


Article

Assessment of in-Plane Behavior of Metal Compressed Members with Equivalent Geometrical Imperfection

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Abstract: A new procedure was presented with the objective of proving that it is the generalization of current attempts in designing compressed members and structures which is able to solve cases where other authors have problems. It is the further development of the former methods published by Chladný, Baláž, Agüero et al., which are based on the shape of the elastic critical buckling mode of the structure. Chladný's method was accepted by CEN/TC 250 working groups creating Eurocodes. Both current Eurocodes EN 1993-1-1:2005 and EN 1999-1-1:2007 in their clauses 5.3.2(11) enable applying the geometrical equivalent unique global and local initial (UGLI) imperfection. The imperfection has the shape of the elastic critical buckling mode with amplitude defined in 5.3.2(11). UGLI imperfection is an alternative to the global sway and local bow initial imperfections defined in 5.3.2(3) and to the imperfections described in the clause 5.3.2(6). The determination of the location and value of UGLI imperfection proved to be onerous by some authors, especially in cases of members with variable cross-sections or/and axial forces. The paper also provides for special cases a procedure to detect the critical cross-section along the member which is defined as the one in which the utilization factor obtains maximum values. The new approach is validated by the investigation of five complex structures made of steel and one made of aluminum alloy solved by other authors. Comparisons of the results with those of other authors and with the Geometrically and Materially Nonlinear Analysis with Imperfections (GMNIA) results showed very good agreements with negligible differences. The information concerning the differences between current Eurocodes EN 1993-1-1:2005 and EN 1999-1-1:2007 is provided. Working drafts of Eurocodes of new generation prEN 1993-1-1:2020 and prEN 1999-1-1:2020 are also commented on.

Keywords: buckling in plane of the structure; UGLI imperfection based on buckling modes; steel; aluminum alloy; EN 1993-1-1; EN 1999-1-1

Highlights

- A new procedure is presented to obtain the location of the critical section and the value of the amplitude of unique global and local initial (UGLI) imperfection having the shape of elastic critical buckling mode. The new procedure is also used for the structures consisting of strongly irregular members in compression.
- Validation of the procedure by the investigation of five complex structures made of steel and one made of aluminum alloy solved by other authors.

- Verification of the procedure by comparisons with results of other authors and Geometrically and Materially Nonlinear Analysis with Imperfections (GMNIA) results.
- Information about differences in methods given in four Eurocodes: EN 1993-1-1:2005, EN 1999-1-1:2007, prEN 1993-1-1:2020 and prEN 1999-1-1:2020.

1. Introduction

1.1. Overview and Analysis of Current State

According to Eurocode EN 1993-1-1 [1] and EN 1999-1-1 [2], second order effects and initial imperfection depending on the type of frame structure may be taken into account in global analysis. The design of steel structures with compression elements must consider the existence of residual stresses and geometric imperfections such as initial out-of-plumbness, flatness and fit tolerances. The negative effects of all these imperfections are covered in the global analysis by equivalent geometrical imperfection with a relevant shape and value of amplitude. There are the following types of imperfections in [1]: (i) global initial sway imperfection for the global analysis of frames; (ii) local initial bow imperfection for global and member analysis; (iii) imperfection for the analysis of bracing systems; and (iv) unique global and local initial (UGLI) imperfection based on elastic critical buckling modes. In this paper, the global analysis with UGLI imperfection is utilized.

Section 5.3.2(11) of [1] provides formulae which are the base of a procedure for obtaining UGLI imperfection. Unfortunately, the procedure in [1] is limited to the structures with a uniform cross-section under uniform axial force distribution. No iterations are needed in such cases. The critical section will take place where the curvature has maximum value. The detailed description of the development of this method is given by Chladný in [3]. The procedure described in [3] is without the above limitations and may be used for the most practical cases. Generally, the critical section must be found by iterations. The more general procedure without above limitations is given in EN 1999-1-1 [2]. The new generation of Eurocodes is in preparation and it should be available to the technical public in 2023. In the working drafts prEN 1993-1-1 [4] and prEN 1999-1-1 [5] the more general procedure was improved. The formulation in draft [5] is more useful and clear compared to its former edition [2]. The working draft prEN 1993-1-1 [4] differs from its former edition [1] and from [2] and [5]. In Eurocode [4], the partial material safety factor γ_{M1} was removed from the formula for the imperfection amplitude e_0 of the equivalent member. The consequence is that the value of imperfection amplitude e_0 is in [1,2,5] greater than in [4].

