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Additional Information

1 **Assessment of the Performance of a Modified USBR Type II Stilling**  
2 **Basin by a Validated CFD Model**

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16 **ABSTRACT**

17 The adaptation of existing dams is of paramount importance to face the challenge posed by climate  
18 change and new legal frameworks. Thus, it is crucial to optimise the design of stilling basins so that a  
19 reduction in the hydraulic jump dimensions is achieved without jeopardising the energy dissipation in  
20 the structure. A numerical model was developed to simulate a United States Bureau of Reclamation  
21 Type-II basin. The model was validated with a specifically-designed physical model and then was used  
22 to simulate and test the performance of the basin after adding a second chute blocks row. The results  
23 showed a reduction in the hydraulic jump dimensions in terms of the sequent depths ratio and the roller  
24 length, which were respectively 2.5% and 1.4% lower in the modified design. These results would  
25 allow an estimated increase of the discharge in the basin close to the 10%. Furthermore, this new

26 design showed a higher efficiency, with a 1.2% increase in this parameter. Consequently, the  
27 modifications proposed for the basin design suggest an improved performance of the structure. The  
28 issue of the hydraulic jump length estimation is also discussed, introducing and comparing different  
29 approaches. These methods follow a structured and systematic procedure and show consistent results  
30 for the developed models.

31 *Keywords:* Flow-structure interactions; hydraulic jumps; hydraulic structure design & management;  
32 RANS models; USBR Type II stilling basin; chute blocks.

### 33 **INTRODUCTION**

34         Adaptation to new scenarios posed by climate change effects and increasing society demands  
35 regarding hydraulic structures security leads to consider higher potential discharges in the design of  
36 large dams (Carrillo et al., 2020; Macián-Pérez, García-Bartual, et al., 2020). These more demanding  
37 conditions are in line with new legal frameworks and design guidelines recently developed (Ministerio  
38 para la Transición Ecológica y el Reto Demográfico, 2018). In such context, adaptation of already  
39 existing hydraulic structures becomes a key issue of enormous engineering importance. There is  
40 particular interest in the adaptation process of large dams to higher discharges than those originally  
41 considered, in which case the energy dissipation structure constitutes the most challenging part both  
42 from a technical and an economic perspective (Fernández-Bono & Vallés-Morán, 2006).

43         The use of typified stilling basins for energy dissipation purposes in large dams is widely  
44 spread. The design of a majority of these basins dates back many decades and has remained unaltered  
45 since then (Hager, 1992). The more demanding requirements previously mentioned, together with the  
46 development of new modelling approaches (Valero et al., 2019; Viti et al., 2019), justify the in-depth  
47 study of the flow taking place in energy dissipation structures. This will guide future design  
48 modifications, in order to optimise their hydraulic performance.

49         A better understanding of the flow in typified stilling basins requires a deep insight into the  
50 hydraulic jump phenomenon. The hydraulic jump is defined as the abrupt transition from supercritical  
51 to subcritical flow in open channels. This phenomenon involves strong velocity and pressure  
52 fluctuations, intense air entrainment and significant energy dissipation. It is precisely the latter feature  
53 what motivates their use in stilling basins. The so called Classical Hydraulic Jump (CHJ) is the one  
54 occurring in a horizontal, rectangular, prismatic, smooth channel. It has been investigated for almost

55 two centuries (Chanson & Gualtieri, 2008; Hager, 1992). First studies focused on basic characteristics  
56 like the sequent depths ratio or the free surface profile (Bakhmeteff & Matzke, 1936; Bélanger, 1841).  
57 Later, internal features of the hydraulic jump such as the pressure field or the velocity distribution were  
58 approached (McCorquodale & Khalifa, 1983; Rajaratnam, 1965). During the last decades, the turbulent  
59 characteristics of the phenomenon were brought into the spotlight (Wang & Chanson, 2015; Jesudhas  
60 et al., 2018; Toso & Bowers, 1988), together with the study of the aeration (Chanson & Brattberg,  
61 2000; Chanson & Gualtieri, 2008; Murzyn et al., 2005). The efforts devoted to the study of the CHJ  
62 have significantly contributed to an increased knowledge of the phenomenon, which is constantly  
63 growing. However, the inherent complexity of the hydraulic jump requires further research to achieve a  
64 full understanding of the phenomenon. In particular, the study of the hydraulic jump developed in  
65 stilling basins has not received as much attention as the CHJ, despite of its practical interest (Valero et  
66 al., 2019).

67 In spite of the traditional experimental approach to the study of the hydraulic jump,  
68 Computational Fluid Dynamics (CFD) techniques also constitute a useful tool with undoubtedly  
69 increasing potential, as the computational power becomes larger. In fact, the complementary nature of  
70 both techniques enables a desirable double modelling approach for hydraulic engineering problems.  
71 CFD techniques have been used to successfully simulate CHJ in terms of its internal characteristics and  
72 the aeration features (Bayon et al., 2016; Macián-Pérez, Bayón, et al., 2020; Witt et al., 2015).  
73 Furthermore, some numerical studies have addressed the hydraulic jump developed in stilling basins,  
74 analysing its characteristics and the influence of the energy dissipation devices (Carvalho et al., 2008;  
75 Macián-Pérez, García-Bartual, et al., 2020; Valero et al., 2018). Nevertheless, CFD techniques still  
76 present some limitations when simulating complex hydraulic phenomena (Blocken & Gualtieri, 2012;  
77 Bombardelli, 2012). Therefore, the support of physical modelling to provide validated numerical  
78 models remains of paramount importance (Valero et al., 2019).

