , with zero inlet discharge and closed tailgate, which were  
 202 organized in tests series comprising  $r = 0, 0.397, 0.6,$  and  $0.8$  (Table 1). First, the flume  
 203 was filled with water up to the level  $h_d$  considered, and then the high-speed sluice gate  
 204 closed. Thereafter, the upstream side of the gate further filled up to the desired  $r$  was  
 205 fixed. Afterwards, the high-speed sluice gate was released (Fig. 2a).



**Fig. 2.** Experimental tests: (a) photograph of the test 1 right after the sluice gate release, (b) corrected image of camera 4 at  $t = 6$  s with unsteady hydraulic jump at the toe of the obstacle, and (c) corrected image of camera 6 at  $t = 1$  s in test 1 with the advancing dam break bore. Digitized data points marked by red crosses.

The monitoring system was set to record images at 25 fps, which was enough to take a detailed experimental characterization of the unsteady water waves. Each test was recorded during 12 seconds, enabling to capture all the relevant hydraulic processes, namely the positive and negative dam-break wave generation, wave reflection at the closed tailwater gate, formation of a hydraulic jump at the lee-side of the obstacle, and interaction of the reflected wave with the flow developing over the obstacle.

218 Once the images collected by the system of cameras were assembled and distortion  
219 errors were corrected for selected instants of time, they were used to extract the  
220 instantaneous flow profiles (Fig. 2b, c). The free surface was digitalized from the  
221 images over the free surface curve in the crystal wall using Golden Software Grapher®  
222 13.3.754. By trial-and-error calibration tests, it was found that the procedure permits  
223 to measure the experimental flow profile with  $\pm 0.1$  cm accuracy. For illustration  
224 purposes, panoramic instants at  $t = 1, 6$  and  $10$  s for test 1 are shown in Fig. 3 using  
225 the corrected images of the cameras 4, 5 and 6 (from left to right in each subfigure).



226  
227 **Fig. 3.** Panoramic views of cameras 4, 5 and 6 (from left to right) for test 1 at: (a)  $t =$   
228  $1$  s, (b)  $t = 6$  s, and (c)  $t = 10$  s. The piezometric tubes are shown in the left images.

229 An accurate modeling of the upstream boundary condition in the numerical solver  
 230 requires use as input the time-variation of the static upstream water level at the inlet  
 231 tank solid wall  $h_{uw}$ . This time-dependent variable was measured during the  
 232 experiments with the high-speed camera Fastec Ts5, and from the ensuing  
 233 measurements, the following 4<sup>th</sup>-order polynomial was found to describe inlet flow  
 234 conditions:  $h_{uw}$  (m) =  $(0.0015 \cdot t^4 - 0.032 \cdot t^3 + 0.1656 \cdot t^2 - 0.2503 \cdot t + 0.0393 +$   
 235  $100 \cdot h_1)/100$  ( $R^2 = 0.98$ ), where  $h_{uw}$  is the flow depth at the upstream wall of the water  
 236 tank,  $t$  is the time and  $h_1$  is the flow depth level at the upstream section of the flume  
 237 (see Fig. 1).

238 Steady flow experiments were conducted with fully open tail- and high-speed sluice  
 239 gates with various discharges up to the maximum of the system,  $0.1826 \text{ m}^2/\text{s}$ . The bed  
 240 pressure head was measured by visual observation of a piezometric panel allowing  
 241 readings of accuracy  $\pm 0.1$  cm.

### 242 3. Shallow-water turbulent flow modeling

#### 243 3.1 Weighted-averaged residual equations

244 Consider steady two-dimensional flow in a vertical plane (Fig. 4), as in the  
 245 previously described experiments (Fig. 3). The modeling approximation pursued here  
 246 entails the development of weighted-averaged residual equations from the RANS  
 247 equations of turbulent free surface flow following Steffler and Jin (1993). In a first  
 248 step, a sequence of Vertically Averaged and Moment (VAM) equations is produced  
 249 by using the first shifted Legendre polynomial as test function, resulting (Steffler and  
 250 Jin, 1993; Khan and Steffer, 1996a; b):

$$251 \quad \frac{\partial h}{\partial t} + \frac{\partial h \bar{u}}{\partial x} = 0, \quad (1)$$

$$252 \quad \frac{\partial h \bar{u}}{\partial t} + \frac{\partial h \bar{u}^2}{\partial x} = -\frac{1}{\rho} \left[ \frac{\partial h (\bar{p} - \bar{\sigma}_x)}{\partial x} + (p_b - \sigma_{xb}) \frac{\partial z_b}{\partial x} + \tau_{xzb} \right], \quad (2)$$

$$\frac{\partial h\bar{w}}{\partial t} + \frac{\partial h\bar{u}w}{\partial x} = \frac{1}{\rho} \left( p_b + \frac{\partial h\bar{\tau}_{xz}}{\partial x} + \tau_{xzb} \frac{\partial z_b}{\partial x} - \sigma_{zb} \right) - gh, \quad (3)$$

$$\frac{h}{2} \frac{\partial h}{\partial t} + \frac{\partial hz\bar{u}}{\partial x} - \bar{z} \frac{\partial h\bar{u}}{\partial x} - h\bar{w} = 0, \quad (4)$$

$$\begin{aligned} \frac{\partial hz\bar{u}}{\partial t} + \frac{\partial hz\bar{u}^2}{\partial x} - \bar{z} \frac{\partial h\bar{u}}{\partial t} - \bar{z} \frac{\partial h\bar{u}^2}{\partial x} - h\bar{u}w = & -\frac{1}{\rho} \left( \frac{\partial hz\bar{p}}{\partial x} - \bar{z} \frac{\partial h\bar{p}}{\partial x} - \frac{hp_b}{2} \frac{\partial z_b}{\partial x} \right) \\ & + \frac{1}{\rho} \left( \frac{\partial hz\bar{\sigma}_x}{\partial x} - \bar{z} \frac{\partial h\bar{\sigma}_x}{\partial x} - \frac{h\sigma_{xb}}{2} \frac{\partial z_b}{\partial x} + \frac{h\tau_{xzb}}{2} - h\bar{\tau}_{xz} \right), \end{aligned} \quad (5)$$

$$\begin{aligned} \frac{\partial hz\bar{w}}{\partial t} + \frac{\partial hz\bar{u}w}{\partial x} - \bar{z} \frac{\partial h\bar{w}}{\partial t} - \bar{z} \frac{\partial h\bar{u}w}{\partial x} - h\bar{w}^2 = & -\frac{h}{\rho} \left( \frac{p_b}{2} - \bar{p} \right) \\ & + \frac{1}{\rho} \left( \frac{\partial hz\bar{\tau}_{xz}}{\partial x} - \bar{z} \frac{\partial h\bar{\tau}_{xz}}{\partial x} - \frac{h\tau_{xzb}}{2} \frac{\partial z_b}{\partial x} + \frac{h\sigma_{zb}}{2} - h\bar{\sigma}_z \right), \end{aligned} \quad (6)$$

where  $x$  and  $z$  are the horizontal and vertical Cartesian coordinates, respectively;  $u(x, z, t)$  and  $w(x, z, t)$  are the horizontal and vertical velocity components;  $z_b(x)$  is the bed profile;  $h(x, t)$  is the flow depth;  $p(x, z, t)$  is the fluid pressure;  $p_b(x, t)$  is bottom pressure;  $\tau(x, z, t)$  and  $\sigma(x, z, t)$  are the Reynolds tangential and normal stresses, respectively;  $\rho$  is the fluid density;  $g$  is the gravitational acceleration; and  $t$  is the time. The overbar operator denotes vertically-averaged quantities. Equations (1)–(3) are the continuity,  $x$ -, and  $z$ -momentum equations, respectively, while Eqs. (4)–(6) are the moment of continuity,  $x$ -, and  $z$ -momentum equations, respectively. Note that  $\bar{z}$  is the elevation of the centroid of a section ( $= z_b + h/2$ ). Equations (1)–(6) are weighted-averaged open-channel flow equations, yet not residual, given that  $(u, w, p)$  are still general.

In the VAM model [Eqs. (1)–(6)], predictors for the velocity components  $(u, w)$  and the fluid pressure  $p$  are required. Steffler and Jin (1993) used finite-element type

273 expansions consisting of a base of functions with a series of coefficients independent  
 274 of the vertical coordinate. In particular, they expanded  $u$  using the first shifted  
 275 Legendre polynomial, and  $w$  and  $p$  using the first and second shifted Legendre  
 276 polynomials, resulting (Steffler and Jin, 1993; Khan and Steffer, 1996a; b):

$$277 \quad u(x, z, t) = u_0(x, t) + u_1(x, t)[2\eta(x, z, t) - 1], \quad (7)$$

$$278 \quad w(x, z, t) = [w_b(x, t) + w_2(x, t)4\eta(x, z, t)][1 - \eta(x, z, t)] + w_s(x, t)\eta(x, z, t), \quad (8)$$

$$279 \quad p(x, z, t) = [\rho gh(x, t) + p_1(x, t) + p_2(x, t)4\eta(x, z, t)][1 - \eta(x, z, t)]. \quad (9)$$

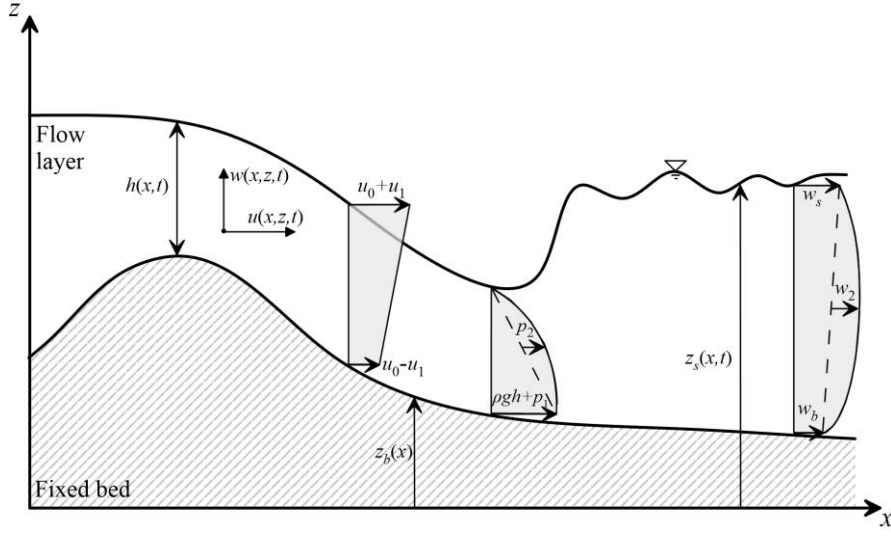
280 Here  $u_0$  is the depth-averaged horizontal velocity;  $u_1$  is the  $x$ -velocity at the free  
 281 surface in excess of  $u_0$ ;  $w_2$  is the mid-depth  $z$ -velocity in excess of the average of the  
 282 vertical velocities at the bed and free surface levels;  $w_b$  and  $w_s$  are the vertical velocity  
 283 at the bed and free surface levels, respectively;  $p_1$  is the bed pressure in excess of  
 284 hydrostatic;  $p_2$  is the mid-depth deviation from the linear non-hydrostatic law; and  $\eta$  is  
 285 the dimensionless vertical coordinate  $[= (z - z_b)/h]$ . The kinematic boundary conditions  
 286 are expressed as follows (Cantero-Chinchilla et al., 2018; 2020):

$$287 \quad w_b = (u_0 - u_1) \frac{\partial z_b}{\partial x}, \quad (10)$$

$$288 \quad w_s = \frac{\partial h}{\partial t} + (u_0 + u_1) \frac{\partial z_s}{\partial x}, \quad (11)$$

289 where  $z_s$  is the free surface elevation ( $= h + z_b$ ).