The working draft Eurocode prEN 1999-1-1:2020 [5] contains the following formulae for:

- (i). The characteristic value of the initial imperfection amplitude of the equivalent member related to the critical section m :

$$e_{0,k,m} = \alpha(\bar{\lambda}_m - \bar{\lambda}_0) \frac{M_{Rk,m}}{N_{Rk,m}} \quad (1)$$

- (ii). Design value of the initial imperfection amplitude of the equivalent member related to the critical section m :

$$e_{0,d,m} = e_{0,k,m} \frac{1 - \chi_m \bar{\lambda}_m^2}{1 - \chi_m \bar{\lambda}_m^2} \frac{\gamma_{M1}}{\gamma_{M1}} = \alpha(\bar{\lambda}_m - \bar{\lambda}_0) \frac{M_{Rk,m}}{N_{Rk,m}} \frac{1 - \chi_m \bar{\lambda}_m^2}{1 - \chi_m \bar{\lambda}_m^2} \frac{\gamma_{M1}}{\gamma_{M1}} \quad (2)$$

Note that the symbol $e_{0,d,m}$ used in Equation (2) must not be confused with the amplitude of the initial local bow imperfection e_0 given in [1] or [2] in Table 5.1.

(iii). Design value of UGLI imperfection amplitude $\eta_{init,max}$:

$$\eta_{init,max} = e_{0,d,m} \frac{N_{cr,m}}{EI_m |\eta''_{cr,m}|} \tag{3}$$

For the simply supported members with a uniform cross-section and uniform axial force distribution $\eta_{init,max} = e_{0,d,m}$. The symbols $e_{0,k,m}$ and $\eta_{init,max}$ are used here for the better understanding of the procedure. They are not explicitly used in Eurocodes [1,2,4,5].

The differences of Eurocodes [1,2,4] compared with the working draft Eurocode prEN 1999-1-1:2019 [5] are as follows:

- (i). Current Eurocode EN 1993-1-1:2005 [1] contains the above Equations (2) and (3) but in Corrigendum AC (2009) the index “d” was removed and some print errors were corrected.
- (ii). Current Eurocode EN 1999-1-1:2007 [2] contains the above Equations (2) and (3) but in Amendment A2 (2013) [2] there are several changes and index “d” was removed.
- (iii). Working draft Eurocode prEN 1993-1-1:2020 [4] removed from the above Equations (2) and (3) the partial material safety factor γ_{M1} and index “d”. Removing γ_{M1} from Equation (2) destroyed the main assumption of Chladný’s method defined below, but the code committee followed this idea because it leads to an ease of use in practical applications [6]. The authors intend to publish the results of the study showing the consequences of this decision leading to an important difference between the values of UGLI imperfection amplitude calculated according to [4] or [5].

Eurocodes use the symbol $\eta_{init,max}$ for the design value of UGLI imperfection amplitude. In the paper, the symbol $\eta_0 = \eta_{init,max}$ is also used, which is not in Eurocodes. The symbol η_0 is used in the text, Equations and Tables because it is shorter than $\eta_{init,max}$. The critical section is indicated in this paper by symbol x_{cr} and not by index “m” as it is done in Eurocodes. Equation (4) can be rearranged in the form of Equation (5):

$$\eta_{init,m}(x) = e_{0,d,m} \frac{N_{cr,m}}{EI_m |\eta''_{cr,m}|} \eta_{cr}(x) = \eta_{init,max} \eta_{cr}(x) \tag{4}$$

where χ is the reduction factor for the relevant buckling curve depending on the relevant cross-section, see 6.3.1.

Equation (4) is used in Eurocodes in the clauses: 5.3.2(11) [1], 5.3.2(11) [2], 7.3.6(1) [4] and 7.3.2(11) [5]. Equation (5) is just another form of (4). It is more convenient for programming. When flexural buckling occurs about the strong axis, UGLI imperfection may be written in the form:

$$\eta_{init,w}(x) = \left(\frac{\alpha(\bar{\lambda}_y - \bar{\lambda}_0)}{\bar{\lambda}_y^2} \frac{1 - \frac{\bar{\lambda}_y^2 \chi_y}{\gamma_{M1}}}{1 - \bar{\lambda}_y^2 \chi_y} \frac{f_y}{E \cdot \left(\frac{I_y}{W_y} \frac{d^2 \eta_{cr,w}}{dx^2} \right)} \right)_{x_{cr}} \eta_{cr,w}(x) = \eta_0 \eta_{cr,w}(x) \tag{5}$$