79 The present study approaches the performance of a United States Bureau of Reclamation  
80 (USBR) Type-II stilling basin using a CFD numerical model. This model was validated with the  
81 experimental data collected in a physical model designed for this purpose (Macián-Pérez, Vallés-  
82 Morán, et al., 2020). According to Hager (1992), stilling basins provide an enhanced energy dissipation  
83 and shorter and more stable hydraulic jumps, when compared to CHJ. Consequently, the performance  
84 of the basin was tested focusing on basic geometrical features, such as the sequent depths ratio and the

85 roller and hydraulic jump lengths, as well as the energy dissipation efficiency. These characteristics  
86 were compared with those obtained for CHJ in previous studies (Bayon et al., 2016; Hager, 1992;  
87 Hager et al., 1990; Hager & Bremen, 1989; Schulz et al., 2015). In order to deepen in the analysis of  
88 the model developed, literature results regarding stilling basin studies were also included (Macián-  
89 Pérez, García-Bartual, et al., 2020; Padulano et al., 2017; Peterka, 1978). Despite this research focuses  
90 on geometrical characteristics of the hydraulic jump, it is important to highlight that there are some  
91 other relevant features to analyse the performance of the basin, such as the bottom pressure, that have  
92 been addressed by recent studies (Stojnic et al., 2021; 2020). Finally, a modified design of the stilling  
93 basin was proposed and included in the analysis to assess a possible optimisation of the structure  
94 performance.

## 95 **NUMERICAL MODEL**

96 The CFD numerical models presented in this study were developed using the commercial software  
97 FLOW-3D<sup>®</sup>, version 11. The results provided by this code are based on the Navier-Stokes equations  
98 written in their form for incompressible fluids:

$$99 \quad \nabla \mathbf{u} = 0 \quad (1)$$

$$100 \quad \frac{\partial \mathbf{u}}{\partial t} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \mathbf{u} + \mathbf{f}_b \quad (2)$$

101 where  $\mathbf{u}$  is the velocity,  $t$  is the time,  $\rho$  is the fluid density,  $p$  the pressure and  $\nu$  the fluid kinematic  
102 viscosity. Finally,  $\mathbf{f}_b$  accounts for the body forces. In particular, FLOW-3D<sup>®</sup> uses the Finite Volume  
103 Method (FVM) (McDonald, 1971) for the spatial discretisation of the conservation laws. In regards  
104 with the time-step, its size is automatically adjusted, using a Courant-type stability criterion to  
105 minimise numerical divergence risk.

## 106 **Turbulence modelling**

107 The flow governing equations were numerically solved through a Reynolds Averaged Navier-Stokes  
108 (RANS) approach. This approach has proved to be efficient regarding computational times and  
109 resources for real-life engineering applications, as compared to other methods such as the Direct  
110 Numerical Simulation (DNS) or the Large Eddy Simulation (LES) (Bayon et al., 2016; Viti et al.,

111 2019). Nevertheless, the averaging process bound to the RANS approach leads to the well-known  
 112 Closure Problem. Accordingly, a turbulence model is required to estimate the eddy viscosity that  
 113 results from the approach.

114 To this end, the RNG  $k$ - $\varepsilon$  model (Yakhot et al., 1992) was employed. This two-equation  
 115 turbulence model addresses the transport of the turbulent kinetic energy ( $k$ ) and its dissipation rate ( $\varepsilon$ ).  
 116 Statistical methods allow deriving the averaged equations for the turbulence quantities, in contrast with  
 117 the traditional  $k$ - $\varepsilon$  model. The equations used to model the transport of  $k$  and  $\varepsilon$  are:

$$118 \quad \frac{\partial}{\partial t}(\rho k) + \frac{\partial}{\partial x_i}(\rho k u_i) = \frac{\partial}{\partial x_j} \left[ \left( \mu + \frac{\mu_t}{\sigma_k} \right) \frac{\partial k}{\partial x_j} \right] + P_k - \rho \varepsilon \quad (3)$$

$$119 \quad \frac{\partial}{\partial t}(\rho \varepsilon) + \frac{\partial}{\partial x_i}(\rho \varepsilon u_i) = \frac{\partial}{\partial x_j} \left[ \left( \mu + \frac{\mu_t}{\sigma_\varepsilon} \right) \frac{\partial \varepsilon}{\partial x_j} \right] + C_{1\varepsilon} \frac{\varepsilon}{k} P_k - C_{2\varepsilon} \rho \frac{\varepsilon^2}{k} \quad (4)$$

120 where  $x_i$  and  $x_j$  are the coordinates in the  $i$  and  $j$  axes respectively,  $\mu$  is the fluid dynamic viscosity and  
 121  $\mu_t$  the turbulent dynamic viscosity, whereas  $P_k$  is the production of turbulent kinetic energy. Finally the  
 122 terms  $\sigma_k$ ,  $\sigma_\varepsilon$ ,  $C_{1\varepsilon}$  and  $C_{2\varepsilon}$  are parameters whose values are reported in Yakhot et al. (1992). The RNG  $k$ - $\varepsilon$   
 123 turbulence model has proved its efficiency for hydraulic engineering applications. Its performance has  
 124 been compared with other turbulence models when analysing both, a CHJ (Bayon et al., 2019) and the  
 125 hydraulic jump developed in a stilling basin (Macián-Pérez et al., 2019).

## 126 **Free surface modelling**

127 The modelling and tracking of the free surface was approached on the basis of the Volume Of Fluid  
 128 (VOF) method (Hirt & Nichols, 1981). Accordingly, the variable Fraction of Fluid ( $F$ ) was used to  
 129 determine the fractional volume of water in each cell. This variable reaches a value of 1 when the  
 130 corresponding cell is completely full and a value of 0 when it is empty. The free surface is then lying  
 131 along cells with values of  $F$  between 0 and 1. In these terms, the free surface elevation is computed as  
 132 the coordinate of the free surface in the topmost fluid element in a vertical column. The evolution of  $F$   
 133 throughout the domain is solved through the equation:

$$134 \quad \frac{\partial F}{\partial t} + \nabla \cdot (\mathbf{u}F) = 0 \quad (5)$$

135 The case study here presented was addressed under a one-fluid approach for the resolution of the flow

136 equations, as recommended by FLOW-3D® for problems involving a free surface between water and  
 137 air. This approach implies that the boundary conditions were applied to the free surface, so that the  
 138 equations could be solved only for the water phase. In contrast, the air phase was assumed to have  
 139 negligible inertia and only applied normal pressure to the free surface, allowing a significant reduction  
 140 on computing times (Bombardelli et al., 2011). Regarding the refinement of the free surface, a  
 141 mechanism that creates small negative divergences in internal fluid cells was used to close up partial  
 142 voids and add interface sharpening.