291 **Fig. 4.** Definition sketch of two-dimensional unsteady turbulent flow.

292 Inserting Eqs. (7)–(8), which are the trial functions approximating  $(u, w, p)$ , into Eqs.  
 293 (1)–(6), the following system of approximate (residual) partial differential equations  
 294 (PDEs) results:

$$295 \quad \frac{\partial h}{\partial t} + \frac{\partial q}{\partial x} = 0, \quad (12)$$

$$296 \quad \frac{\partial q}{\partial t} + \frac{\partial}{\partial x} \left( g \frac{h^2}{2} + \frac{q^2}{h} \right) = - \frac{\partial}{\partial x} \left( \frac{u_1^2 h}{3} + \frac{h p_1}{2\rho} + \frac{2h p_2}{3\rho} - \frac{h \overline{\sigma_x}}{\rho} \right) - gh \frac{\partial z_b}{\partial x} - \frac{p_1}{\rho} \frac{\partial z_b}{\partial x} - \frac{\tau_b}{\rho}, \quad (13)$$

$$297 \quad \frac{\partial h \overline{w}}{\partial t} + \frac{\partial q \overline{w}}{\partial x} = \frac{1}{6} \frac{\partial (h u_1 w_*)}{\partial x} + \frac{p_1}{\rho} - \frac{\tau_b}{\rho} \frac{\partial z_b}{\partial x} + \frac{1}{\rho} \frac{\partial h \overline{\tau_{xz}}}{\partial x}, \quad (14)$$

$$298 \quad \frac{h}{2} \frac{\partial h}{\partial t} + q \frac{\partial \overline{z}}{\partial x} + \frac{1}{6} \frac{\partial (h^2 u_1)}{\partial x} = h \overline{w}, \quad (15)$$

$$299 \quad \frac{\partial h u_1}{\partial t} + \frac{\partial q u_1}{\partial x} = \frac{h}{2\rho} \frac{\partial p_1}{\partial x} + \left( \frac{q u_1}{h} - \frac{p_1}{2\rho} \right) \frac{\partial h}{\partial x} - u_1 \frac{\partial q}{\partial x} - \frac{4 p_2}{\rho} \frac{\partial \overline{z}}{\partial x} \\ + \frac{6}{\rho} \left( \frac{\tau_b}{2} - \overline{\tau_{xz}} \right) + \frac{6}{\rho h} \left( \frac{\partial h \overline{z \sigma_x}}{\partial x} - \overline{z} \frac{\partial h \overline{\sigma_x}}{\partial x} \right), \quad (16)$$

$$\begin{aligned} \frac{\partial}{\partial t} \left( \frac{h^2 w_*}{12} \right) + \frac{\partial}{\partial x} \left( \frac{h q w_*}{12} \right) = & \frac{h \bar{w}}{2} \frac{\partial h}{\partial t} + \left( q \bar{w} - \frac{h u_1 w_*}{6} \right) \frac{\partial \bar{z}}{\partial x} + \frac{\partial}{\partial x} \left[ \frac{h^2 u_1}{10} \left( \bar{w} + \frac{w_b}{3} + \frac{w_s}{3} \right) \right] \\ & - h \bar{w}^2 - \frac{2 h p_2}{3 \rho} - \frac{h \tau_b}{2 \rho} \frac{\partial z_b}{\partial x} + \frac{h \bar{\sigma}_z}{\rho} - \frac{1}{\rho} \left( \frac{\partial h z \bar{\tau}_{xz}}{\partial x} - \bar{z} \frac{\partial h \bar{\tau}_{xz}}{\partial x} \right), \end{aligned}$$

$$(17)$$

where  $q$  is the discharge per unit width ( $= hu_0$ );  $w_*$  is the vertical velocity difference between the bed and free surface levels ( $= w_b - w_s$ );  $\tau_b$  is the bed shear stress;  $\bar{\sigma}_x$  and  $\bar{\tau}_{xz}$  are the depth-averaged normal and shear stresses, respectively; and  $\bar{w}^2 = \bar{w}^2 + \frac{1}{12} w_b^2 + \frac{1}{12} w_s^2 - \frac{1}{6} w_b w_s - \frac{1}{20} (2\bar{w} - w_b - w_s)^2$ . The mean vertical velocity is  $\bar{w} = w_b/2 + 2w_s/3 + w_s/2$ . Given that the base functions used in the trial solution are used as test functions, the VAM model is a Galerkin-type system of weighted-averaged residual equations (Finlayson and Scriven, 1966). The moment of  $x$ -momentum Eq. (16) differs from that previously used (Cantero-Chinchilla et al., 2018; Gamero et al., 2020) in that the conservative variable is  $hu_1$  instead of  $u_1$ . This modification was found to increase the numerical robustness of the VAM model when handling dry-wet fronts.

Further, Eqs. (16) and (17) include all turbulent stress terms originating from the weighted-averaging process. Former models in Cantero-Chinchilla et al. (2018) and Gamero et al. (2020) only considered bed-shear effects, thus here, the complete turbulent modeling terms were accounted for in the model equations. The following reactive equation is written based on the kinematic boundary conditions:

$$w_* = \left( \frac{q}{h} - u_1 \right) \frac{\partial z_b}{\partial x} + \frac{\partial q}{\partial x} - \left( \frac{q}{h} + u_1 \right) \frac{\partial (z_b + h)}{\partial x}. \quad (18)$$

It is a mathematical statement to be verified by the solution at any instant of time. Turbulence closure is required to estimate  $\tau_b$ ,  $\bar{\sigma}_x$ ,  $\bar{\sigma}_z$ ,  $\bar{\tau}_{xz}$ ,  $z\bar{\sigma}_x$ , and  $z\bar{\tau}_{xz}$  in Eqs. (13),

321 (14), (16), and (17). Using an eddy-viscosity approach, these terms read after  
 322 averaging:

$$\overline{\sigma_x} = 2\rho \overline{v_x} \left( \frac{\partial q}{\partial x} - u_s \frac{\partial z_s}{\partial x} + u_b \frac{\partial z_b}{\partial x} \right), \quad (19)$$

$$\overline{\sigma_z} = 2\rho \overline{v_z} (w_s - w_b), \quad (20)$$

$$\overline{\tau_{xz}} = \frac{\rho \overline{v_z}}{h} \left( 2u_1 + \frac{\partial h \bar{w}}{\partial x} - w_s \frac{\partial z_s}{\partial x} + w_b \frac{\partial z_b}{\partial x} \right), \quad (21)$$

$$\overline{z\sigma_x} = \frac{\rho \overline{v_x} h}{3} \left[ 3 \frac{\partial}{\partial x} \left( \frac{q}{h} \right) + \frac{\partial u_1}{\partial x} \right] - \frac{\rho \overline{v_x} u_1 z_b}{h} \left( 2 \frac{\partial h}{\partial x} + 4 \frac{\partial z_b}{\partial x} \right) - \frac{\rho \overline{v_x}}{3} \left[ 4u_1 \frac{\partial h}{\partial x} - 6z_b \frac{\partial}{\partial x} \left( \frac{q}{h} \right) + 6u_1 \frac{\partial z_b}{\partial x} \right], \quad (22)$$

$$\overline{z\tau_{xz}} = \rho \overline{v_z} u_1 + \frac{2\rho \overline{v_z} u_1 z_b}{h} + \frac{\rho \overline{v_z} h}{2} \frac{\partial}{\partial x} \left( \bar{w} - \frac{w_b}{6} + \frac{w_s}{6} \right) + \rho \overline{v_z} \left( \bar{w} - \frac{w_b}{6} - \frac{5w_s}{6} \right) \frac{\partial h}{\partial x} + \rho \overline{v_z} z_b \frac{\partial \bar{w}}{\partial x} + \rho \overline{v_z} (\bar{w} - w_s) \frac{\partial z_b}{\partial x} + \frac{\rho \overline{v_z} z_b}{h} (\bar{w} - w_s) \frac{\partial h}{\partial x} + \frac{\rho \overline{v_z} w_s z_b}{h} \frac{\partial z_b}{\partial x}, \quad (23)$$

328 where  $\overline{v_x}$  and  $\overline{v_z}$  are the vertically-averaged eddy viscosities in the  $x$ - and  $z$ -directions,  
 329 respectively. In the traditional shallow-water quasi-3D approach, e.g., Rodi (1993), the  
 330 pressure is assumed to be vertically hydrostatic, with corrections using turbulent  
 331 stresses based on the depth-averaged horizontal velocity  $u_0$ . In the present formulation,  
 332 pressures are dynamic ones, with perturbation parameters  $p_1$  and  $p_2$ , which are  
 333 corrected in the model equations by the depth-averaged turbulent stresses considering  
 334 the non-uniform variation of the  $u$ -velocity with elevation, given by  $u_s$ ,  $u_b$  and  $u_1$ , as  
 335 well as the vertical velocity  $w$  variation, determined by  $w_s$ ,  $w_b$  and  $w_2$ .

336 Following Ghamry and Steffler (2002a; b), the depth-averaged eddy viscosities are  
 337 estimated as (Fischer et al., 1979):

$$\overline{v_x} = 0.5u_*h, \quad (24)$$

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$$\overline{v_z} = 0.07u_*h, \quad (25)$$

340 where  $u_*$  is the shear velocity [=  $(|\tau_b|/\rho)^{1/2}$ ]. The bed shear stress is modelled using  
341 Manning's formula including vertical velocity effects as follows (Castro-Orgaz and  
342 Hager, 2017; Cantero-Chinchilla et al., 2018; 2020):

$$\tau_b = \rho g \frac{n^2 (u_b^2 + w_b^2)}{h^{1/3}}, \quad (26)$$

344 where  $n$  is the Manning's roughness coefficient.

### 345 3.2 Numerical modeling and Software development

346 The software development entails the solution of the turbulent VAM model [Eqs.  
347 (12)–(17)] through numerical techniques, given that an analytical solution of the  
348 system of PDEs equations is unknown. A semi-implicit finite volume (FV)-finite  
349 difference (FD) scheme is developed based on former works (Cantero-Chinchilla et  
350 al., 2018; 2020; Gamero et al., 2020). Former solvers were prone to numerical  
351 instabilities if shocks or moving hydraulic jumps were progressively developed in the  
352 solution, causing the solution failure in some cases. Therefore, a special feature of the  
353 new solver constructed is its ability for handling the formation of shocks with  
354 robustness, as described below.

