Fascicle of Management and Technological Engineering ISSUE #1, MAY 2013, http://www.imtuoradea.ro/auo.fmte/

FEM3DD SOFTWARE VERIFICATION

Samuel SANCHEZ-CABALLERO¹, Miguel Angel SELLÉS², Miguel Angel PEYDRO³, David JUAREZ⁴

- ¹ Institute of Design and Manufacturing, Universitat Politècnica de València, Pl. Ferrandiz y Carbonell, s/n; 03801; Alcoy Alicante (Spain), sasanca@dimm.upv.es
- ² Department of Mechanical and Materials Engineering, Universitat Politècnica de València, Pl. Ferrandiz y Carbonell, s/n; 03801; Alcoy Alicante (Spain), miselcan@dimm.upv.es
- ³ Department of Mechanical and Materials Engineering, Universitat Politècnica de València, Pl. Ferrandiz y Carbonell, s/n; 03801; Alcoy Alicante (Spain), mpeydro@mcm.upv.es
- ⁴ Department of Mechanical and Materials Engineering, Universitat Politècnica de València, Pl. Ferrandiz y Carbonell, s/n; 03801; Alcoy Alicante (Spain), djuarez@mcm.upv.es

Abstract—Structural optimization requires a large number of structures to be evaluated. In simultaneous size and shape optimization, where only metaheuristics techniques can be used, the number of evaluations can easily ran into thousands. Renowned and widespread commercial software like ANSYS or MSC Nastran are multipurpose FEM (Finite Element Method) programs have high initialization times which are even higher than the computing ones. This has a vital influence on the algorithm performance, especially in population based metaheuristic techniques. In order to avoid this issue, lighter FEM software or even embedded FEM code should be used. Among the Openware FEM code, the software FEM3DD can be highlighted. From its features, it is worthy to stand out: fast initialization, multi-OS open source code (GNU license), command line operation, CSV and Matlab files reading, and well documented.

Keywords—Structural, optimization, FEM3DD, verification

I. INTRODUCTION

 $T^{\rm HE}$ price increase of raw materials has forced the engineers to reduce the weight of the structures. This has brought the flowering of a new set of optimization algorithms.

Traditional optimization algorithms do not use simultaneous size and shape optimization, which biases solutions towards local optima. In contrast, new metaheuristics techniques as Genetic Algorithms [1], Evolutionary Strategies [2], Particle Swarm Optimization [3] or Ant Colony Optimization [4, 5] are capable to handle both size and shape variables simultaneously in an easy way. These techniques can lead to lighter structures than traditional Linear Programming [6], Non Linear Programming, Branch and bound [7] algorithms.

The main drawback of this new techniques is that are population based and they require from a large number of evaluations which run from hundreds to thousands, and even millions depending on the structure complexity. This requires the computing time to be very low as on the contrary they become pointless.

Renowned and widespread commercial software like ANSYS or MSC Nastran are multipurpose FEM

programs have high initialization times which are even higher than the computing ones. For example, the lighter Ansys v.12.1 startup from command line requires 2 seconds which is usually higher than the structure computing time. To avoid this drawback it is necessary to use a lighter FEM software or even embedded FEM code.

There are a lot of free FEM utilities, some of them even with free open source. Among them FEM3DD is a good choice. The main features of this software are:

- 1) Fast initialization time for binaries.
- 2) Binaries for windows, OS X and Linux.
- 3) Open source code with GNU license [8].
- 4) Multiple load cases in each analysis with gravity loading, partial trapezoidal loads, thermal loads, and prescribed displacements.
- 5) Shear deformation effects on geometric stiffness are included.
- 6) Matlab and spreadsheet interface, graphical output via Gnuplot.

The software uses very common FEM routines, several of them from Numerical Recipes, and is being used by lots of researchers, but the documentation does not show a numerical verification with other renowned software or examples.

The aim of this paper is to do such verification through the evaluation of a set of different structures.

II. EVALUATED STRUCTURES

A. 32 bar roof structure

The structure is an isostatic structure with two supports with point loads on the upper joints.