### 143 **Air entrainment modelling**

144 The air entrainment process was modelled through a balance between stabilising (gravity and surface  
 145 tension) and destabilising (turbulent kinetic energy) forces. When destabilising forces overcome  
 146 stabilising ones, air enters the flow at a rate that can be modelled as:

$$147 \quad \delta V = \begin{cases} k_{air} A_S \left[ \frac{2(P_t - P_d)}{\rho} \right]^{1/2} & \text{if } P_t > P_d \\ 0 & \text{if } P_t < P_d \end{cases} \quad (6)$$

$$148 \quad P_t = \rho k; P_d = \rho g L_T + \frac{\sigma}{L_T} \quad (7)$$

$$149 \quad L_T = \frac{C_\mu^{3/4} k^{3/2}}{\varepsilon} \quad (8)$$

150 where  $\delta V$  is the volume of entrained air rate and  $P_t$  and  $P_d$  are respectively the destabilising and  
 151 stabilising forces. In addition,  $k_{air}$  is a coefficient calibrated for each particular case and  $A_S$  is the free  
 152 surface area for each cell. The calculation of the stabilising forces involves the gravity component  
 153 normal to the free surface ( $g$ ), the surface tension coefficient ( $\sigma$ ) and the turbulent length scale ( $L_T$ ). The  
 154 latter is calculated with the parameter  $C_\mu$ . For the RNG  $k$ - $\varepsilon$  turbulence model the value of this  
 155 parameter is 0.085 (Bayon et al., 2016). The varying density in the flow, resulting from the entrained  
 156 air, is accounted for in terms of a fluid mixture density ( $\rho_m$ ) defined as:

$$157 \quad \rho_m = F \rho_w + (1 - F) \rho_a \quad (9)$$

158 where  $\rho_w$  and  $\rho_a$  are respectively the water and air densities. Besides, the drag force produced by

159 bubbles upon the carrier phase is also considered. The modelling process to obtain the drag per unit  
160 volume and the relative velocity between phases using FLOW-3D® is developed in Brethour and Hirt  
161 (2009).

## 162 **Meshing and boundary conditions**

163 The spatial domain of the case study was meshed with a three-dimensional structured mesh formed by  
164 regular hexahedral cells. Structured meshes are usually associated with the existence of regular  
165 connectivity, generally providing a good level of accuracy (Biswas & Strawn, 1998; Hirsch, 2007).  
166 Moreover, structured meshes lead to low latency during simulations (Keyes et al., 2000) and to a  
167 reduced numerical diffusivity for free surface modelling (Bayon & Lopez-Jimenez, 2015).

168 Two different blocks were employed in the meshing process to save computing time. Firstly, a  
169 refined block was used to mesh the stilling basin, covering the main area of interest, where higher flow  
170 gradients are to be expected. Secondly, a coarser mesh block downstream the stilling basin was linked  
171 to the refined one. The coarser one was used to mesh the remaining spatial domain with cells doubling  
172 the size of those forming the refined block.

173 Regarding the boundary conditions set for the spatial domain, a supercritical flow was set  
174 upstream the stilling basin by imposing the corresponding discharge and fluid elevation. Downstream  
175 the basin, an outflow condition was used, allowing the flow to leave the domain. Additionally, an  
176 atmospheric pressure condition was imposed in the upper boundary of the domain. For the solid  
177 contours a wall non-slip boundary condition was set, assuming a law-of-the-wall velocity profile in the  
178 vicinities of these boundaries.

## 179 *Mesh convergence analysis*

180 The cell sizes used in the meshing process were chosen after a mesh convergence analysis, in order to  
181 ensure a low degree of uncertainty in the results. The analysis was carried out following the American  
182 Society of Mechanical Engineers (ASME) procedure (Celik et al., 2008). Accordingly, three different  
183 meshes were tested and for each of these meshes 11 basic variables (i.e. averaged velocities along the  
184 hydraulic jump longitudinal axis) were used as indicators. Table 1 shows the cell sizes for each of the  
185 meshes assessed. These sizes were chosen considering the minimum recommended refinement ratio of  
186 1.3 established by Celik et al. (2008). Once this is done, the apparent order was calculated, following



187 the ASME procedure, as an indicator to assess grid convergence.

188 The results of the convergence analysis for the finest of the meshes tested consisted in a mesh  
189 apparent order of 2.09, close to the numerical model's formal order of 2, which means a good  
190 indication of the grids being in the asymptotic range (Celik et al., 2008). In addition, the resulting grid  
191 convergence index was 14.46%, in line with previous research involving numerical models of complex  
192 hydraulic phenomena like the one here presented (Bayon et al., 2016; Valero et al., 2018). Finally,  
193 Celik et al. (2008) recommends reporting the percentage occurrence of oscillatory convergence. For  
194 this particular case the percentage is 9%. On the basis of this analysis, the finest mesh was chosen for  
195 the study, meaning that the smallest chute block dimension was covered by at least four cells.