355 The VAM model is in matrix form:

$$\frac{\partial \mathbf{U}}{\partial t} + \frac{\partial \mathbf{F}}{\partial x} = \mathbf{S}_o + \mathbf{S}_\tau, \quad (27)$$

357 where vectors  $\mathbf{U}$ ,  $\mathbf{F}$  and  $\mathbf{S}$  enclose, respectively, the flow conservative variables, fluxes  
358 and source terms:

359

$$\mathbf{U} = \begin{bmatrix} h \\ q \\ h\bar{w} \\ hu_1 \\ \frac{h^2 w_*}{12} \end{bmatrix}, \mathbf{F} = \begin{bmatrix} q \\ \frac{q^2}{h} + g \frac{h^2}{2} \\ q\bar{w} \\ qu_1 \\ \frac{hqw_*}{12} \end{bmatrix}, \quad (28)$$

360

$$\mathbf{S}_o = \begin{bmatrix} 0 \\ -\frac{\partial}{\partial x} \left( \frac{u_1^2 h}{3} + \frac{hp_1}{2\rho} + \frac{2hp_2}{3\rho} \right) - gh \frac{\partial z_b}{\partial x} - \frac{p_1}{\rho} \frac{\partial z_b}{\partial x} \\ \frac{1}{6} \frac{\partial (hu_1 w_*)}{\partial x} + \frac{p_1}{\rho} \\ \frac{h}{2\rho} \frac{\partial p_1}{\partial x} + \left( \frac{qu_1}{h} - \frac{p_1}{2\rho} \right) \frac{\partial h}{\partial x} - u_1 \frac{\partial q}{\partial x} - \frac{4p_2}{\rho} \frac{\partial \bar{z}}{\partial x} \\ \left[ -\frac{h\bar{w}}{2} \frac{\partial q}{\partial x} + \left[ q\bar{w} - \frac{hu_1 w_*}{6} \right] \frac{\partial \bar{z}}{\partial x} \right. \\ \left. + \frac{\partial}{\partial x} \left[ \frac{h^2 u_1}{10} \left( \bar{w} + \frac{w_b}{3} + \frac{w_s}{3} \right) \right] - h\bar{w}^2 - \frac{2hp_2}{3\rho} \right] \end{bmatrix},$$

361

$$\mathbf{S}_\tau = \begin{bmatrix} 0 \\ +\frac{1}{\rho} \frac{\partial h\bar{\sigma}_x}{\partial x} - \frac{\tau_b}{\rho} \\ -\frac{\tau_b}{\rho} \frac{\partial z_b}{\partial x} + \frac{1}{\rho} \frac{\partial h\bar{\tau}_{xz}}{\partial x} \\ +\frac{6}{\rho} \left( \frac{\tau_b}{2} - \bar{\tau}_{xz} \right) + \frac{6}{\rho h} \left( \frac{\partial h z \bar{\sigma}_x}{\partial x} - \bar{z} \frac{\partial h\bar{\sigma}_x}{\partial x} \right) \\ -\frac{h\tau_b}{2\rho} \frac{\partial z_b}{\partial x} + \frac{h\bar{\sigma}_z}{\rho} - \frac{1}{\rho} \left( \frac{\partial h z \bar{\tau}_{xz}}{\partial x} - \bar{z} \frac{\partial h\bar{\tau}_{xz}}{\partial x} \right) \end{bmatrix}, \quad (29)$$

362 The subscripts  $o$  and  $\tau$  refer to the inviscid and the turbulent stress source terms,  
 363 respectively. Note that only transport Eqs. (12)–(14), (16), and (17) are contained into  
 364 Eq. (27) which, together with the reactive Eqs. (15) and Eq. (18), conforms the VAM  
 365 model.

1 366 This shallow-water VAM model is more complex than the Saint Venant equations, but  
 2  
 3 367 its features are significantly better. It may be noted that the RANS equations are  
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 5 368 simpler in its formulation. However, the computational cost of a full 3D approach is  
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 7 369 still high at a river scale (Katopodes 2019), involving the determination of the free  
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 9 370 surface boundary using the volume of fluid or level set methods. In contrast, the  
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 11 371 position of the free surface is directly resolved in this depth-averaged formulation.  
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 13 372 Note that the VAM equations look complex given their long source terms [see Eqs.  
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 15 373 (29)], but the architecture of the equations is similar to that of the standard shallow  
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 17 374 water equations [see Eq. (27)]. The solution of this model is vectorized in the present  
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 19 375 code, such that implementation is easy.

20 376 The semi-implicit FV-FD scheme follows a splitting approach in two stages: (i) a  
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 22 377 hyperbolic step and (ii) an elliptical step.

23  
 24 378 Dividing the  $x$ - $t$  plane into quadrilateral finite volume cells of dimensions  $\Delta x \times \Delta t$ ,  
 25  
 26 379 in the hyperbolic step an intermediate solution is obtained from the homogenous part  
 27  
 28 380 of Eq. (27) using a Godunov-type finite-volume scheme (Toro, 2001; 2009):

$$\hat{\mathbf{U}}_i = \mathbf{U}_i^k - \frac{\Delta t}{\Delta x} (\mathbf{F}_{i+1/2} - \mathbf{F}_{i-1/2}). \quad (30)$$

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 37 383 In Eq. (30),  $\mathbf{U}$  and  $\mathbf{F}$  are space- and time-averaged vectors;  $i$  is the cell index;  $k$  is a  
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 39 384 time index;  $\Delta x$  is the  $x$ -dimension of the control volume;  $\Delta t$  is the  $t$ -dimension of the  
 40  
 41 385 control volume. The indices  $i \pm 1/2$  refer to the control volume interfaces between cells  
 42  
 43 386  $i$  and  $i \pm 1$ . The numerical flux  $\mathbf{F}_{i+1/2}$  is determined using the approximate Riemann  
 44  
 45 387 solver HLLC (Toro, 2001; 2009). Note that contact waves are accounted for in the  
 46  
 47 388 solution of Eq. (30), as those described by the conservative variables  $h\bar{w}$ ,  $hu_1$  and  
 48  
 49 389  $\frac{1}{12} h^2 w_*$ . Here, the MUSCL–Hancock scheme (Toro, 2001; 2009), which is the second-  
 50  
 51 390 order accurate in space and time, is applied to reconstruct  $\mathbf{U}$ . This produces more  
 52  
 53 391 efficient and robust computations than former solvers based on 4<sup>th</sup>-order accurate

reconstructions (Cantero-Chinchilla et al., 2018; Gamero et al., 2020). Besides, to avoid unphysical numerical flux during the reconstruction of the flow depth over uneven topography, the weighted surface-depth gradient method (WSDGM) (Aureli et al., 2008) is used, where at the cell interfaces, the water depth is determined as an average of the values obtained reconstructing the free surface and the water depth independently. Dry cells are identified as those where the flow depth  $h$  is below a prescribed tolerance, which is adopted as  $h_{tol} = 10^{-6}$  in this work. At a dry cell all variables are reset to zero.

In the elliptical step, the solution is updated in two stages using finite-difference schemes. The first stage of the elliptical step is called inviscid finite-difference step, which is designed to incorporate the non-hydrostatic pressure effects into the solution using an implicit scheme. Using the backward Euler formula, the system of equations to solve is in compact form:

$$\tilde{\mathbf{U}}_i = \hat{\mathbf{U}}_i + \Delta t \left[ \mathbf{S}_0 \left( \tilde{\mathbf{U}}_i \right) \right]. \quad (31)$$

The implicit system of equations described by Eq. (31) is coupled to the reactive Eqs. (15) and (18) and solved using a Newton–Raphson (NR) method to obtain  $\tilde{\mathbf{U}}$ . The spatial derivatives in Eqs. (15), (18) and (31) are discretized using second-order central finite differences. If shocks are formed in any portion of the computational domain, the gradients of some of the flow variables may reach large values. These may result in numerical instabilities. A new special method for handling shock development is presented in the next section. Here, it is assumed that the solution is smooth throughout the computational domain.

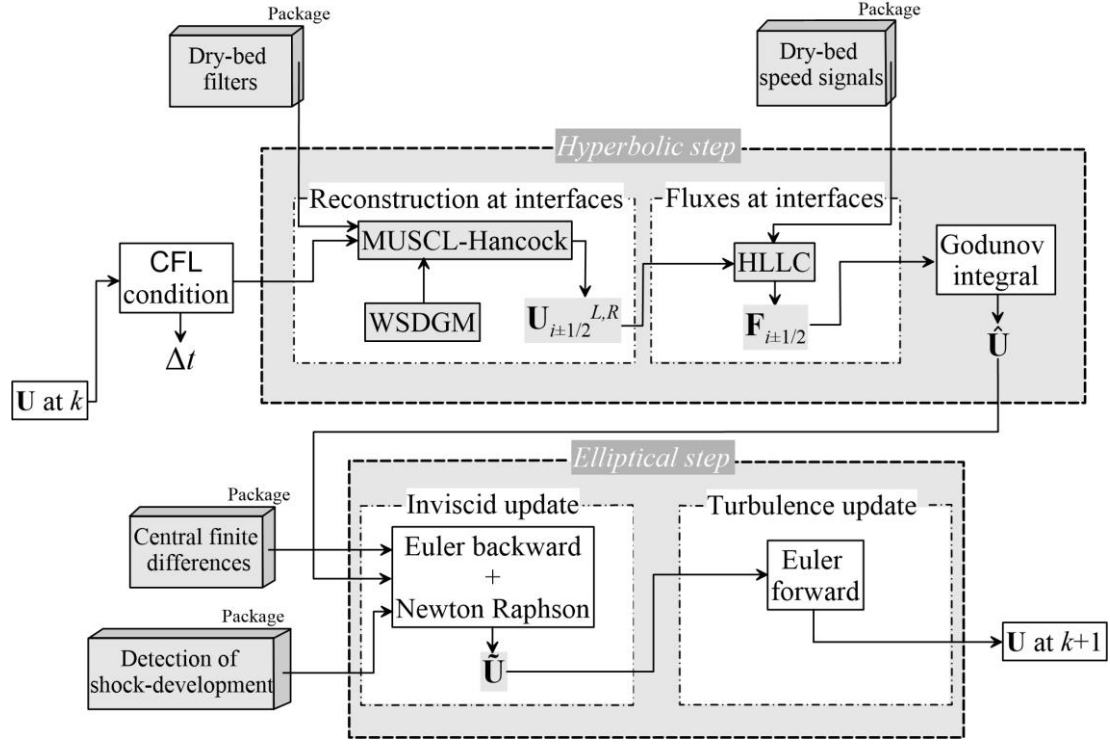
Setting appropriate initial values for the unknown variables ( $q, u_1, p_1, p_2, \bar{w}$  and  $w^*$ ) a vector of residuals  $\mathbf{r} = \tilde{\mathbf{U}}_i^m - \hat{\mathbf{U}}_i - \Delta t \left[ \mathbf{S}_0 \left( \tilde{\mathbf{U}}_i^m \right) \right]$  is defined, where  $m$  is a recursion index. Residuals are reduced by computing an analytical Jacobian  $\mathbf{J}$  for the unknown variables at nodes  $i-1, i$  and  $i+1$ , yielding a  $6N \times 6N$  diagonal matrix formed by 108