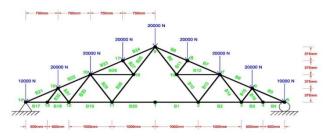


Fig. 1. Geometry of the 32 bar roof truss.

Fascicle of Management and Technological Engineering ISSUE #1, MAY 2013, http://www.imtuoradea.ro/auo.fmte/

Mechanical properties.

Young Modulus. E = 210000 MPa

Poisson coefficient. N = 0.296

Shear Modulus. G = 81000 MPa

Density. $\rho = 7.85E-9 \text{ kg/mm}^3$

Upper and lower bars properties:

Profile: T 40

Transverse section. (A_x) : 377 mm²

Y axis moment of inertia (I_{YY}) : 52800 mm⁴ Z axis moment of inertia (I_{ZZ}): 25800 mm⁴ Polar moment of inertia (J_{XX}): 78600 mm⁴ Vertical and diagonal bars properties:

Profile: L 40x4

Transverse section. (A_x) : 308 mm²

Y axis moment of inertia (I_{YY}): 70900 mm⁴ Z axis moment of inertia (I_{ZZ}): 18600 mm⁴ Polar moment of inertia (J_{XX}): 89500 mm⁴

B. Ten bar truss

The structure is very well known in structural optimization and is used as benchmark problem by the researchers to test their algorithms with more than 55 researchers [9] during the last forty years. Fig. 2 shows the best optimum reported with a reduced number of bars.

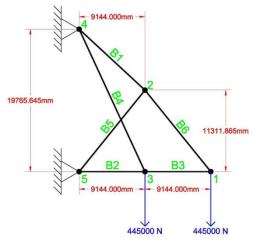


Fig. 2. Geometry of the optimum ten bar truss.

Mechanical properties:

Young Modulus. E = 68900 MPa Poisson coefficient. N = 0.33

Density. $\rho = 2,65851E-9 \text{ kg/mm}^3$

Area B1: 8709,66 mm² Area B2: 8709,66 mm² Area B3:. 4658,0552 mm² Area B4: 3703,2184 mm² Area B5: 8709,66 mm²

C. 22 bar truss

This is also a well-known structure in structural optimization [10].

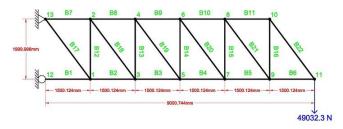


Fig. 3. Geometry of the 22 bar truss.

Mechanical properties.

Young Modulus. E: 68900 MPa Density. $\rho = 2,65851E-7 \text{ kg/mm}^3$

Area B1: 1729,029 mm²

Area B2: 1729,029 mm²

Area B3:.1438,707 mm²

Area B4: 1096,772 mm²

Area B5: 690,321 mm²

Area B6: 515,483 mm²

Area B7: 1438,707 mm²

Area B8: 1096,772 mm²

Area B9: 954,837 mm²

Area B10: 515,483 mm²

Area B11: 279,354 mm²

Area B12: 690,321 mm²

Area B13:.690,321 mm²

Area B14: 690,321 mm² Area B15: 690,321 mm²

Area B16: 690,321 mm²

Area B17: 412,257 mm²

Area B18:. 412,257 mm²

Area B19: 412,257 mm²

Area B20: 412,257 mm²

Area B21: 412,257 mm²

Area B22: 412,257 mm²

D.Two supported beam

The structure is an isostatic beam with two supports with several loads and moments.

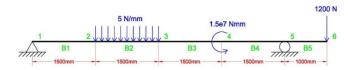


Fig. 4. Geometry of the two supported beam.

Mechanical properties.

Young Modulus. E = 210000 MPaPoisson coefficient. N = 0.296Shear Modulus. G = 81000 MPa

Density. $\rho = 7.85E-9 \text{ kg/mm}^3$

Profile. IPN 200

Transverse section. (A_x) : 3350 mm²

Y axis moment of inertia (I_{YY}): 21400000 mm⁴ Z axis moment of inertia (I_{ZZ}): 1170000 mm⁴ Polar moment of inertia (J_{XX}): 22570000 mm⁴

Fascicle of Management and Technological Engineering ISSUE #1, MAY 2013, http://www.imtuoradea.ro/auo.fmte/

III. FEM MODELING

The computing results for verification were obtained using Ansys APDL v12.1.