## 196 **MODEL VALIDATION**

197 The numerical model was developed reproducing the exact conditions of the USBR Type-II stilling  
198 basin physical model available in the Hydraulics Laboratory of the Department of Hydraulic  
199 Engineering and Environment at the Universitat Politècnica de València (UPV, Spain). According to  
200 Valero et al. (2019), it is highly recommendable to assume the same geometry of the experimental  
201 device in the numerical model for calibration and validation purposes. The design of the physical  
202 model followed the guidelines of the USBR for typified stilling basins (Peterka, 1978). In addition, the  
203 recommendations posed by Heller (2011) to avoid significant scale effects in the experimental device  
204 were considered. The flow conditions chosen in the design of the case study (Table 2) led to an inflow  
205 Froude number ( $F_1$ ) of 9, which provides an adequate energy dissipation for the analysed basin  
206 (Peterka, 1978). Figure 1 shows the experimental device in which the campaign was conducted. The  
207 details of this physical model can be found in Macián-Pérez, Vallés-Morán, et al. (2020).

208 The mean free surface longitudinal profile and the maximum forward averaged velocity decay  
209 were measured in the physical model. A time-of-flight camera using light detection and ranging  
210 (LIDAR) techniques was employed to determine the free surface, whereas velocity measurements were  
211 taken with a Pitot tube. The measured features were used to validate the numerical model. To do so, a  
212 comparison of the experimental data with the simulated results was carried out, using dimensionless  
213 metrics (Fig. 2):

$$214 \quad X = \frac{x - x_0}{L_r} \quad (10)$$

215 
$$Y = \frac{y - y_1}{y_2 - y_1} \quad (11)$$

216 
$$U_{\max} = \frac{u_{\max} - u_2}{u_1 - u_2} \quad (12)$$

217 where  $x_0$  is the jump toe position, which corresponds with the beginning of the basin for both models.  
218  $L_r$  is the hydraulic jump roller length estimation (see section 5.2). Besides,  $y_1$  and  $y_2$  are respectively the  
219 supercritical and subcritical flow depths upstream and downstream of the hydraulic jump. In regards  
220 with Eq. (12),  $u_{\max}$  is the maximum forward averaged velocity for each vertical profile measured in the  
221 roller and  $u_1$  and  $u_2$  are respectively the supercritical and subcritical mean flow velocities.

222 Figure 2 shows a good level of agreement between the numerical and the experimental model,  
223 both for the free surface profile and for the maximum velocity decay. In particular, the accuracy of the  
224 numerical model can be assessed using the coefficient of determination ( $R^2$ ) (Bennett et al., 2013). The  
225 values of this coefficient achieved by the model were 0.992 for the free surface profile and 0.964 for  
226 the maximum forward velocity decay, indicating a successful validation of the numerical model.

## 227 **STILLING BASIN MODIFIED DESIGN**

228 A modification of the original USBR Type-II stilling basin design was tested with the validated  
229 numerical model. The main purpose of this structure is to enhance energy dissipation, generally  
230 involving shorter and more stable hydraulic jumps (Hager, 1992). Any improvement in the design  
231 should therefore yield to a reduction in the dimensions of the hydraulic jump without losing energy  
232 dissipation benefits. The improved performance of a typified basin could not only reduce the economic  
233 cost of new structures, but more importantly, contribute to the adaptation of existing structures to more  
234 demanding operational conditions.

235 Previous attempts to optimise the performance of stilling basins were made focusing on both  
236 the design and the flow conditions. On the one hand, Valero et al. (2015) tested different sizes for the  
237 USBR Type-II stilling basin chute blocks. These devices lift a portion of the inflow water jet, leading  
238 to an increased number of dissipating eddies, which in turn results in shorter jump lengths. In addition,  
239 the chute blocks help to stabilise the hydraulic jump in the basin under adverse tailwater conditions  
240 (Peterka, 1978; Valero et al., 2015). These authors found that the original design dimensions of the  
241 chute blocks led to the optimal performance of the basin, since they achieved better submergences and

242 higher dissipation efficiency. Soori et al. (2017) also tested different sizes for the chute blocks in a  
243 USBR Type-II stilling basin, changing the end sill for a series of steps too. The results presented by  
244 these authors also point out to an optimal dimension of the chute blocks in line with the original design.

245 On the other hand, Montano and Felder (2019) conducted an experimental research to  
246 optimise the performance of stilling basins without energy dissipation devices by changing the  
247 hydraulic jump type in regards with the tailwater conditions. These authors found similar energy  
248 dissipation and higher stabilities for hydraulic jumps partially developed in the slope upstream the  
249 basin. Thus, an improved performance compared to the case of traditional stilling basins, for which the  
250 hydraulic jump strictly takes place downstream the slope, was found. Nevertheless, the experimental  
251 campaign only covered inlet slopes up to  $5^\circ$  and values of  $F_1$  up to 4.6. Such slopes are quite reduced  
252 for prototype dam cases. Besides, the tested  $F_1$  values only covered the lowest part of the range for  
253 adequate energy dissipation in stilling basins, as established by the USBR (Peterka, 1978). Montano  
254 and Felder (2019) found that some of the benefits of changing the tailwater conditions decreased with  
255 increasing slopes, suggesting the need for further research involving steeper slopes and higher  $F_1$ .

256 In line with these previous considerations, the present research proposes a modification of the  
257 USBR Type-II stilling basin original design. It consists in the addition of a second row of chute blocks  
258 right upstream the original row (Fig. 3). These additional chute blocks are located in chessboard order  
259 so that immediately downstream of one of the new blocks, there exists a gap between two of the  
260 original blocks. The modification aims at reducing the hydraulic jump dimensions, without affecting  
261 the energy dissipation performance of the structure.

262 The modified stilling basin was simulated using FLOW-3D<sup>®</sup>, with the numerical setup  
263 developed for the validated model. Thus, similar flow conditions ( $F_1 = 9$ ,  $q = 0.147 \text{ m}^2 \text{ s}$ ), meshing and  
264 modelling parameters were employed. As stated by Schulz et al. (2015), numerical models can be used  
265 to test geometrical modifications in hydraulic structures, if the resulting design is sufficiently close to  
266 the experimental background. Figure 4 shows an example of the results obtained from the simulations  
267 of both basin designs.