When flexural buckling occurs about the weak axis, UGLI imperfection is as follows:

$$\eta_{init,v}(x) = \left(\frac{\alpha(\bar{\lambda}_z - \bar{\lambda}_0)}{\bar{\lambda}_z^2} \frac{1 - \frac{\bar{\lambda}_z^2 \chi_z}{\gamma_{M1}}}{1 - \bar{\lambda}_z^2 \chi_z} \frac{f_y}{E \cdot \left(\frac{I_z}{W_z} \frac{d^2 \eta_{cr,v}}{dx^2} \right)} \right)_{x_{cr}} \eta_{cr,v}(x) = \eta_0 \eta_{cr,v}(x) \tag{6}$$

The method with UGLI imperfection was developed by Chladný and published with necessary details and applications in [3,7,8]. The first applications of Chladný’s method may be found in [9], where two examples were solved: a portal frame and a member with a non-uniform cross-section. In the second edition [10], the same two examples as in [8] may be found together with an example of the steel structure of three-story building. Baláž, who participated in developing Chladný’s

method (see pages 693–694 [7]), derived this method in a different way and published the details of a step-by-step procedure for non-uniform members in [11]. Baláž and Koleková published detailed illustrative numerical examples in [12,13]. They investigated in [14] the in-plane stability of the large two-hinged arch bridge (Žďárkov bridge in Czech Republic). Baláž showed that for the uniform members, UGLI imperfection amplitude has important geometrical meaning, which enables doing calculation and results verification by “hand calculation”. The importance of so called “hand calculation” was illustrated in several examples in [15]. In [3,7,8], Chladný solved in detail a member with a non-uniform cross-section and three-story building as in [8] and three new examples relating to the stability of parts of bridge structures: (i) the bottom flange of the continuous composite steel and concrete road bridge; (ii) the upper chord of the steel railway half-through truss bridge; and (iii) the basket handle type of the arch bridge. Numerical examples in [3,7,8] provide guidance of the proper application of Chladný’s method. Chladný showed there the good agreement with other methods of Eurocode EN 1993-1-1:2005 described in clauses 5.2.2(3) and 5.3.2(3). The guidance for the design of members of the steel basket handle arch bridges is given in the clause NB.3 of Slovak National Annex to Eurocode STN EN 1993-2 Design of steel structures—Part 2 Steel bridges. Chladný pointed out there that it is not enough for a such type of bridges to take into account only the first buckling mode. The members of such bridges are in biaxial bending. Chladný in [3,7,8] generalized his method for members when flexural buckling occurs about both axes. For a member in biaxial bending, UGLI imperfection may be defined as follows:

$$\begin{Bmatrix} \eta_{init,v}(x) \\ \eta_{init,w}(x) \end{Bmatrix} = \left(\frac{\alpha(\bar{\lambda} - \bar{\lambda}_0) \left(1 - \frac{\bar{\lambda}^2 \chi}{\gamma M1}\right) f_y}{\bar{\lambda}^2 \left(1 - \bar{\lambda}^2 \chi\right) E \cdot \left(\frac{I_z}{W_z} \frac{d^2 \eta_{cr,v}}{dx^2} + \frac{I_y}{W_y} \frac{d^2 \eta_{cr,w}}{dx^2}\right)} \right)_{x_{cr}} \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,w}(x) \end{Bmatrix} = \eta_0 \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,w}(x) \end{Bmatrix} \quad (7)$$

The new methodology has been developed in recent years in different works. Generalization for: (i) flexural torsional buckling of member due to compression was given by Agüero et al. in [16,17]:

$$\begin{Bmatrix} \eta_{init,v}(x) \\ \eta_{init,w}(x) \\ \eta_{init,\theta x}(x) \end{Bmatrix} = \left(\frac{\alpha(\bar{\lambda}_{TF} - \bar{\lambda}_0) \left(1 - \frac{\bar{\lambda}_{TF}^2 \chi_{TF}}{\gamma M1}\right) f_y}{\bar{\lambda}_{TF}^2 \left(1 - \bar{\lambda}_{TF}^2 \chi_{TF}\right) E \cdot \left(\frac{I_z}{W_z} \frac{d^2 \eta_{cr,v}}{dx^2} + \frac{I_y}{W_y} \frac{d^2 \eta_{cr,w}}{dx^2} + \frac{I_w}{W_B} \frac{d^2 \eta_{cr,\theta x}}{dx^2}\right)} \right)_{x_{cr}} \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,w}(x) \\ \eta_{cr,\theta x}(x) \end{Bmatrix} = \eta_0 \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,w}(x) \\ \eta_{cr,\theta x}(x) \end{Bmatrix} \quad (8)$$

and for (ii) lateral torsional buckling of member due to bending moment by Agüero et al. [16,18]:

$$\begin{Bmatrix} \eta_{init,v}(x) \\ \eta_{init,\theta x}(x) \end{Bmatrix} = \left(\frac{\alpha_{LT}(\bar{\lambda}_{LT} - \bar{\lambda}_{LT,0}) \left(1 - \frac{\bar{\lambda}_{LT}^2 \chi_{LT}}{\gamma M1}\right) f_y}{\bar{\lambda}_{LT}^2 \left(1 - \beta \bar{\lambda}_{LT}^2 \chi_{LT}\right) E \cdot \left(\frac{I_z}{W_z} \frac{d^2 \eta_{cr,v}}{dx^2} + \frac{I_w}{W_B} \frac{d^2 \eta_{cr,\theta x}}{dx^2}\right)} \right)_{x_{cr}} \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,\theta x}(x) \end{Bmatrix} = \eta_0 \begin{Bmatrix} \eta_{cr,v}(x) \\ \eta_{cr,\theta x}(x) \end{Bmatrix} \quad (9)$$

The method with UGLI imperfection was also used by two Baláž’s Ph.D. students: Kováč [19] and Dallemule [20], where other numerical examples may be found. Dallemule created a general computer program based on Chladný’s method. Bijlaard et al. [21] and Wieschollek et al. [22] have also generalized the equations given in [1] for the lateral torsional buckling of a member in bending, and Papp [23] investigated the buckling of member under bending and compression.

Brodniansky [24], another Ph.D. student from Bratislava, used a modified Dallemule’s computer program and pointed out some numerical obstacles when calculating the location of the critical section x_{cr} for the column with step change in cross-sectional parameters and step change in normal force along the members.

1.2. Research Significance, Contributions and Basic Assumptions of Presented New Method

The innovation of the presented research is the numerical method for obtaining the equivalent UGLI imperfection amplitude for metal members susceptible to buckling due to axial forces. The method is

consistent with the Eurocode procedure and allows to obtain the location of critical section x_{cr} and consequently UGLI imperfection amplitude for any case.

Several assumptions are considered in this work:

- (i). The first assumption is that the first buckling mode of the examined structural member is dominant and therefore the effect of the higher buckling modes can be neglected. However, in some cases, the higher buckling modes have to be considered in the design. See suggestions by Agüero et al. [17].
- (ii). The second assumption is that Chladný's method given in [3] remains valid. Quoting Chladný [3], where experimentally established values are unavailable, the amplitude of the equivalent imperfection in the shape of the elastic buckling mode may be determined assuming that the buckling resistance of a structure with axially loaded members shall be equal to the buckling resistance of the equivalent member. The buckling resistance of axially loaded columns is defined in clauses 6.3.1.1–6.3.1.4 [1]. The relative slenderness $\bar{\lambda}$ relates to the critical section. The equivalent member has pinned ends, its cross-section and axial force are the same as in the critical cross-section m of the frame and its length is such that its critical force equals the axial force in the critical cross-section m at the critical loading of the structure. The position of the critical cross-section m is determined by the condition that the utilization U_m , with allowance for the effect of the axial force and bending moments due to imperfections in the critical cross-section m , is greater than the utilization $U(x)$ at all other cross-sections of the verified member or frame structure.
- (iii). The third assumption is that the linear interaction Formula (22) is used to verify the member according to Eurocodes and [25,26].
- (iv). The fourth assumption is that if plastic cross-section resistance is taken into account, a maximum plastic shape factor of 1.25 may be taken into account [4,26]. This means that if the criterion is applied with reference to the plastic resistance, the design value of the moment resistance $M_{c,Rd}$ should be limited to $1.25 M_{el,Rd}$ for both a strong axis and weak axis.
- (v). The fifth assumption accepts the common method for finding the equivalent local bow imperfections given in [1] 5.3.2 (3). For the new generation of Eurocodes, intensive investigations were done to enhance these values, now in draft [4] see 7.3.3.1 (1) [25,26]. In practice, we often deal with members under compression and bending. If the external bending moments are present, the results may be more unfavorable, because a larger part of the cross-section in longitudinal direction may be plastified, thus leading to a stiffness reduction and therefore higher necessary bow imperfections for simplified calculations. If this is neglected, the load carrying capacity may be underestimated. In our cases of centrally loaded members, there is no need to use the fourth and fifth assumptions.

2. Method for Obtaining UGLI Imperfection Amplitude

According to Eurocodes [1,2], Equation (4) shall be used to obtain the value of the initial imperfection amplitude of the equivalent member. The following method is presented when buckling about the y axis takes place. This can be applied in a similar way for buckling about the z axis. The presented method and Chladný's method as well may be used in both cases: when buckling about the y axis and z axis takes place.

The buckling shape has been scaled to have a maximum value of 1.0, i.e., $\max(\eta_{cr,w}) = 1.0$. The meaning of η_0 is the amplitude of UGLI imperfection. The flow chart is shown in Figure 1.