418 partial derivatives per cell, where  $N$  is the number of cells. The matrix equation  
 419  $\mathbf{r} = \mathbf{J} \times \mathbf{d}\tilde{\mathbf{U}}_i$  is solved at each iteration of the solution, where  $\mathbf{d}\tilde{\mathbf{U}}_i$  is the vector of  
 420 corrections. Should a dry cell be detected, the corresponding residuals are set to zero  
 421 [all except Eq. (13), which is not updated]. Once  $\mathbf{d}\tilde{\mathbf{U}}_i$  is determined through the  
 422 pertinent matrix inversion, it is employed to update the solution vector, i.e.,  
 423  $\tilde{\mathbf{U}}_i^{m+1} = \tilde{\mathbf{U}}_i^m + \mathbf{d}\tilde{\mathbf{U}}_i$ , where  $\tilde{\mathbf{U}}_i^{m+1}$  is the updated solution. This process is repeated in the  
 424 NR method at every time step until convergence. The convergence criterion suggested  
 425 by Khan and Steffler (1996a) is implemented, stopping the iterations if the mean  
 426 relative error is below a prescribed tolerance, settled as  $10^{-6}$  in this work. To save  
 427 computational cost, at each time step  $\mathbf{J}$  is frozen, i.e., computed at the start of the loop  
 428 (Cantero-Chinchilla et al., 2018; 2020).

429 The second stage of the elliptical step consists in an explicit update of the solution  
 430 vector  $\tilde{\mathbf{U}}$  by incorporating the turbulent source terms,  $\mathbf{S}_\tau$ , resulting using the forward  
 431 Euler formula:

$$432 \quad \mathbf{U}_i^{k+1} = \tilde{\mathbf{U}}_i + \Delta t \left[ \mathbf{S}_\tau \left( \tilde{\mathbf{U}}_i \right) \right]. \quad (32)$$





433

434 **Fig. 5.** Flow chart of the numerical scheme.

435 Figure 5 shows a flow chart of the numerical sequence described above, which is  
 436 followed in every time step. The Courant–Friedrichs–Lewy (CFL) condition is used  
 437 to compute  $\Delta t$  thus ensuring numerical stability of the hybrid FV–FD scheme, i.e.,  $\Delta t$   
 438 =  $(\text{CFL} \cdot \Delta x) / (|u_0 + c|)$  where  $c = (gh)^{1/2}$  is the long wave celerity and  $\text{CFL} \leq 0.5$  by  
 439 numerical experimentation. Although the numerical scheme is stable for  $\text{CFL} \leq 0.5$ ,  
 440 we generally used  $\text{CFL} = 0.2$  in the computations presented to reduce truncation errors  
 441 in the output solutions.

442 **3.2.1** *Detection of shock-development*

443 During wave propagation simulations, the VAM model has the ability to generate  
 444 shocks or moving hydraulic jumps, which are mathematically represented as a weak  
 445 solution of the system of conservations laws involving discontinuities in one or some  
 446 of the flow variables, namely  $q$ ,  $z_s$ ,  $u_1$ ,  $\bar{w}$ ,  $p_1$  and  $p_2$ . The shocks or discontinuity-like

portions of the solutions are generated in the hyperbolic solver of the software. When these portions of the solution are processed by the elliptic solver, the computation of the gradients of the flow variables near such steep fronts produces quantities with an extremely large magnitude, producing numerical instabilities when the Jacobian matrix is formed and inverted, if feasible. The pathological computations described are especially dramatic if one attempts to make a mesh-refinement study of the solution, which is mandatory when presenting numerical solutions. If  $\Delta x$  is progressively reduced to get mesh-independent results, the hyperbolic solver produces sharper discontinuities, given the increased resolution. Processing of these solutions by the elliptic solver encounters not only sharper shocks, but also a smaller  $\Delta x$ , thus much higher gradients, thereby guaranteeing solution crashing. Previous solvers (Cantero-Chinchilla et al., 2018; Gamero et al., 2020) were found to suffer from this issue while dealing with the moving hydraulic jumps experimentally generated in this research, thus a special solver for robust handling of shocks was developed as follows.

The following gradients of the flow variables are used as shock-development sensors:

$$\frac{\partial z_s}{\partial x}, \frac{\partial}{\partial x} \left( \frac{q^2}{gh^2} \right), \frac{\partial}{\partial x} \left( \frac{u_1^2}{g} \right), \frac{\partial}{\partial x} \left( \frac{\bar{w}^2}{g} \right), \frac{\partial}{\partial x} \left( \frac{p_1}{\gamma} \right), \frac{\partial}{\partial x} \left( \frac{p_2}{\gamma} \right), \quad (33)$$

thereby permitting to detect formation of shocks in the output of the hyperbolic solver. A threshold value for the gradients  $\Phi_{thr}$  is defined, which is set at  $75^\circ$  by numerical experimentation for meshes involving  $\Delta x \leq 0.01$  m. A shock is considered formed in the output of any of the flow variables if the corresponding gradient is above this threshold. If this occurs in a cell, the non-hydrostatic flow variables are reset to zero in a bandwidth equal to the stencil used to compute dispersive terms, e.g.,  $2\Delta x$ . This process eliminates the gradients near sharp discontinuities and permits a robust numerical handling. At the initial stage of the dam-break flow generation this process is not applied, given that the initial condition is itself a discontinuity. Thus, the shock-

development detection is applied for  $t > t_0$ , where  $t_0$  is the hydrodynamic time scale [=  $(h_w/g)^{1/2}$ ] (Wu and Wang, 2007), which is of the order of the gate opening time in the experiments.

### 3.2.2 Boundary conditions in dam-break wave experiments

To mimic numerically the experiments conducted in the flume, boundary conditions must be modelled with accuracy. The boundary conditions are incorporated in the mathematical model using ghost cells at boundaries. Computational cells are from  $i = 1$  to  $i = N$ . At the tailwater section of the flume, the closed gate is modeled as a reflective boundary condition, i.e.,  $h_{N+2}=h_{N+1}=h_N$  and  $q_{N+2}=q_{N+1}=-q_N$  (Toro, 2001) with the remaining variables in the VAM model reset to zero. This was found to reproduce well the experimental observations. At the upstream end of the flume, a transmissive boundary condition is implemented for all variables but except the flow depth, i.e.,  $(q, u_1, p_1, p_2, \bar{w}, w^*)_{-1} = (q, u_1, p_1, p_2, \bar{w}, w^*)_2$  and  $(q, u_1, p_1, p_2, \bar{w}, w^*)_0 = (q, u_1, p_1, p_2, \bar{w}, w^*)_1$ . The flow depth in the cells  $i = -1$  and  $i = 0$  is computed by setting energy conservation in the water tank using the experimentally determined time-variation of  $h_{uw}$ :

$$E_{-1} = E_0 = h_{uw} = \frac{1}{100} (0.0015t^4 - 0.032t^3 + 0.1656t^2 - 0.2503t + 0.0393) + h_1, \quad (34)$$

Here  $E$  is the specific energy. Flow depths are thus obtained by analytical inversion of the specific energy diagram as follows:

$$\begin{bmatrix} h_{-1} \\ h_0 \end{bmatrix} = \begin{bmatrix} E_{-1} \left[ \frac{1}{3} + \frac{2}{3} \cos \left( \frac{\gamma_{-1}}{3} \right) \right] \\ E_0 \left[ \frac{1}{3} + \frac{2}{3} \cos \left( \frac{\gamma_0}{3} \right) \right] \end{bmatrix}, \quad (35)$$

where:

$$\begin{bmatrix} \gamma_{-1} \\ \gamma_0 \end{bmatrix} = \begin{bmatrix} \arccos \left[ 1 - \frac{27}{4} \left( \frac{E_{-1}}{h_{c,-1}} \right)^{-3} \right] \\ \arccos \left[ 1 - \frac{27}{4} \left( \frac{E_0}{h_{c,0}} \right)^{-3} \right] \end{bmatrix}, \quad (36)$$

$$\begin{bmatrix} h_{c,-1} \\ h_{c,0} \end{bmatrix} = \begin{bmatrix} \left( \frac{q_2^2}{g} \right)^{1/3} \\ \left( \frac{q_1^2}{g} \right)^{1/3} \end{bmatrix}. \quad (34)$$

The flow depth computed using Eq. (35) corresponds to the subcritical root of the specific energy diagram (Castro-Orgaz and Hager, 2019).

The inviscid part of the VAM equations is not new in the solver presented, but the model equations here include the turbulent stresses. The numerical solver is new, and includes important capabilities not available in a previous model by Cantero-Chinchilla et al. (2018):

1. Inclusion of turbulence by an eddy-viscosity approach.
2. Inclusion of a new module for shock detection in the elliptic step, allowing mesh refinement.
3. Use of the robust MUSCL-Hancock scheme in the hyperbolic step.
4. Dry bed treatment allowed.

The old VAM-2018 model failed during trials to run it for the new experimental conditions presented, due to the formation of hydraulic jumps and the dry bed zones, resulting in a collapse in computations.