## 268 **RESULTS AND DISCUSSION**

269 An appropriate design for energy dissipation stilling basins must consider the dimensions of the  
270 hydraulic jump developed within the structure. According to Schulz et al. (2015), the most relevant

271 geometrical characteristics of the hydraulic jump for stilling basin design purposes are the sequent  
272 depths ratio and the length of both, the roller region and the hydraulic jump. Thus, all these features  
273 were analysed, together with the hydraulic jump efficiency, to assess the energy dissipation  
274 performance of the basin.

### 275 **Free surface profile**

276 The study of the hydraulic jump longitudinal free surface profile provides an insight into the  
277 dimensions of the phenomenon, which can be used to assess the effect of the stilling basin design.  
278 Firstly, the sequent depths ratio ( $y_2/y_1$ ) was analysed. To this end, the values obtained in the numerical  
279 simulations were compared with the theoretical expression proposed by Hager and Bremen (1989) for  
280 CHJ:

$$281 \quad \frac{y_2}{y_1} = \frac{1}{2} \left[ (1 + 8F_1^2)^{1/2} - 1 \right] \cdot \left\{ 1 - 0.7 \left[ (\log R_e)^{-5/2} \right]^{F_1/8} \right\} \cdot \left[ 1 - \frac{3.25 y_1}{b^{F_1/7}} (\log R_e)^{-3} \right] \quad (13)$$

282 where  $b$  is the width of the hydraulic jump or the stilling basin and  $R_e$  is the Reynolds number. The  
283 analysis also includes bibliographic values of the sequent depths ratio for hydraulic jumps with similar  
284  $F_1$  occurring in different types of stilling (Table 3).

285 Table 3 shows a significant variability of the results depending on the source. These  
286 differences suggest that the sequent depths ratio is not only influenced by the inflow Froude number but  
287 also by the other factors included in Eq. (13). In these terms, in spite of the similar  $F_1$  values, the reason  
288 behind the differences observed could be the particular inflow depth, inflow Reynolds number or unit  
289 discharge of the sources under analysis. Focusing on the stilling basin cases modelled in this research,  
290 which share the same value for the previously mentioned factors ( $F_1$ ,  $R_e$ ,  $y_1$ ,  $b$ ), the hydraulic jump in  
291 the modified design shows a lower sequent depths ratio than the one in the original USBR Type-II  
292 basin. In particular, the sequent depths ratio is 2.5% lower in this modified design. This suggests a  
293 reduction in the dimensions of the hydraulic jump with the addition of the second row of chute blocks,  
294 leading to an optimisation in the structure design.

295 The mean free surface profile along the longitudinal axis of the hydraulic jump was also  
296 measured in the numerical models (Fig. 5). Figure 5a shows that the hydraulic jump free surface profile  
297 is quite similar in the original stilling basin and in the modified design. However, some relevant  
298 differences should be pointed out. The additional row of chute blocks, placed in the modified design

299 immediately upstream of the original row, leads to higher flow depths at the beginning of the basin.  
300 These higher flow depths disappear for downstream positions. Thus, for  $x$  values greater than 0.5 m  
301 until the end of the basin, the hydraulic jump profile in the modified structure is placed slightly below  
302 the one developed in the original design (around 2.5% lower flow depth values in the modified design).

303         Regarding the dimensionless values shown in Fig. 5b, all of the represented profiles follow  
304 similar trends. Nevertheless, a particular difference between the CHJ and the stilling basin should be  
305 pointed out. For the basin profiles, the subcritical flow depth is reached for  $X$  values around 1, whereas  
306 the CHJ profiles keep increasing for a longer distance, so that the subcritical depth is reached  
307 downstream (around  $X = 1.5$ ). This result indicates a shortened hydraulic jump both for the original and  
308 the modified basins, in comparison with the CHJ.

### 309 **Hydraulic jump roller length**

310 The hydraulic jump roller length ( $L_r$ ) is of paramount importance for the design of stilling basins since  
311 it constitutes a geometrical feature strictly linked to the structure dimensions. The roller region  
312 determines the boundary between backward and forward flow, starting at the toe of the jump and  
313 ending at the surface stagnation point (Hager & Bremen, 1989). Besides, the most intense energy  
314 dissipation within the hydraulic jump is enclosed in this region. The estimation of this feature was  
315 conducted following the stagnation point criterion for both numerical models (Fig. 6). Accordingly, a  
316 series of streamwise velocity vertical profiles were characterised along the hydraulic jump longitudinal  
317 axis. For each of these profiles, the stagnation point (i.e. point where velocity tends to zero) was  
318 identified. Finally, the intersection between the line joining all the stagnation points and the mean free  
319 surface profile marks the end section for the roller region (Hager et al., 1990). The extrapolation done  
320 to meet the free surface consisted in an exponential adjustment with  $R^2$  values above 0.95.

321         Following this procedure, the estimated dimensionless roller lengths ( $L_r/y_1$ ) were respectively  
322 48.00 and 47.33 for the original and for the modified basin numerical models, which means a 1.4%  
323 reduction for this dimensionless value. For comparison purposes, the expression proposed by Hager et  
324 al. (1990) to determine the hydraulic jump roller length was employed:

$$\begin{aligned} L_r &= y_1 \left[ -12 + 160 \tanh(F_1/20) \right] \text{ for } y_1/b < 0.10 \\ L_r &= y_1 \left[ -12 + 100 \tanh(F_1/12.5) \right] \text{ for } 0.10 < y_1/b < 0.70 \end{aligned} \quad (14)$$

326 This expression, originally thought for CHJ, provides a dimensionless length of 55.67.  
327 Therefore, the basin objective of shortening the space in which energy dissipation occurs (Hager, 1992)  
328 is successfully accomplished. In terms of the basin design, the roller region for the hydraulic jump  
329 developed in the modified basin was slightly shorter, which could lead to a reduction in the dimensions  
330 of the structure. Figure 6b includes the roller boundary for the original USBR II model, showing this  
331 reduction in the roller length. The comparison also shows that the roller region in the modified design  
332 is lifted up from its original position.