For the 32 bar roof structure a BEAM54 element with a mesh of ten divisions per bar was used.

For the ten bar truss a LINK1 element was used.

For the 22 bar truss a LINK1 element was used.

For the two supported beam a BEAM3 element with a mesh of twenty divisions per bar was used.

The results given by FEM3DD are: joint loads and displacements, frame element forces and moments, and reactions. So, stresses are not computed by the program and must be subsequently computed.

Following the formulae used for stress computation are shown:

A. Normal axial stress:

$$\sigma_{A} = \frac{N_{X}}{A_{Y}} \tag{1}$$

Where:

 σ_A is the axial normal stress.

 N_X is the axial force.

A_X is the transversal area considered

B. Normal asymmetric bending stress (compression side):

$$\sigma_{F_{-C}} = \frac{\left| \mathbf{M}_{Y} \right| \cdot \mathbf{y}_{C}}{\mathbf{I}_{Y}} + \frac{\left| \mathbf{M}_{Z} \right| \cdot \mathbf{z}_{C}}{\mathbf{I}_{Z}}$$
 (2)

Where:

 σ_{F_C} is the bending compressive stress due to bending

 M_Y is the bending moment around the local Y axis.

 I_Y is the moment of inertia around the local Y axis

y_C is the Y axis distance from the neutral axis to the furthest compression point

 M_Z is the bending moment around local Z axis. I_Z is the moment of inertia around the local Z axis

 z_C is the Y axis distance from the neutral axis to the furthest compression point

C. Normal asymmetric bending stress (tensile side)

$$\sigma_{\mathbf{F}_{-\mathbf{T}}} = \frac{\left| \mathbf{M}_{\mathbf{Y}} \right| \cdot \mathbf{y}_{\mathbf{T}}}{\mathbf{I}_{\mathbf{Y}}} + \frac{\left| \mathbf{M}_{\mathbf{Z}} \right| \cdot \mathbf{z}_{\mathbf{T}}}{\mathbf{I}_{\mathbf{Z}}}$$
(3)

Where:

 σ_{F_T} is the bending tensile stress due to bending y_T is the Y axis distance from the neutral axis to the farest compression point

 z_T is the Y axis distance from the neutral axis to the farest tensile point

D. Average Shear stress along Yaxis:

$$\tau_{XY} = \frac{|\mathbf{V}_{Y}|}{\mathbf{A}_{SV}} + \frac{|\mathbf{T}_{X}|}{\mathbf{C}} \tag{4}$$

Where:

 τ_{xy} is the shear stress along Y axis.

 V_Y is the Y axis shear force.

A_{SY} is the Shear Area (Collignon Area).

 T_X is the torsional moment.

C is the torsion shear constant.

E. Average Shear stress along Zaxis:

$$\tau_{XZ} = \frac{|\mathbf{V}_{\mathbf{Z}}|}{\mathbf{A}_{SZ}} + \frac{|\mathbf{T}_{\mathbf{X}}|}{\mathbf{C}} \tag{5}$$

Where:

 τ_{xz} is the shear stress along Z axis.

 V_Z is the Z axis shear force.

A_{SZ} is the Shear Area (Collignon Area).

IV. RESULTS AND DISCUSSION

The verification of the FEM3DD code was done by comparing the computed solutions with the Ansys computed ones. The selected results to make the comparison where: support reactions, joint displacement and bar compound stress.

Once the solutions were computed, the relative error was computed:

$$\varepsilon(\%) = \left| \frac{\text{value}_{\text{ansys}} - \text{value}_{\text{frame3DD}}}{\text{value}_{\text{ansys}}} \right| \cdot 100$$
 (6)

Following the results for all the structures are shown.

A. 32 bar roof structure

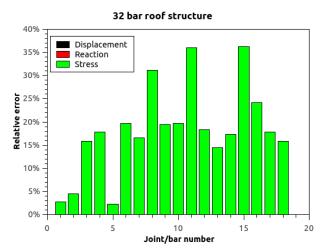


Fig. 5. Relative error of the 32 bar roof structure.