1 509 **4. Results**

2  
3  
4 510 *4.1 Benchmark numerical tests*

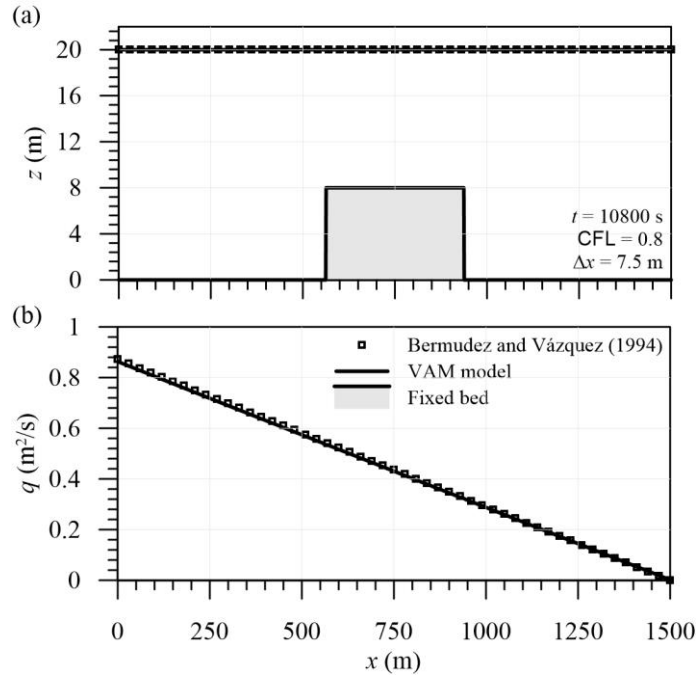
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7 511 Several challenging benchmark tests are selected in this section to evaluate the  
8  
9 512 ability of the VAM model to deal with discontinuous topography, to grant the C-  
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11 513 property and to tackle dry-wet unsteady fronts. First, a tidal wave test over a  
12  
13 514 submerged rectangular step by Bermudez and Vázquez (1994) is used to validate the  
14  
15 515 capability of the model on dealing with discontinuous topography (Zhou et al., 2002;  
16  
17 516 Liang and Marche, 2009). The tidal wave is designed to occur in a 1500-m-long  
18  
19 517 frictionless channel with asymptotic analytical flow solution as follows:

20  
21 518 
$$h = 20 - z_b - 4 \sin \left[ \pi \left( \frac{4t}{86400} + \frac{1}{2} \right) \right], \quad (35)$$

22  
23  
24  
25 519 
$$u_0 = \frac{(x-L)\pi}{5400h} \cos \left[ \pi \left( \frac{4t}{86400} + \frac{1}{2} \right) \right], \quad (36)$$

26  
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28  
29 520 where  $L = 1500$  m. The bed profile is:

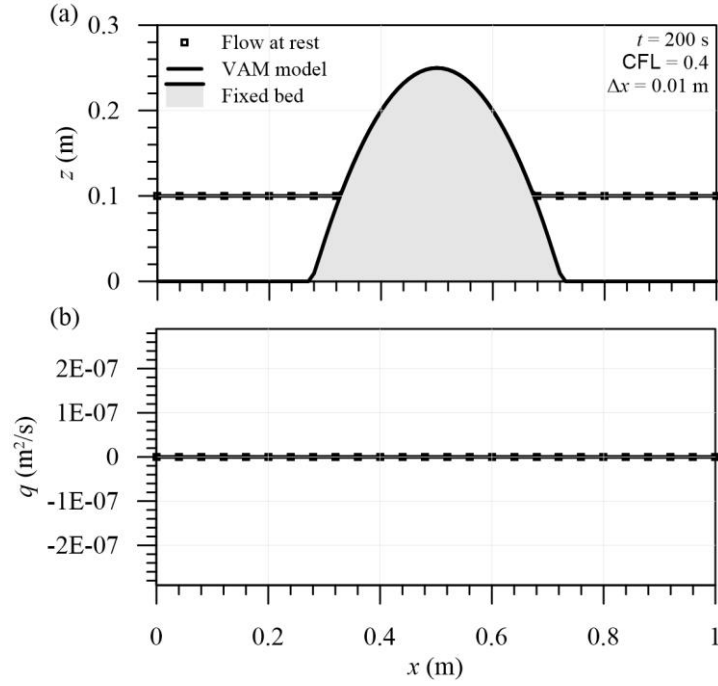
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$$z_b = \begin{cases} 8 & \text{if } |x-750| \leq 187.5, \\ 0 & \text{otherwise.} \end{cases} \quad (37)$$



522

523 **Fig. 6.** Bed discontinuity test using the analytical solutions for tidal waves over a  
 524 submerged vertical step by Bermudez and Vázquez (1994) in comparison with VAM  
 525 model results at  $t = 10800$  s for: (a) flow depth and (b) unit discharge.

526 The test is conducted herein using  $\Delta x = 7.5$  m and  $CFL = 0.8$  until  $t = 10800$  s is  
 527 reached. Equations (38) and (39) are used to set the initial conditions, i.e., substituting  
 528  $t = 0$  s, while Eq. (38) is used to impose the inlet boundary condition at  $x = 0$  m. The  
 529 right end of the channel is modelled using a reflective, closed boundary condition.  
 530 VAM model results for this test are shown in Fig. 6, depicting an excellent agreement  
 531 with the analytical solution in both water surface (Fig. 6a) and unit discharge (Fig. 6b)  
 532 predictions at  $t = 10800$  s, where bed discontinuities did not impact the numerical  
 533 solution.



534

535 **Fig. 7.** Preservation of still water surface at a surface-piercing hump test by Liang and  
 536 Marche (2009) in comparison with VAM model results at  $t = 200$  s for: (a) flow depth  
 537 and (ii) unit discharge.

538 Second, a still water test at a surface-piercing hump (Liang and Marche, 2009), i.e.,  
 539 with dry-wet interfaces, is selected to evaluate the preservation of C-property by the  
 540 proposed VAM model. For this test, the channel is 1-m-long and frictionless, with the  
 541 surface-piercing hump defined by:

$$542 \quad z_b = \max \left[ 0, 0.25 - 5(x - 0.5)^2 \right], \quad (38)$$

543 where the flow is at rest with maximum depth of 0.1 m. The test is conducted herein  
 544 using  $\Delta x = 0.01$  m and  $CFL = 0.4$  until  $t = 200$  s. Figure 7 shows the VAM model  
 545 results for the free surface and the unit discharge after 200 s of computation, where the  
 546 agreement with the static solution at rest is excellent.

547 Finally, the analytic solutions by Thacker (1981) for an oscillating shoreline in a  
 548 parabolic bowl are used to evaluate the ability of the proposed VAM model to tackle

dry-wet fronts in movement over a non-uniform slope (Liang and Marche, 2009; Lai and Khan, 2018). The analytical solutions for the flow depth and the unit discharge in a rectangular channel with parabolic bed profile are (Thacker, 1981; Lai and Khan, 2018):

$$z_s = \max \left[ z_b, -\frac{B^2 \cos(2\omega t) + B^2 + 4Bx\omega \cos(\omega t)}{4g} \right] \quad (39)$$

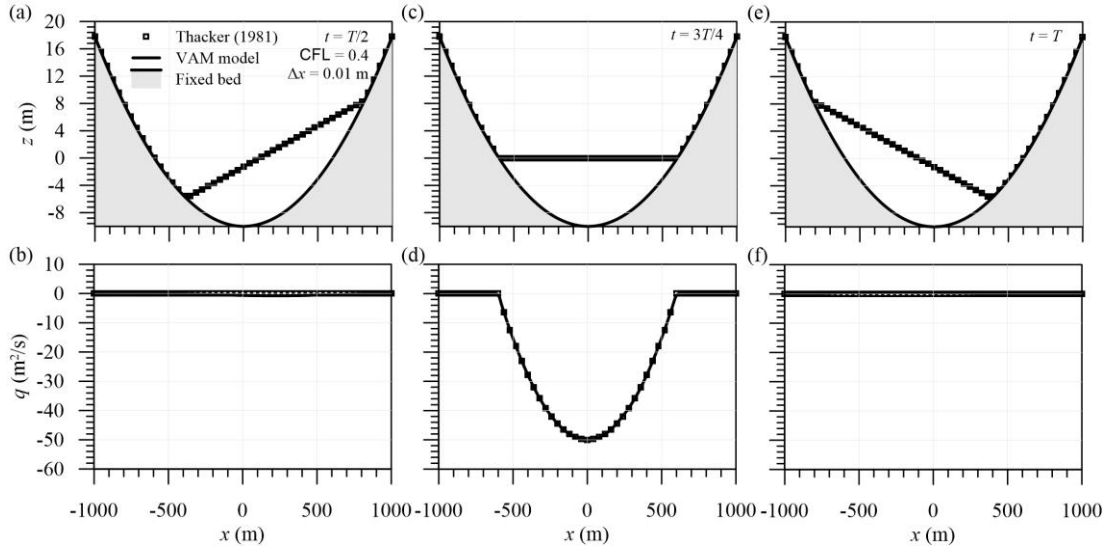
$$q = \max [0, (z_s - z_b)] B \sin(\omega t) \quad (40)$$

with

$$\omega = \frac{2\pi}{T} = \frac{\sqrt{2gh_0}}{l_0} \quad (41)$$

where  $T$  is the period,  $\omega$  is the frequency,  $B$  is a speed parameter, and  $h_0$  and  $l_0$  are definition parameter for the parabolic bed, which is  $z_b = h_0[(x/l_0)^2 - 1]$ . This test is conducted herein using  $\Delta x = 0.01$  m,  $\text{CFL} = 0.4$ ,  $h_0 = 10$  m,  $l_0 = 600$  m and  $B = 5$  m/s until  $t = T = 269$  s. The domain extended to  $x \in [-1000, 1000]$  m, thus yielding the boundaries unaffected by the flow during the computation. Figure 8 shows the VAM model results for flow depth and unit discharge at three different instants: (i) at  $t = T/2$  when the flow reaches its highest position to the right (Figs. 8a and b), (ii) at  $t = 3T/4$  when the flow passes through the horizontal position advancing towards the left end (Figs. 8c and d), and (iii) at  $t = T$  when the flow reaches its highest position to the left (Figs. 8e and f). The VAM model results show an accurate agreement with the analytical solutions at all computed instants, where the moving dry-bed fronts are accurately tackled validating the wetting-drying algorithm in the model.





569

570 **Fig. 8.** Evolution of shorelines over a frictionless parabolic bowl test by Thacker  
 571 (1981) in comparison with VAM model results for flow depth and unit discharge at  
 572 different instants:  $t = T/2$  (a)-(b),  $t = 3T/4$  (c)-(d), and  $t = T$  (e)-(f).

#### 573 4.2 Lee-side waves

574 The ability of the VAM model on reproducing the challenging experimental flow  
 575 tests produced in this work is evaluated in this section. To this end, all tests described  
 576 in section 2.2, Table 1, were simulated using  $\Delta x = 0.01$  m,  $CFL = 0.2$  and  $n = 0.01$   
 577  $\text{ms}^{-1/3}$ . The numerical tests were repeated using  $CFL = 0.1$ , thereby confirming mesh-  
 578 independence of the numerical results. The VAM model results for the tests 1–4 are  
 579 shown in Figs. 10–13, respectively, where four instants were selected to illustrate the  
 580 main phenomena in the transformation of waves in the test series: (i) initial stages of  
 581 dam-break flow generation at  $t = 1$  s or 2 s, (ii) wave reflection at the right end of the  
 582 flume at  $t = 5$  s, (iii) wave interaction with the unsteady hydraulic jump formed at the  
 583 toe of the obstacle at  $t = 8$  s, and (iv) reflected and diffracted waves produced after the  
 584 incoming surge surpassed the obstacle, at  $t = 12$  s.