### 333 **Hydraulic jump length**

334 From an engineering perspective, the hydraulic jump length ( $L_j$ ) can be identified as the distance in  
335 which bottom protection against erosion is needed for the design of stilling basins (Hager, 1992).  
336 However, there is no a clear or unique theoretical definition for this dimension. According to Valero et  
337 al. (2019), the estimation of the hydraulic jump length usually implies an important degree of  
338 uncertainty. In fact, traditional approaches are based on the visual determination of this feature. For  
339 instance, the hydraulic jump end section has been previously identified with the section where the  
340 hydraulic jump is fully deaerated or where the free surface is essentially horizontal (Hager et al., 1990;  
341 Kramer & Valero, 2020).

342 This study aims at shedding light on the determination of the hydraulic jump length and thus,  
343 different methods were tested. Overall, the objective was to achieve a reliable estimation of this  
344 parameter to assess the influence of the stilling basin design. Some recent studies shared this objective  
345 and developed physical criteria, less based on subjective interpretation (Stojnic et al., 2021). For this  
346 particular research, two different procedures were assessed. On the one hand, the streamwise averaged  
347 velocity vertical profiles were analysed. In these terms, Hager (1992) referred to the hydraulic jump  
348 end section as the section where gradually varied flow conditions reappear, whereas Bayon et al.  
349 (2016) pointed to the study of the velocity profile to identify the hydraulic jump end section. On this  
350 basis, streamwise averaged velocity vertical profiles along the basin longitudinal axis were obtained for  
351 both numerical models. These profiles were compared with the expression proposed by Kirkgöz and  
352 Ardiçlioğlu (1997) for open channel flow:

$$353 \quad \frac{u_{\max} - u}{u^*} = -2.44 \ln \left( \frac{z}{y_2} \right) + 0.488 \left[ \cos \left( \frac{\pi z}{2y_2} \right) \right]^2 \quad (15)$$

354 where  $z$  is the vertical position in the profile and  $u^*$  is the shear velocity, that can be obtained as:

$$355 \quad u^* = \left( \frac{\tau_0}{\rho} \right)^{1/2} \quad (16)$$

$$356 \quad \tau_0 = \gamma R_H I \quad (17)$$

357 where  $\tau_0$  is the wall shear stress,  $\gamma$  is the specific weight and  $R_H$  the hydraulic radius. In addition,  $I$  is the  
358 energy line slope which can be estimated from Manning equation. The comparison between the  
359 modelled profiles and Eq. (15) was then used to assess where the open channel flow conditions were  
360 reached.

361 Figure 7 shows that there is an evolution of the velocity profiles downstream of the stilling  
362 basin ( $x = 1.76$  m). Thus, the shape of modelled profiles tends to the open channel flow profile as the  
363 distance from the hydraulic jump toe increases. In both numerical models, the profile for  $x = 2.4$  m  
364 shows a good agreement ( $R^2 \geq 0.9$ ) with Eq. (15).

365 On the other hand, the turbulent kinetic energy ( $k$ ) decay was also employed to figure out the  
366 hydraulic jump length, as proposed by Bayon et al. (2016). The turbulent kinetic energy can be defined  
367 as half the sum of the variances of the spatial velocity components:

$$368 \quad k = \frac{1}{2} \left[ \overline{(u_x')^2} + \overline{(u_y')^2} + \overline{(u_z')^2} \right] \quad (18)$$

369 The aforementioned authors established a 95% decay of the maximum turbulent kinetic energy  
370 as an approximate threshold to determine the hydraulic jump end section in a numerical model. Hence,  
371 the values of  $k$  along the hydraulic jump longitudinal axis were obtained in both numerical models and  
372 compared in Fig. 8.

373 The criterion developed by Bayon et al. (2016) provides a hydraulic jump length of 3.2 m for  
374 both numerical models. Nevertheless, if a 90%  $k$  decay threshold is taken as reference, the hydraulic  
375 jump length would be 2.4 m, in perfect agreement with the previously presented method. Considering  
376 that this procedure was developed by Bayon et al. (2016) for CHJ, the proposed threshold could be  
377 varied by the stilling basin design. The chute blocks immediately upstream the jump toe provide  
378 additional energy dissipation, so that the maximum  $k$  is lower than the one obtained in a CHJ with the  
379 same conditions. Hence, a lower decay of the maximum  $k$  would be needed to achieve the subcritical

380 flow conditions. It is also important to remark that the turbulent kinetic energy for the modified basin  
381 model is constantly below that of the original model (Fig. 8), especially in the vicinity of the jump toe.

382 Peterka (1978) established an experimental relationship between the hydraulic jump length  
383 and the  $F_1$  value for a variety of typified energy dissipation structures. Furthermore, Hager (1992)  
384 proposed the following expression to estimate the jump length in CHJ with  $F_1$  values between 4 and  
385 12:

$$386 \quad L_j = y_1 220 \tanh \left[ \frac{(F_1 - 1)}{22} \right] \quad (19)$$

387 Movahed et al. (2018) argued that the accuracy in the estimation of the hydraulic jump length  
388 can be improved by considering the Froude number downstream of the hydraulic jump ( $F_2$ ) and  
389 provided a semi-analytical equation to obtain  $L_j$ .

$$390 \quad L_j = y_1 \left( 3.7 + 3/F_2 \right) \quad (20)$$

391 The different hydraulic jump length dimensionless values obtained are displayed in Table 4  
392 for convenient comparison. In summary, it can be stated that the methods presented in this research for  
393 the estimation of the hydraulic jump length provide different results, being the one based on the  
394 velocity profiles closer to the bibliographic data. However, if the previously mentioned variation in the  
395  $k$  decay threshold is assumed, both methods show a perfect agreement. The agreement between these  
396 two methods contributes to the consistency in the estimation of a parameter such as the  $L_j$ , usually  
397 surrounded by a high degree of uncertainty.