Fascicle of Management and Technological Engineering ISSUE #1, MAY 2013, http://www.imtuoradea.ro/auo.fmte/

As can be seen the relative error of displacement and reaction is negligible. However, the stress solution shows a large deviation, up to 38%. This can be due to the fact that the stress computation through equations from (1) to (5) is a rough approximation to the real solution.

B. 10 bar truss

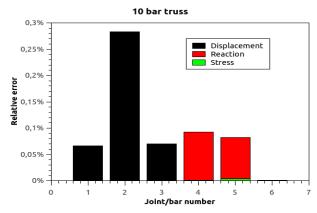


Fig. 6. Relative error of the 10 bar truss.

As Fig. 6 shows the error is negligible.

C. 10 bar truss

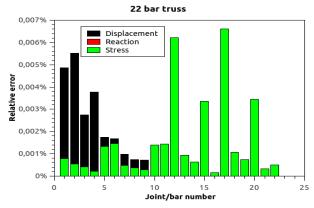


Fig. 7. Relative error of the 22 bar truss.

As Fig. 7 shows the error is negligible.

D.Two supported beam

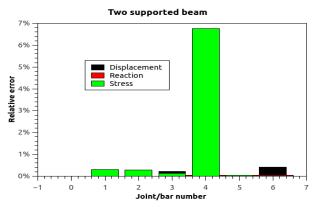


Fig. 8. Relative error of the 22 bar truss.

Like the roof structure, the stress relative error is significant, due to the same effects described before. The reaction and displacement error is negligible.

V.CONCLUSIONS

According to the previous analysis, the displacement and reaction differences regarding to Ansys results is negligible. However, the stress results can have a large divergence when working with beam elements due to the fact that the computed results during post-processing are a rough approximation to the real phenomena.

So, when working with beam elements, the computed stresses through FEM3DD should be affected by a safety factor of about 1.5

ACKNOWLEDGMENT

The author thanks Professor Henri P. Gavin, from the Department of Civil and Environmental Engineering of the Duke University, for the development of FEM3DD and making the code open for everybody.

REFERENCES

- D. E. Goldberg and M. Samtani, "Engineering Optimization via Genetic Algorithm," Ninth Conference on Electronic Computation, pp. 1-9, 1986.
- [2] G. A. Ebenau, J. B. Rottschäfer, and G. B. Thierauf, "An advanced evolutionary strategy with an adaptive penalty function for mixed-discrete structural optimisation," *Advances in Engineering Software*, vol. 36, pp. 29-38, 2005.
- [3] J. F. Schutte and A. A. Groenwold, "Sizing design of truss structures using particle swarms," Structural and Multidisciplinary Optimization, vol. 25, pp. 261-269, 2003.
- [4] A. Kaveh and S. Talatahari, "Hybrid Algorithm of Harmony Search, Particle Swarm and Ant Colony for Structural Design Optimization," in *Harmony Search Algorithms for Structural Design Optimization*. vol. 239, Z. Geem, Ed., ed: Springer Berlin / Heidelberg, 2009, pp. 159-198.
- [5] A. Kaveh and S. Talatahari, "Particle swarm optimizer, ant colony strategy and harmony search scheme hybridized for optimization of truss structures," *Computers & Structures*, vol. 87, pp. 267 -283, 2009.
- [6] C. Y. Sheu and L. A. Schmit Jr., "Minimum weight design of elastic redundant trusses under multiple static loading conditions," *AIAA Journal*, vol. 10, pp. 155-162, 1972.
- [7] U. T. Ringertz, "A branch and bound algorithm for topology optimization of truss structures," *Eng. Opt.*, vol. 10, pp. 111-124, 1986.
- [8] H. P. Gavin. http://sourceforge.net/projects/frame3dd/.
- [9] S. Sanchez-Caballero, "Structural and topological optimization of morphologically undefined structures using genetic algorithms," Phd thesis, 2012.
- [10] F. Erbatur, O. Hasançebi, I. Tütüncü, and H. Kiliç, "Optimal design of planar and space structures with genetic algorithms," *Computers* \& Structures, vol. 75, pp. 209-224-, 2000.