585 Figure 9 shows the VAM model results in comparison with the experimental data  
 586 for test 1, where  $r = 0.397$ , as well as the Saint Venant equations (dSV) model

1 587 predictions. In test 1, the non-linearity effects are the highest among the test series,  
2 588 also depicting the most challenging unsteady hydraulic jump formed at the toe of the  
3 589 lee slope of the obstacle. The VAM model accurately predicts the initial stages of the  
4 590 dam-break flow as shown in Fig. 9a, which represents the experimental data at  $t = 1$  s.  
5 591 The amplitude of the leading broken wave (Fig. 9a,  $x = 10.7$  m) is, however, slightly  
6 592 overestimated. While the VAM model is able to accurately approximate the advancing  
7 593 bore position, the dSV model anticipates its location (Fig. 10a,  $x = 10.9$  m). Figure 9b  
8 594 represents the experimental data at  $t = 5$  s, after the flow reflection at the right end of  
9 595 the flume. The VAM model results show excellent agreement with the experimental  
10 596 data of the reflected train of waves, the hydraulic jump profile, and the drawdown in  
11 597 the upstream water level as the rarefaction wave advances backwards. The dSV model,  
12 598 however, fails to approximate the reflected train of waves, which is attributed to the  
13 599 lack of dispersive terms. The experimental data at  $t = 8$  s is represented in Fig. 9c,  
14 600 depicting the interaction between the reflected train of waves and the moving-forward  
15 601 hydraulic jump at the toe of the obstacle. While the effect of the rarefaction wave as  
16 602 well as the reflected train of waves near the flume end are accurately predicted by the  
17 603 VAM model, discrepancies between the model results and the data are found in both  
18 604 the interaction zone and the hydraulic jump, with a peak predicted by the VAM model  
19 605 not observed in the experiments. The VAM model computed the dynamic pressures in  
20 606 an elliptic step by solving the relevant system of equations iteratively. In the vicinity  
21 607 of a hydraulic jump, the perturbation parameters determining the pressure field  $p_1$  and  
22 608  $p_2$  are subjected to abrupt changes, and thus inviscid pressure peaks are generated as  
23 609 solution in the elliptic step, thereby resulting also in abrupt wave peaks in the free  
24 610 surface profile at the next time step. Turbulence is incorporated in the model using an  
25 611 eddy-viscosity approach. The mismatch of computations and predictions at  $t = 8$  s  
26 612 indicates that the modeled turbulent stresses are not strong enough to suppress the  
27 613 effect of the inviscid pressure peaks at some time instants. The approximated  
28 614 turbulence closure in the proposed VAM model, which incorporates the bed-  
29 615 dominating turbulence approximation for eddy-viscosity by Ghamry and Steffler

(2002a; b), may be the origin of this misprediction. A more sophisticated approach including turbulence diffusion and production may be needed to characterize the interaction zone of the hydraulic jump and the reflected wave. Finally, Fig. 9d shows the experimental data at  $t = 12$  s in comparison with VAM and dSV model results. Overall, the VAM model is capable of predicting most of the reflected and diffracted waves, leading to some mispredictions especially for the leading wave amplitude in the reflected bore. These discrepancies may stem from the misprediction of the flow interaction around  $t = 8$  s (Fig. 9c), but are unrelated to stability issues of the model. Despite this weakness, the turbulent VAM model prediction of the experimental data by test 1 is accurate as compared to that of the dSV model.  $L$  and  $L^2$  norms for VAM and dSV simulations are given in Table 2.

627

628 **Table 2.**  $L$  and  $L^2$  norms for VAM and dSV simulations in Fig. 9

	$t = 1$ s		$t = 5$ s		$t = 8$ s		$t = 12$ s	
	VAM	dSV	VAM	dSV	VAM	dSV	VAM	dSV
$L(\text{m})$	0.563	0.728	1.573	1.843	1.604	1.841	1.105	1.468
$L^2(\text{m})$	0.092	0.107	0.369	0.380	0.265	0.271	0.140	0.182

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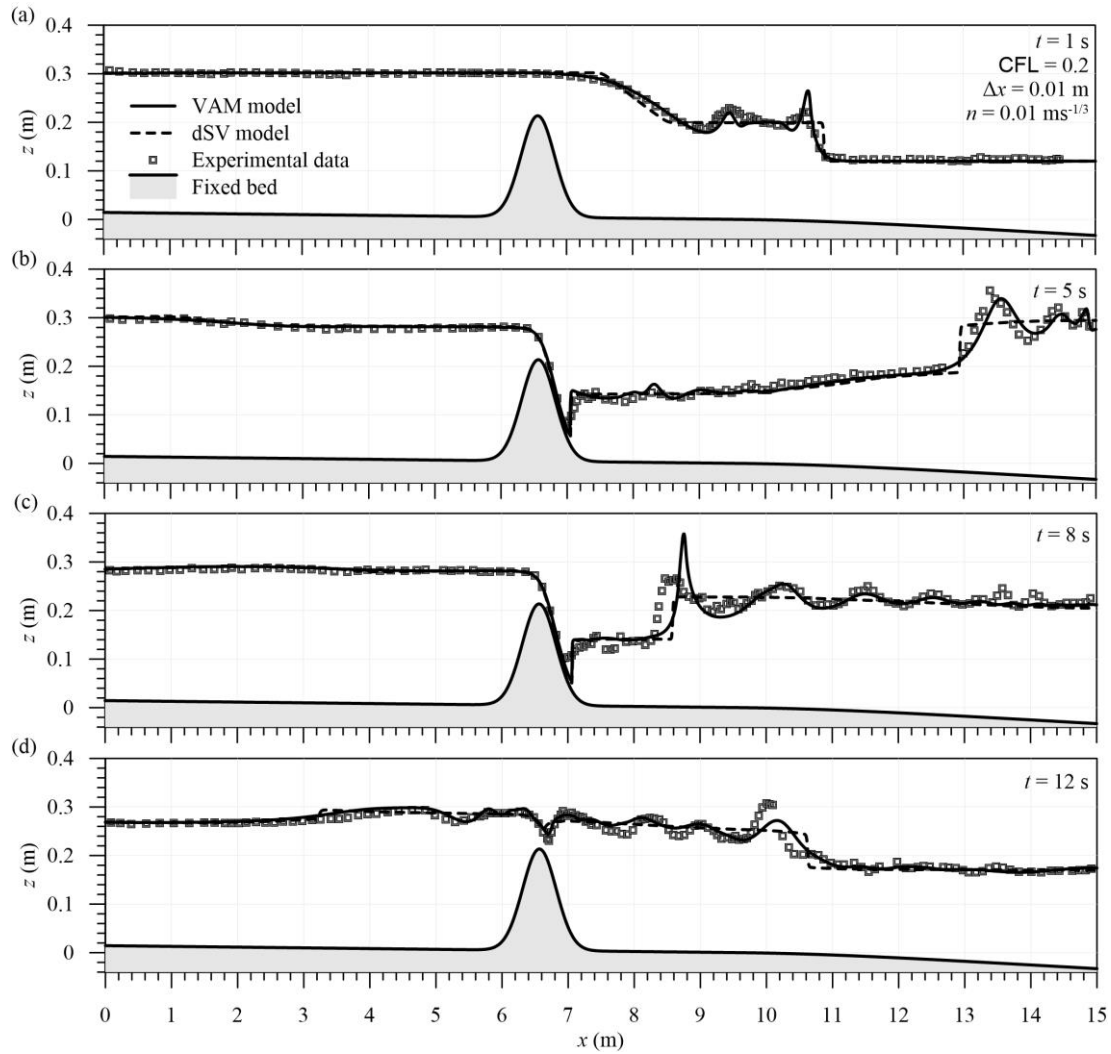
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631 **Fig. 9.** VAM model results for test 1 in comparison with the experimental data and the  
 632 dSV model predictions at: (a)  $t = 1$  s, (b)  $t = 5$  s, (c)  $t = 8$  s, and (d)  $t = 12$  s.

633

634 Figure 10 shows the VAM model results in comparison with the experimental data  
 635 extracted for test 2, where  $r = 0.6$ , as well as the dSV model predictions. In test 2, the  
 636 non-linear effects in the advancing bore are expected to decrease as compared to those  
 637 given by test 1, thus leading to a non-breaking bore at the initial stages of the dam-  
 638 break flow as observed during the experimentation. In addition, during the test, the  
 639 unsteady hydraulic jump was observed to develop a shorter front than that by test 1.

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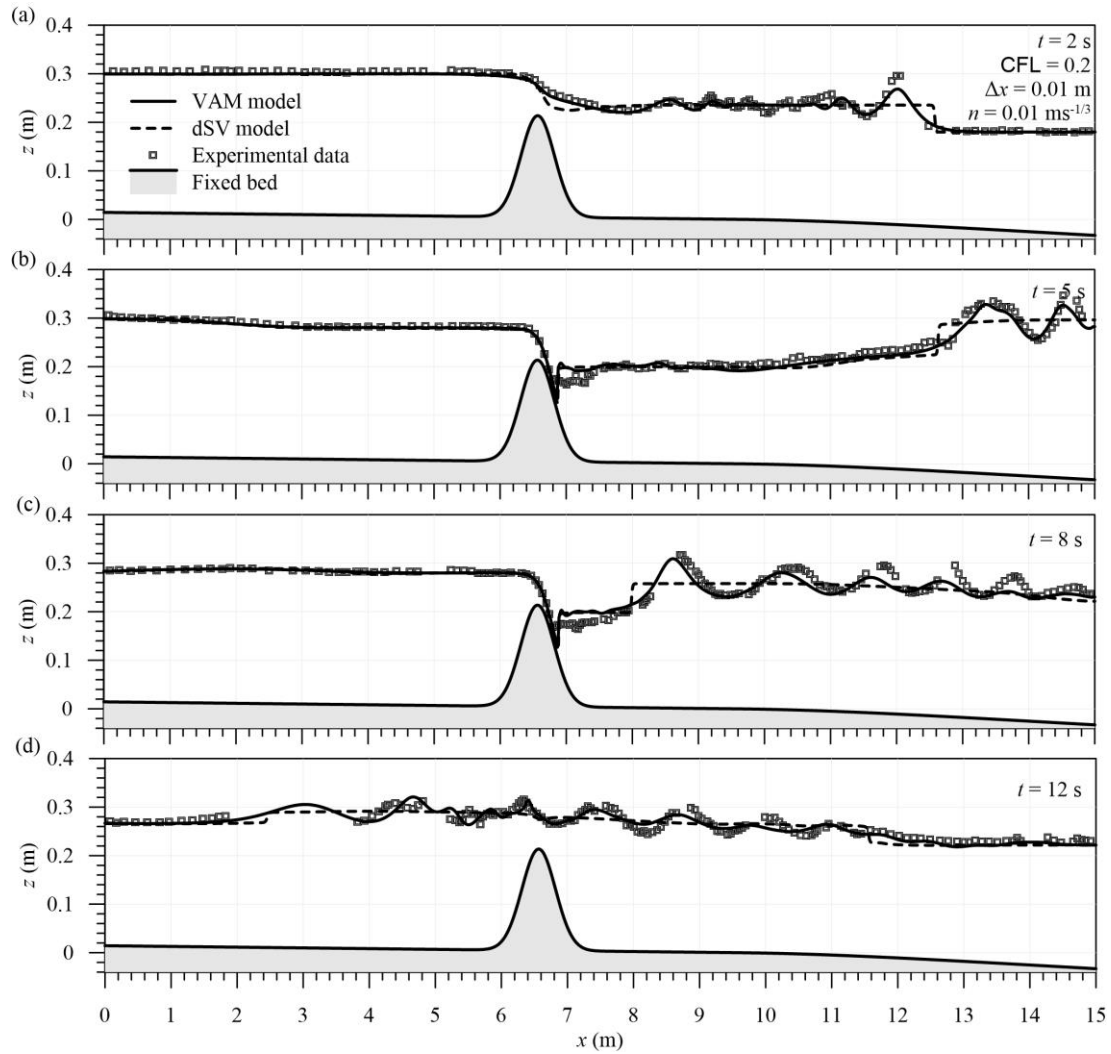
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Figure 10a shows the experimental data at  $t = 2$  s, i.e., the dam-break flow right after the initial stages, in comparison with the VAM and dSV model results. Here, the VAM model provides the results in excellent agreement with the experimental data but for the leading wave amplitude (Fig. 10a,  $x = 12$  m), it is slightly underestimated. In Fig. 10b, the experimental data at  $t = 5$  s is shown, where the dam-break flow has already been reflected at the right end of the flume and the hydraulic jump at the toe of the obstacle begins to develop, in comparison with the results by both models. While the dSV model less predicts the reflected train of waves, the VAM model is able to accurately approximate the experimental data at  $t = 5$  s; however, it leads to an overestimation of the zone of development of the hydraulic jump at the toe of the obstacle. Note that the latter referred zone is mainly turbulence dominated and, thus, the accuracy of VAM model results may suffer from the approximate turbulence closure considered in this work. The overestimation of the initial stages of the hydraulic jump is also evident in Fig. 10c, where the experimental data at  $t = 8$  s is plotted against the VAM and dSV model results. Albeit the two leading waves at the front of the reflected bore are accurately approximated by the VAM model, the amplitude and phase of the train of waves are slightly misinterpreted. However, the impact of this underestimation in the train of waves at  $t = 8$  s is minimal on the approximation of the subsequent test data, as shown in Fig. 10d for the experimental data at  $t = 12$  s in comparison with both models results. The results by the VAM model are overall satisfactory for  $t = 12$  s, where the major discrepancies with respect to the experimental data focus on the amplitude of the train of waves upstream and downstream the obstacle.  $L$  and  $L^2$  norms for VAM and dSV simulations are given in Table 3.