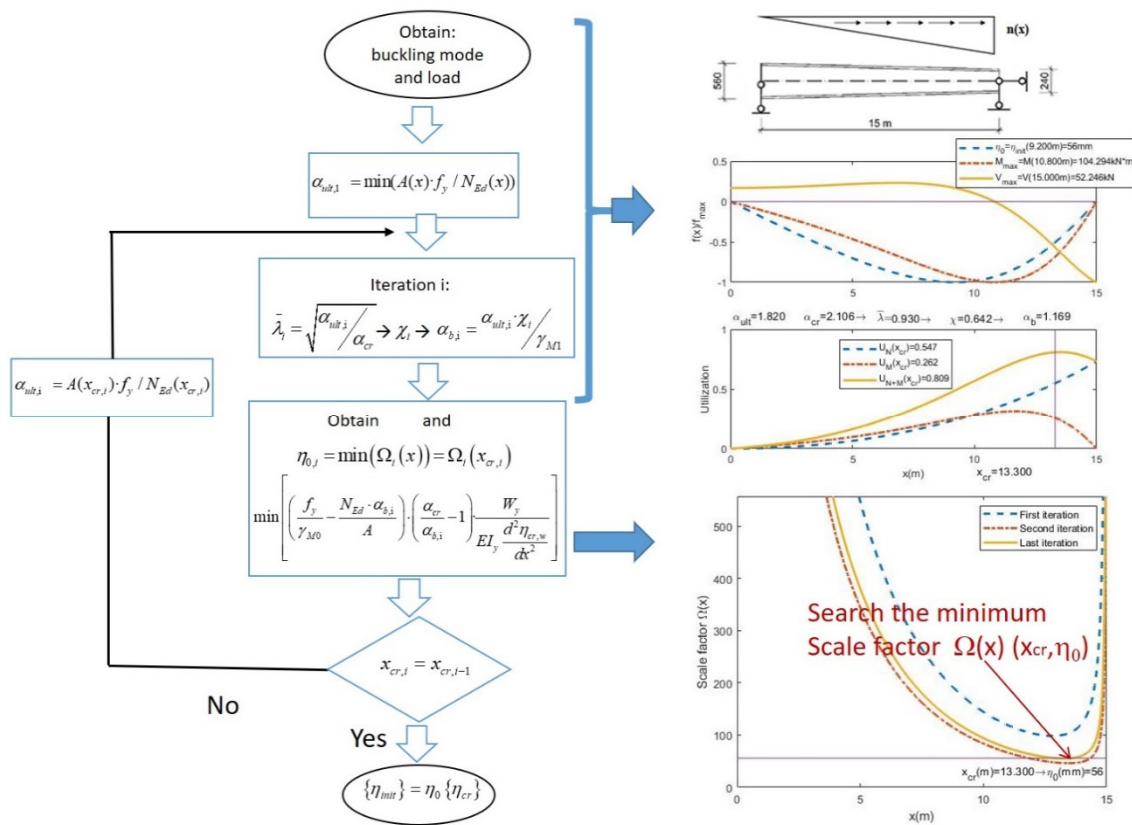


Figure 1. Flow chart describing the procedure on how to obtain the location of the critical section x_{cr} and the unique global and local initial (UGLI) imperfection amplitude η_0 . Illustrative example.

Steps for Obtaining UGLI Imperfection

The first step is obtaining the buckling load α_{cr} and buckling shape $\{\eta_{cr}\}$. They can be computed using finite element method, for instance, according to Trahair [27].

The second step is the calculation of bending moments and stresses related to the buckling mode as follows:

$$M_{y,\eta} = -EI_y \frac{d^2 \eta_{cr,w}}{dx^2} \tag{10}$$

$$\sigma_M = \frac{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}}{W_y} \tag{11}$$

The third step is the first iteration.

Initial guess:

$$\alpha_{ult,1} = \min\left(\frac{A \cdot f_y}{N_{Ed}}\right) \rightarrow \bar{\lambda}_1 = \sqrt{\frac{\alpha_{ult,1}}{\alpha_{cr}}} \rightarrow \chi_1 \rightarrow \alpha_{b,1} = \frac{\alpha_{ult,1} \cdot \chi_1}{\gamma_{M1}} \tag{12}$$

Another initial guess may be considered in the location where the stress calculated according Equation (11) has a maximum value:

The imperfection (scale factor $\Omega_1(x)$) needed at each section in order to reach the cross-section resistance when the buckling load level is reached $\alpha_{b,1}$ is calculated from formula:

$$\frac{N_{Ed} \cdot \alpha_{b,1}}{A} + \frac{\Omega_1(x)}{\left(\frac{\alpha_{cr}}{\alpha_{b,1}} - 1\right)} \frac{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}}{W_y} = \frac{f_y}{\gamma_{M0}} \tag{13}$$

$$\Omega_1(x) = \left(\frac{f_y}{\gamma_{M0}} - \frac{N_{Ed} \cdot \alpha_{b,1}}{A} \right) \left(\frac{\alpha_{cr}}{\alpha_{b,1}} - 1 \right) \frac{W_y}{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}} \tag{14}$$

The minimum of those scale factors is the objective in the first iteration, i.e., $\eta_{0,1}$ takes place at the critical section $x_{cr,1}$:

$$\eta_{0,1} = \min(\Omega_1(x)) = \Omega_1(x_{cr,1}) \tag{15}$$

The fourth step is the second iteration:

$$\alpha_{ult,2} = \frac{A(x_{cr,1}) \cdot f_y(x_{cr,1})}{N_{Ed}(x_{cr,1})} \rightarrow \bar{\lambda}_2 = \sqrt{\frac{\alpha_{ult,2}}{\alpha_{cr}}} \rightarrow \chi_2 \rightarrow \alpha_{b,2} = \frac{\alpha_{ult,2} \cdot \chi_2}{\gamma_{M1}} \tag{16}$$

The imperfection (scale factor $\Omega_2(x)$) needed at each section in order to reach the cross-section resistance when the buckling load level is reached $\alpha_{b,2}$ is calculated as follows:

$$\Omega_2(x) = \left(\frac{f_y}{\gamma_{M0}} - \frac{N_{Ed} \cdot \alpha_{b,2}}{A} \right) \left(\frac{\alpha_{cr}}{\alpha_{b,2}} - 1 \right) \frac{W_y}{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}} \tag{17}$$

$\eta_{0,2}$ is the amplitude at the second iteration. It takes place at the critical section $x_{cr,2}$:

$$\eta_{0,2} = \min(\Omega_2(x)) = \Omega_2(x_{cr,2}) \tag{18}$$

If the critical section is the same as in the previous iteration, the critical section is found and consequently the value of UGLI imperfection amplitude may be calculated. If the critical section is different, then the other iteration is needed starting from the fourth step. Finally, the distribution of UGLI imperfection $\eta_{init}(x)$ is calculated:

$$\{\eta_{init}\} = \eta_0 \{\eta_{cr}\} \tag{19}$$

3. Validation of Presented Procedure

Three Chladný's complex examples [3,7,8] are very interesting in order to validate the presented method. The fourth example shows that differences are negligible between α_b values calculated by the new procedure and α_b values obtained by Marques et al. [28] with the help of Geometrically and Materially Nonlinear Analysis with Imperfections (GMNIA).

Plots of the following parameters are implemented to have a full understanding of the examples:

- The relative UGLI imperfection is plotted along the beam. The values are divided by the maximum value of imperfection in order to obtain the relative UGLI imperfection with maximum value 1.0.
- The distribution of relative bending moments due to UGLI imperfection is plotted along the beam, the values are divided by the maximum bending moment value in order to have a maximum value 1.0:

$$\frac{\eta_0}{(\alpha_{cr} - 1)} EI_y \frac{d^2 \eta_{cr,w}}{dx^2} \tag{20}$$

- The distribution of the relative shear forces due to UGLI imperfection is plotted along the beam. The values are divided by the maximum shear force in order to have a maximum value 1.0:

$$\frac{\eta_0}{(\alpha_{cr} - 1)} EI_y \frac{d^3 \eta_{cr,w}}{dx^3} \tag{21}$$

- The plot of utilization factors U distribution: due to axial forces U_N , due to bending moments U_M and the global utilization factor U_{N+M} . This factor shows how close each section is to its maximum strength and the influence of axial forces and bending moments:

$$U_{N+M} = (U_N) + [U_M] = \left(\frac{N_{Ed}}{A \left(\frac{f_y}{\gamma_{M0}} \right)} \right) + \left[\frac{\eta_0}{(\alpha_{cr} - 1)} \frac{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}}{W_y \left(\frac{f_y}{\gamma_{M0}} \right)} \right] \tag{22}$$

- The value of the scale factor distribution along the member is plotted showing the first, the second and the last iteration. The minimum value of each plot is $\eta_{0,i}$. The maximum value shown is 10 times η_0 :

$$\Omega_i(x) = \left(\frac{f_y}{\gamma_{M0}} - \frac{N_{Ed} \cdot \alpha_{b,i}}{A} \right) \cdot \left(\frac{\alpha_{cr}}{\alpha_{b,i}} - 1 \right) \cdot \frac{W_y}{EI_y \frac{d^2 \eta_{cr,w}}{dx^2}} \tag{23}$$

$$\eta_{0,i} = \min\{\Omega_i(x)\} = \Omega_i(x_{cri}) \tag{24}$$

The values of UGLI imperfection amplitude η_0 , the location of critical section x_{cr} and other values have been compared with Chladný’s values which are given in brackets. Very good agreements were achieved. In the Examples 3 and 4, the influence of the local buckling is neglected. It is beyond the scope of this paper. The authors have solutions of these examples which also take into account local buckling. They will be published with all the necessary details in another paper.