398 Past studies generally provided lower  $L_j$  values than the presented models. These differences  
399 could be explained by several factors. On the one hand, different methods were employed to obtain the  
400 hydraulic jump length, which undoubtedly affects the results. In these terms, the data collected by  
401 Peterka (1978) clearly underestimates this parameter in the USBR Type-II stilling basin, as previously  
402 observed by Movahed et al. (2018). On the other hand, Habibzadeh et al. (2019) found an increase in  $L_j$   
403 in the presence of energy dissipation blocks for hydraulic jumps with  $F_1$  greater than 5. These authors  
404 based their estimation on the water surface fluctuations and found that large-scale turbulence structures  
405 created by the blocks in the form of surface fluctuations persisted for longer distances.



## 406 **Hydraulic jump efficiency**

407 Any modification made in the design of a stilling basin to reduce the hydraulic jump dimensions, must  
408 also account for the energy dissipation purpose of the structure. Accordingly, improving or maintaining  
409 appropriate hydraulic jump efficiencies remains an indispensable condition to optimise the  
410 performance of a stilling basin. The hydraulic jump efficiency ( $\eta$ ), based upon differences in the  
411 specific head upstream and downstream the hydraulic jump, gives a measure of the dissipated energy:

$$412 \quad \eta = \frac{H_{01} - H_{02}}{H_{01}} \quad (21)$$

413 where  $H_{01}$  and  $H_{02}$  are respectively the specific energy heads upstream and downstream of the hydraulic  
414 jump. Padulano et al. (2017) and Macián-Pérez, Vallés-Morán, et al. (2020) obtained the hydraulic  
415 jump efficiency in a USBR Type-II basin for  $F_1$  values around 9. Besides, using literature expressions  
416 (Eqs. 13 and 21), the efficiency in a CHJ with the studied inflow conditions can be estimated. Table 5  
417 shows the modelled and bibliographic  $\eta$  values for comparison purposes.

418 As it should be expected, these values are in line with the results shown in Table 3, due to the  
419 strong correlation between the sequent depths ratio and the hydraulic jump efficiency. Thus, the  
420 numerical models here presented provided a higher sequent depths ratio than the bibliographic results,  
421 leading to a lower efficiency. Focusing on the comparison between the original and the modified  
422 USBR II design, there is a slight increase (around 1.2%) in the efficiency caused by the additional  
423 chute blocks row, which also represents an improved performance of the stilling basin.

## 424 **CONCLUSIONS**

425 This research presents a detailed numerical model of a USBR Type-II stilling basin. The model was  
426 developed on the basis of CFD techniques and validated with a specifically-designed physical model. A  
427 modified design resulting from the addition of a second row of chute blocks was also implemented and  
428 tested. The comparison between both designs was carried out in terms of the hydraulic jump  
429 dimensions and the energy dissipation, in order to assess the performance of the basin. The results  
430 obtained were generally quite similar and in good agreement with literature studies. However, the  
431 modified design showed a reduction in the dimensions of the hydraulic jump. This conclusion was  
432 obtained after evaluation of the resulting sequent depths ratios, roller lengths, and hydraulic jump

433 efficiencies. In particular, the sequent depths ratio and the dimensionless roller length were respectively  
434 2.5% and 1.4% lower in the modified design, whereas the energy dissipation efficiency increased a  
435 1.2%. Consequently, the additional chute blocks row seems to enhance the performance of the basin.  
436 This modification constitutes an interesting novelty as it tends to reduce the expected dimension of the  
437 hydraulic jump, and thus, might be of interest in order to reduce the cost of the structure. It is difficult  
438 to quantify the potential discharge increase allowed by the reduction in the hydraulic jump dimensions  
439 provided by the modified design. However, a first estimation can be done. Using the graphs provided  
440 by the USBR for the Type-II stilling basin (Peterka, 1978) that establish a relationship between the  
441 hydraulic jump dimensions and the inflow conditions, the decrease observed in the sequent depths ratio  
442 and the dimensionless roller length is associated with a potential discharge increase around the 10%.  
443 Nevertheless, it is important to highlight that these results were obtained in the application range of the  
444 simulations and must be confirmed with further research, testing more demanding discharges. The  
445 inherent uncertainties around the definition and evaluation of the hydraulic jump length were also  
446 investigated. Two methods were tested and compared. The presented methods follow a structured and  
447 systematic procedure, based on quantitative information, and show consistent results for the developed  
448 models. Therefore, they might be useful for future studies in which the jump length needs to be  
449 determined.

450         The results here presented constitute a first step towards an optimised design of the USBR II  
451 stilling basin. In these terms, the proposed solution establishes a simple and straight forward  
452 modification starting from the traditional USBR Type-II design that can be used by engineers to reduce  
453 the dimensions of the hydraulic jump and still preserve the energy dissipation in the corresponding  
454 basin. These results must be confirmed by future research on the topic, testing different and more  
455 demanding inflow conditions ( $F_1$ ,  $Re$ ,  $y_1$ ,  $q$ ) and alternative modifications in the basin design.  
456 Furthermore, some other crucial features for the performance assessment of the basin such as the  
457 dynamic bottom pressures and the void fraction distribution need to be analysed. Overall, the numerical  
458 model developed was used to simulate a modified design of the USBR II stilling basin providing an  
459 optimised energy dissipation performance with reductions in the hydraulic jump dimensions. This  
460 information could be useful not only for the design of new energy dissipation structures, but also for  
461 the adaptation of existing basins to more demanding conditions posed by climate change and flood  
462 protection requirements.