**Table 3.**  $L$  and  $L^2$  norms for VAM and dSV simulations in Fig. 10

	$t = 2$ s		$t = 5$ s		$t = 8$ s		$t = 12$ s	
	VAM	dSV	VAM	dSV	VAM	dSV	VAM	dSV
$L$ (m)	0.784	1.086	1.257	1.776	1.723	2.357	1.333	1.800
$L^2$ (m)	0.092	0.149	0.140	0.209	0.189	0.256	0.135	0.172



666

667 **Fig. 10.** VAM model results for test 2 in comparison with the experimental data and  
 668 the dSV model predictions at: (a)  $t = 2$  s, (b)  $t = 5$  s, (c)  $t = 8$  s, and (d)  $t = 12$  s.

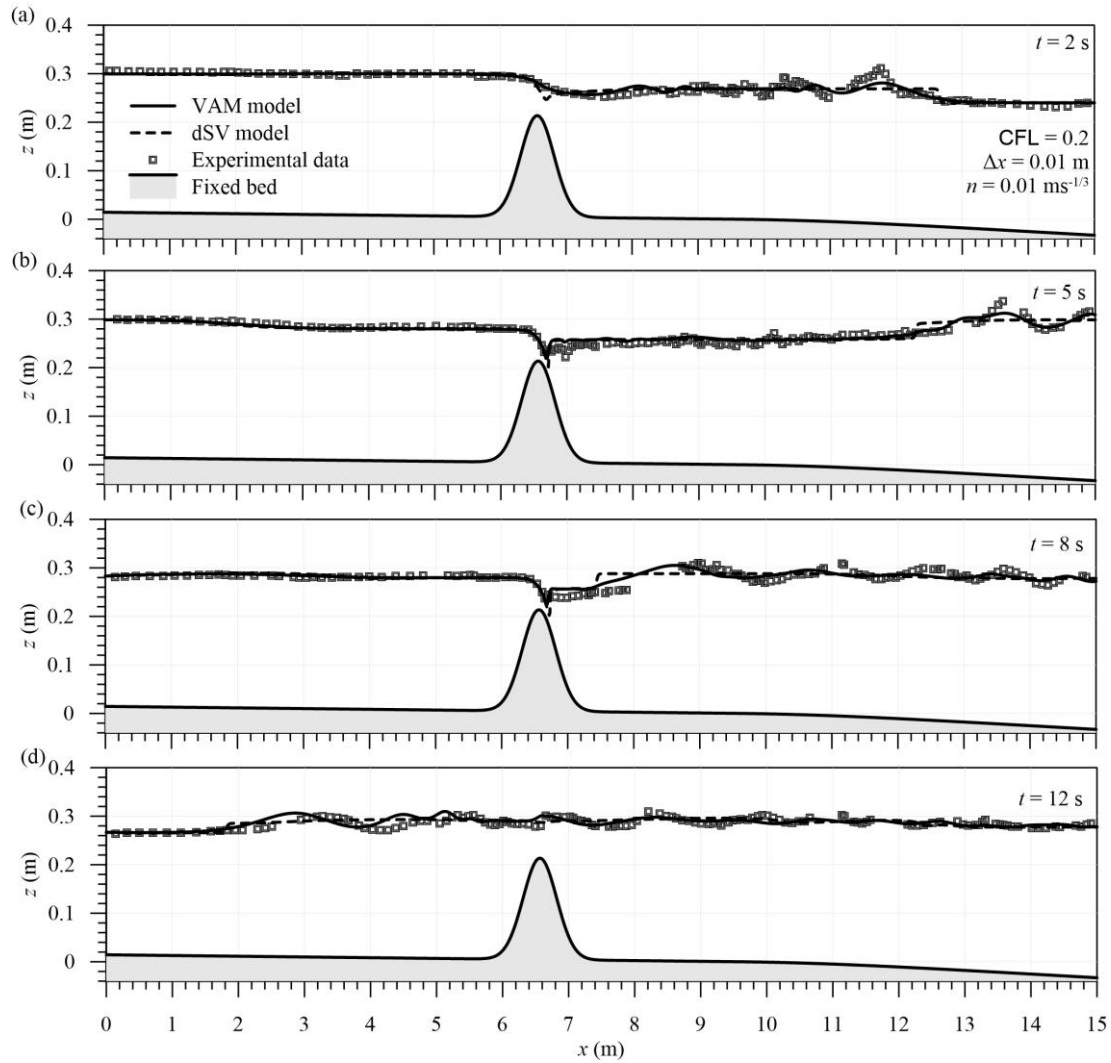
669 Figure 11 shows the VAM model results in comparison with the experimental data  
 670 extracted for test 3, where  $r = 0.8$ , as well as the dSV model predictions. In test 3, the  
 671 wave non-linearity is minimum as the dam-break depth ratio approaches unity. During  
 672 the experiments, the unsteady hydraulic jump at the toe of the obstacle was observed  
 673 to be very close to the crest. The experimental data at  $t = 2$  s is shown in Fig. 11a,  
 674 where the VAM model provides a good approximation of the data in contrast to the  
 675 prediction by the dSV model, which not only anticipates the bore position and

676 mispredicts the train of waves but also depicts a fictitious, incipient hydraulic jump at  
 677 the lee slope of the obstacle. It is noteworthy that during the experimentation in test 3,  
 678 a notable surface water splash was observed after the sluice gate opening, thus  
 679 contaminating the experimental data in the subsequent time instants. In consequence,  
 680 the experimental free surface was hardly tracked at  $t < 2$  s and, therefore, those time  
 681 instants are not shown in Fig. 11. In Fig. 11b, the experimental data at  $t = 5$  s is shown.  
 682 Here, the VAM model is demonstrated to be able to predict all the data. However,  
 683 hydraulic jump and the amplitude of the leading reflected wave are slightly  
 684 overestimated and underestimated, respectively. The experimental data at  $t = 8$  s is  
 685 presented in Fig. 11c, where the VAM model accurately approximates the  
 686 experimental data of the reflected train of waves before they encounter back the  
 687 obstacle. Figure 11d presents the experimental data at  $t = 12$  s, after the transformation  
 688 of the reflected dam-break waves and the interaction back with the obstacle. In line  
 689 with the results for the experimental data at  $t = 12$  s in the experiments 1 and 2, the  
 690 VAM model provides a fair approximation of the experimental data, where only major  
 691 mispredictions are found in the phase of the train of waves upstream the obstacle.  $L$   
 692 and  $L^2$  norms for VAM and dSV simulations are given in Table 4.

694 **Table 4.**  $L$  and  $L^2$  norms for VAM and dSV simulations in Fig. 11

	$t = 2$ s		$t = 5$ s		$t = 8$ s		$t = 12$ s	
	VAM	dSV	VAM	dSV	VAM	dSV	VAM	dSV
$L$ (m)	0.762	0.925	0.817	1.054	0.921	1.138	0.852	0.884
$L^2$ (m)	0.096	0.118	0.093	0.123	0.105	0.139	0.098	0.094

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697

698 **Fig. 11.** VAM model results for test 3 in comparison with the experimental data and  
 699 the dSV model predictions at: (a)  $t = 1$  s, (b)  $t = 5$  s, (c)  $t = 8$  s, and (d)  $t = 12$  s.

700 Figure 12 shows the VAM and dSV model results in comparison with the  
 701 experimental data extracted for test 4, where  $r = 0$ , i.e., dam-break flow under dry bed  
 702 conditions downstream (Castro-Orgaz and Chanson, 2017). In test 4, the parabolic-  
 703 like profile of the dry-bed dam-break bore advanced towards the right end of the flume  
 704 encountering minor bed irregularities at the structural joints, which yielded little free  
 705 surface perturbations. Right after the reflection of the dam-break bore, two unsteady  
 706 hydraulic jumps are formed: (i) a moving-forward one at the toe of the lee slope of the



707 obstacle and (ii) a moving-backward one at the front of the reflected bore. Figure 13a  
708 shows the experimental data at  $t = 1$  s for the dry-bed dam-break waves at the initial  
709 stages, where the VAM model results merely differ from the dSV predictions in the  
710 profile of the rarefaction wave. The experimental data at  $t = 5$  s depicts the first instants  
711 after the bore reflection, where an unsteady hydraulic jump with smooth transition is  
712 developed (Fig. 13b). Here, the VAM model proposed in this study is shown to be  
713 unable to tackle the smooth hydraulic jump front, however showing a direct transition,  
714 which is in line with the predictions of the dSV model due to the shock detection in  
715 this zone. The latter suggests that the turbulence closure may need to be enhanced  
716 Some discrepancies have also been found in the prediction of direct hydraulic jumps,  
717 as shown in Fig. 12c,  $x = 11.3$  m, for the experimental data at  $t = 8$  s. The last  
718 experimental data extracted for test 4 corresponds to  $t = 12$  s (Fig. 12d), where the flow  
719 interaction between the two unsteady hydraulic jumps is shown, leading to a  
720 challenging turbulence-dominated phenomena. However, the VAM model provides a  
721 fair approximation of the test of data showing non-hydrostatic free surface waves after  
722 the hydraulic jump.  $L$  and  $L^2$  norms for VAM and dSV simulations are given in Table  
723 5.