Example 1. The Example 1 is described in Figure 2. It is the scheme of the bottom flange of the continuous composite steel and concrete road bridge. More details about the bridge may be found in Section 5.4 [7]. Steel grade S355 and $\gamma_{M1} = 1.1$ are taken into account.

A [m ²]	0.031	0.025	0.025	0.031
I_z [m ⁴]	0.00072	0.00054	0.00054	0.00072
W_z [m ³]	0.0024	0.0018	0.0018	0.0024

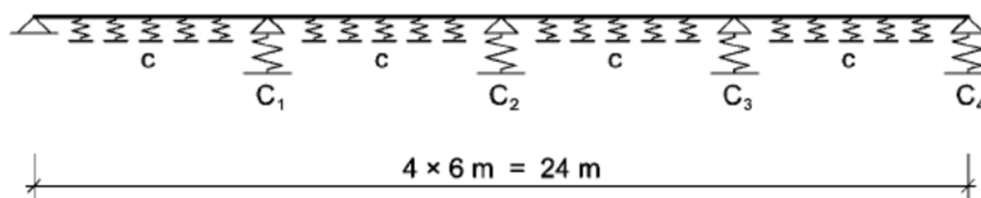


Figure 2. Example 1: description taken from [7], Section 5.4.

The bottom flange of the steel cross-section in compression is subjected to lateral buckling. It may be analyzed by modeling the elements as a column subjected to the compression force N_{Ed} and supported by continuous and discrete elastic restraints modeled as springs (Figure 2). The springs stiffnesses are $C1 = 10 \text{ kN}\cdot\text{mm}^{-1}$, $C2 = C3 = C4 = 4 \text{ kN}\cdot\text{mm}^{-1}$. The elastic restraint due to the cross-section web is $c = 5.863 \text{ kN}\cdot\text{m}^{-2}$. The axial force distribution is given by formula $N_{Ed}(x) = -9000 - 1062x - 17.49x^2$ (kN).

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to UGLI imperfection and utilization factors $U(x)$ are plotted in Figure 3. The critical section takes place after several iterations at $x_{cr} = 1.44 \text{ m}$. The significant step in shear force distribution V is located where the column is supported by the spring with the greatest stiffness. The scale factor is plotted in Figure 4. The minimum value at the last iteration gives the amplitude of the imperfection $\eta_0 = 39 \text{ mm}$. Chladný’s values in the brackets confirm the very good agreements of the results (Figure 3). According to the direct method [23] the critical section takes place where the second order effects have a maximum. This method gives a value of amplitude $\eta_0 = 29 \text{ mm}$ which differs from 39 mm a lot. In all the other examples, the direct method gave similar results as Eurocode’s method.

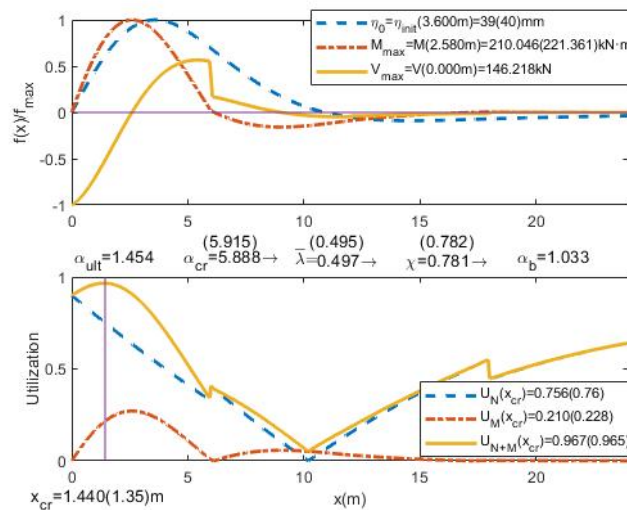


Figure 3. Example 1: UGLI imperfection η_{init} , bending moment M , shear force V , utilization factor U . Values in brackets are from [7].