463 **DATA AVAILABILITY STATEMENT**

464 All data, models and code generated or used during the study appear in the submitted article.

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473 **NOTATION**

474 The following symbols are used in this paper:

475  $A_S$  = free surface area for each cell ( $m^2$ )

476  $b$  = hydraulic jump width (m)

477  $C_\mu$  = parameter for the turbulent length scale (-)

478  $C_{1\epsilon}$ ,  $C_{2\epsilon}$  = turbulence model parameters (-)

479  $f_b$  = body forces (N)

480  $F$  = fraction of fluid (-)

481  $F$  = Froude number (-)

482  $g$  = gravity acceleration ( $m\ s^{-2}$ )

483  $H_0$  = specific head (m)

484  $I$  = linear hydraulic head loss (-)

485  $k$  = turbulent kinetic energy ( $J\ kg^{-1}$ )

486  $k_{air}$  = air entrainment coefficient (-)

487  $L_j$  = hydraulic jump length (m)

488  $L_r$  = hydraulic jump roller length (m)

489  $L_T$  = turbulent length (m)

490  $P_d$  = stabilising forces ( $N\ m^{-2}$ )

491  $P_k$  = production of turbulent kinetic energy ( $kg\ m^{-1}\ s^{-3}$ )

492  $P_t$  = destabilising forces ( $N\ m^{-2}$ )

493  $p$  = pressure (Pa)

494  $q$  = unit discharge ( $m^2\ s$ )

495  $Re$  = Reynolds number (-)

496  $R_H$  = hydraulic radius (m)

497  $t$  = time (s)  
498  $u$  = velocity (m s<sup>-1</sup>)  
499  $u^*$  = shear velocity (m s<sup>-1</sup>)  
500  $x$  = distance along the basin longitudinal axis (m)  
501  $x_0$  = hydraulic jump toe position (m)  
502  $y$  = flow depth (m)  
503  $\gamma$  = specific weight (N m<sup>3</sup>)  
504  $\delta V$  = volume of entrained air rate (m<sup>3</sup> s)  
505  $\varepsilon$  = turbulent kinetic energy dissipation rate (J kg<sup>-1</sup> s<sup>-1</sup>)  
506  $\eta$  = hydraulic jump efficiency (-)  
507  $\mu$  = dynamic viscosity (Pa s)  
508  $\nu$  = kinematic viscosity (m<sup>2</sup> s<sup>-1</sup>)  
509  $\rho$  = density (kg m<sup>-3</sup>)  
510  $\sigma$  = surface tension coefficient (N m<sup>-1</sup>)  
511  $\sigma_k, \sigma_\varepsilon$  = turbulence model parameters (-)  
512  $\tau_0$  = wall shear stress (Pa)

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## 643 **LIST OF FIGURES**

644 **Fig. 1.** Experimental set-up in the Hydraulics Laboratory of the Department of Hydraulic Engineering  
645 and Environment at the Universitat Politècnica de València (UPV, Spain)

646 **Fig. 2.** Validation of the numerical model: (a) mean free surface longitudinal profile, (b) maximum  
647 forward velocity decay

648 **Fig. 3.** Modelled stilling basins: (a) typified USBR II stilling basin, (b) modified design with an  
649 additional chute blocks row

650 **Fig. 4.** Velocity field obtained in the numerical model: (a) typified USBR II stilling basin, (b) modified  
651 design with an additional chute blocks row

652 **Fig. 5.** Hydraulic jump free surface profile: (a) mean longitudinal profile in the typified USBR II  
653 stilling basin and in the modified design, (b) dimensionless comparison with bibliographic data for CHJ

654 **Fig. 6.** Roller length estimation following the stagnation point criterion: (a) typified USBR II stilling  
655 basin, (b) modified design

656 **Fig. 7.** Subcritical velocity profiles: (a) bibliographic profile in open channel flow, (b) modelled  
657 profiles downstream the typified USBR II basin, (c) modelled profiles downstream the modified USBR  
658 II basin

659 **Fig. 8.** Turbulent kinetic energy decay along the hydraulic jump longitudinal axis

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661



662 **Table 1.** Cell sizes tested in the mesh convergence analysis

Mesh	Refined mesh block	Coarse mesh block
1	0.014 m	0.028 m
2	0.010 m	0.020 m
3	0.007 m	0.014 m

663

664 **Table 2.** Case study flow conditions

$Q$ (m <sup>3</sup> /s)	$q$ (m <sup>2</sup> /s)	$y_1$ (m)	$u_1$ (m/s)	$F_1$	$Re_1$
0.113	0.147	0.03	4.89	9	146,753

665

666 **Table 3.** Sequent depths ratio for the numerical models and for bibliographic studies

Case	Hydraulic jump conditions	$y_2/y_1$
Numerical model	Typified USBR II stilling basin	14.43
Numerical model	Modified USBR II stilling basin	14.06
(Hager & Bremen, 1989)	CHJ	12.17
(Schulz et al., 2015)	CHJ	11.73
(Macián-Pérez, Vallés-Morán, et al., 2020)	Typified USBR II stilling basin	12.00
(Padulano et al., 2017)	Typified USBR II stilling basin	10.18

667

668 **Table 4.** Hydraulic jump length for the numerical models and for bibliographic studies

Case	Hydraulic jump conditions	Methodology	$L_j/y_1$
Numerical model	Typified USBR II	Velocity profiles	80.0
	stilling basin	$k$ decay	106.7
Numerical model	Modified USBR II	Velocity profiles	80.0
	stilling basin	$k$ decay	106.7
(Hager, 2013)	CHJ	Empirical expression	76.7
(Movahed et al., 2018)	CHJ	Semi-analytical expression	71.3
(Peterka, 1978)	Typified USBR II stilling basin	Collected data	58.7

669

670 **Table 5.** Hydraulic jump efficiency for the numerical models and for bibliographic studies

Case	Hydraulic jump conditions	$\eta$
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Numerical model	Typified USBR II stilling basin	0.648
Numerical model	Modified USBR II stilling basin	0.656
(Hager & Bremen, 1989)	CHJ	0.701
(Macián-Pérez, Vallés-Morán, et al., 2020)	Typified USBR II stilling basin	0.705
(Padulano et al., 2017)	Typified USBR II stilling basin	0.720

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