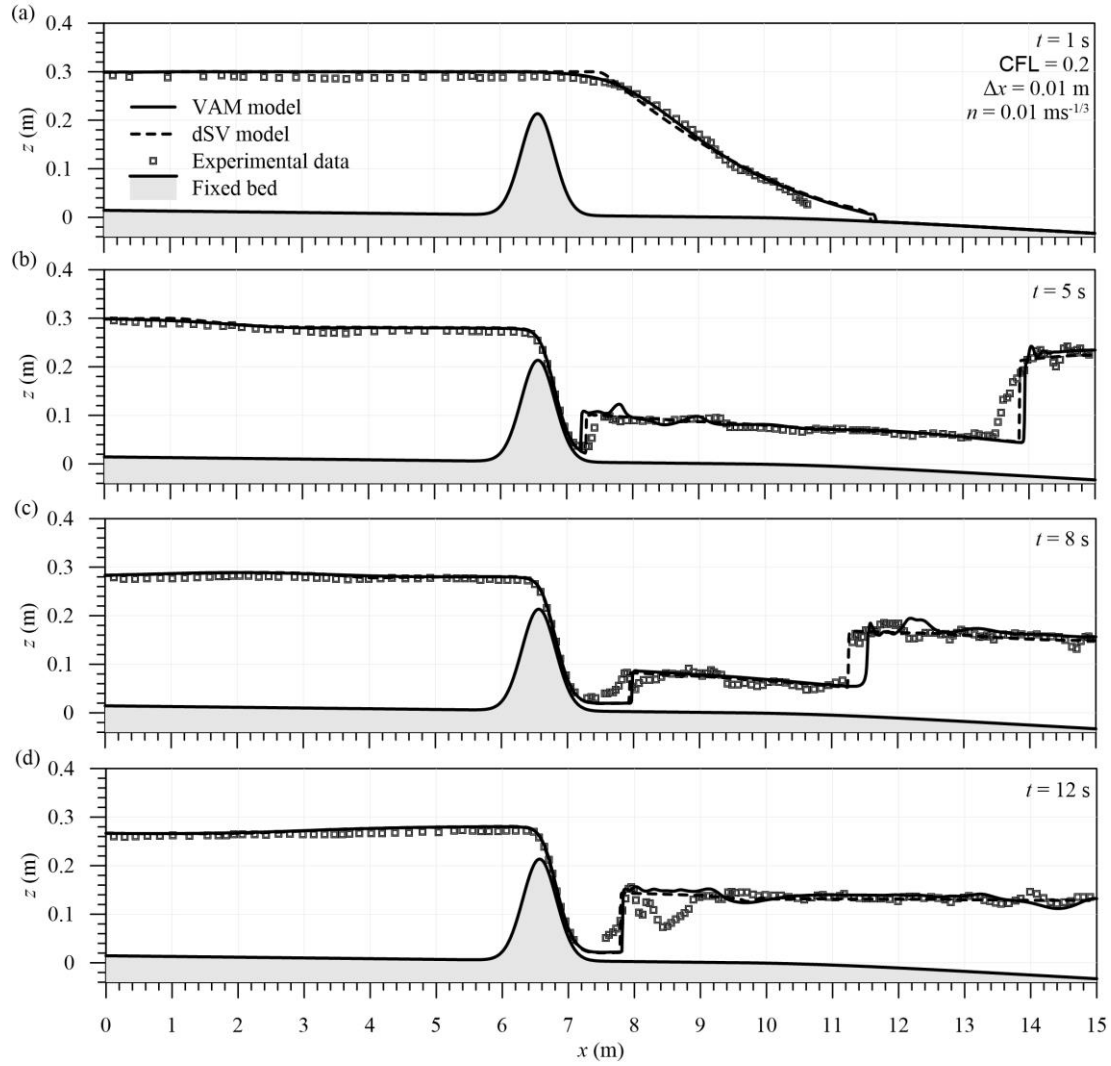
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725 **Table 5.**  $L$  and  $L^2$  norms for VAM and dSV simulations in Fig. 12

	$t = 1$ s		$t = 5$ s		$t = 8$ s		$t = 12$ s	
	VAM	dSV	VAM	dSV	VAM	dSV	VAM	dSV
$L(\text{m})$	0.545	0.729	1.919	1.600	2.193	1.395	1.870	1.471
$L^2(\text{m})$	0.076	0.097	0.342	0.281	0.303	0.163	0.246	0.204

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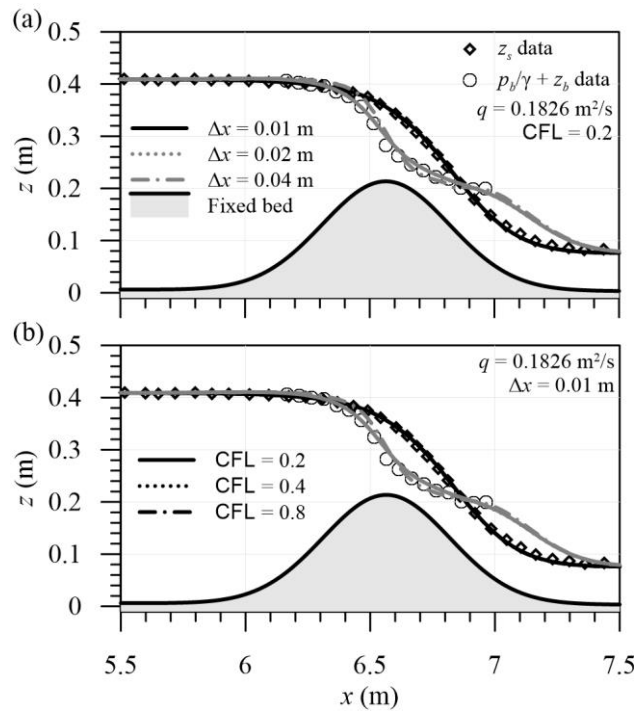
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729 **Fig. 12.** VAM model results for test 4 in comparison with the experimental data and  
 730 the dSV model predictions at: (a)  $t = 1$  s, (b)  $t = 5$  s, (c)  $t = 8$  s, and (d)  $t = 12$  s.

731 *4.3 Steady flow over the obstacle*

732 A steady flow experiment conducted in this work for the maximum discharge  $q =$   
 733  $0.1826 \text{ m}^2/\text{s}$  was used (Figs. 13a and b) to validate the VAM model results over an  
 734 obstacle. The obstacle is a Gaussian profile  $z_{bG} = 0.209 \cdot \exp[-1/2 \cdot ((x - x_{crest})/0.254)^2]$ ,  
 735 where  $z_{bG}$  is the local obstacle height above the flume bed and  $x_{crest}$  is the longitudinal  
 736 location of the crest, installed at  $x_{crest} = 6.565$  m. The experimental obstacle is similar

737 to that of Sivakumaran et al. (1983), but  $q$  is higher in the present experiments, and,  
 738 thus, the degree of non-hydrostaticity of the flow is stronger. In Figure 13, we have  
 739 included the experimental measurements of the free surface profile  $z_s(x)$  and  
 740 piezometric bed pressure head  $(p_b/\gamma+z_b)(x)$  in this obstacle model. Comparison of the  
 741 simulated results for the free surface and bed piezometric pressure head  $p_b/\gamma+z_b$   
 742 obtained from the VAM model in Fig. 13 shows the accuracy of this shallow-water  
 743 formulation predicting the flow features over the obstacle. The mesh-size  
 744 independence of the results was evaluated progressively reducing  $\Delta x$  and CFL.



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746 **Fig. 13.** Comparison of steady flow experiments over the obstacle with numerical  
 747 simulations for the free surface and piezometric bed pressure head

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## 5. Conclusions

A new experimental procedure to investigate wave interaction and flow adjustment over obstacles is presented by constructing a large-scale obstacle model in a flume equipped with a wave-generation mechanism based on a dam-break like set-up. The experiments were used to produce a variety of relevant phenomena over topography, as broken and dispersive undular waves, hydraulic jumps, non-hydrostatic critical flow over a sill crest and wave reflection. In addition to the novelty of the procedure to study flow interaction with obstacles, the experimental database generated is itself of utility for environmental fluid flow modelers, given that it can be directly used as benchmark test cases while testing their models. Steady flow tests were used additionally to determine the dynamic fluid pressures over the obstacle.

A new shallow-water weighted-averaged residual model with the ability to mimic turbulent breaking processes through the formation of shocks or moving hydraulic jumps is presented. This is due to the inclusion of the turbulent velocity profile and Reynolds stresses into the model equations, with a new shock-detection algorithm conferring robustness to the numerical solver. Dispersive effects and non-hydrostatic bed pressures are further tackled by the model given the inclusion of the vertical accelerations. These features make the weighted-averaged residual model presented a suitable tool for environmental modeling of flows over topography with sills.

The turbulent flow model developed reproduces the main features observed during experimentation, namely undular and broken surges, dispersive wave reflection, hydraulic jumps and non-hydrostatic critical flow at sill crest with high non-hydrostatic pressures, with enough accuracy for practical modeling purposes. The dispersionless dSV equations, which is the frequent shallow water flow representation used to study flow adjustment over obstacles, produces only rough estimates or simply does not reproduce the observed experimental phenomena.

The main outcome of this research is a contribution to the physical understanding of the flow adjustment over an obstacle with the new experiments conducted, and by

1 777 producing a new and robust shallow-water solver with capabilities to deal with several  
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3 778 hydraulic phenomena not accounted for in other solvers.  
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## 7 779 **Acknowledgements**

8  
9 780 The work of O. Castro-Orgaz, project leader, was supported by the Spanish projects  
10 781 CTM2017-85171-C2-1-R and PID2020-114688RB-I00, and grant María de Maeztu  
11 782 for Centers and Units of Excellence in R&D (Ref. CEX2019-000968-M). FNCC was  
12 783 partly funded by MCIN/AEI/10.13039/501100011033 and the NextGeneration  
13 784 EU/PRTR through Juan de la Cierva program (IJC2020-042646-I) and a “Selection of  
14 785 Doctoral Researchers” grant by the Junta-de-Andalucía government, Spain (Ref.  
15 786 DOC\_00996). RJB was partly funded by MCIN/AEI /10.13039/501100011033  
16 787 through Juan de la Cierva program (IJC2019-038848-I). We thank the two anonymous  
17 788 reviewers of this work for the comments offered to improve it.  
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## 27 28 29 30 790 **Data availability**

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32 791 The experimental database generated is available as supplementary material in the  
33 792 file “Experiments\_EMS2022.xls”.

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