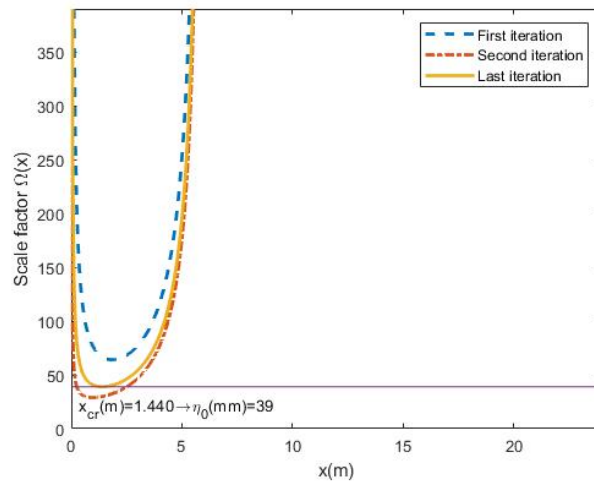


Figure 4. Example 1: the scale factor $\Omega(x)$ for utilization factor $U_{N+M} = 1.0$ in bay No.1. In other bays, the scale factor is greater than 10 times the minimum value.

The value $\eta_0 = 39$ mm given in Figure 4 was obtained after seven necessary iterations. Table 1 shows all the details of the iterations.

Table 1. Example 1: key values for each iteration.

Iteration	α_{ult}	$\bar{\lambda}$	χ	α_b	$x_{cr,i}$	$\frac{\eta_{0,i}}{\eta_0}$
1	1.2228	0.4557	0.8106	0.9011	1.9200	1.6435
2	1.5532	0.5136	0.7698	1.0870	1.0200	0.7413
3	1.3762	0.4835	0.7910	0.9896	1.6800	1.2012
4	1.5023	0.5051	0.7757	1.0594	1.3200	0.8779
5	1.4311	0.4930	0.7842	1.0203	1.5000	1.0586
6	1.4660	0.4990	0.7800	1.0396	1.4400	0.9702
7	1.4542	0.4970	0.7814	1.0331	1.4400	1.0000

Example 2. The Example 2 is described in Figure 5. It is a schema of the upper chord of the steel railway half-through truss bridge. More details about the bridge may be found in Section 5.5 [7]. Steel grade S235 and $\gamma_{M1} = 1.1$ are taken into account.

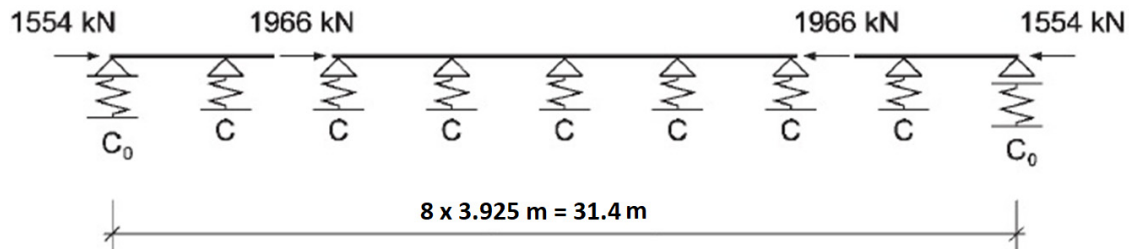


Figure 5. Example 2: description taken from [7] Section 5.5.

The truss chord under compression that is subjected to the lateral buckling may be analyzed by modeling the elements as a column subjected to the compression force N_{Ed} and supported by discrete elastic restraints modeled as springs. The springs stiffnesses are $C_0 = 11 \text{ MN/m}$ and $C = 5.068 \text{ MN/m}$. Input values for the iterative procedure [7] are given in Figure 5. In the two edge fields on both sides of the truss chord: $A = 0.01909 \text{ m}^2$, $I = 96.4 \times 10^{-6} \text{ m}^4$, $W = 459 \text{ cm}^3$, in the four middle fields of the truss chord: $A = 0.02898 \text{ m}^2$, $I = 177.525 \times 10^{-6} \text{ m}^4$, $W = 845 \text{ cm}^3$. The loading is defined in Figure 5.

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to the imperfection and the utilization factors $U(x)$ are plotted in Figure 6. The steps in the shear force distribution are due to steps in the axial forces and in spring supports. The critical section takes place at $x_{cr} = 13.15$ and 18.25 m simultaneously due to the symmetry of the problem. The scale factor is plotted in Figure 7. The scale factor minimum led to UGLI imperfection amplitude $\eta_0 = 9 \text{ mm}$. The significant values are shown in Table 2. Only one iteration was needed. The comparison with Chladný’s values given in brackets shows very good agreement.

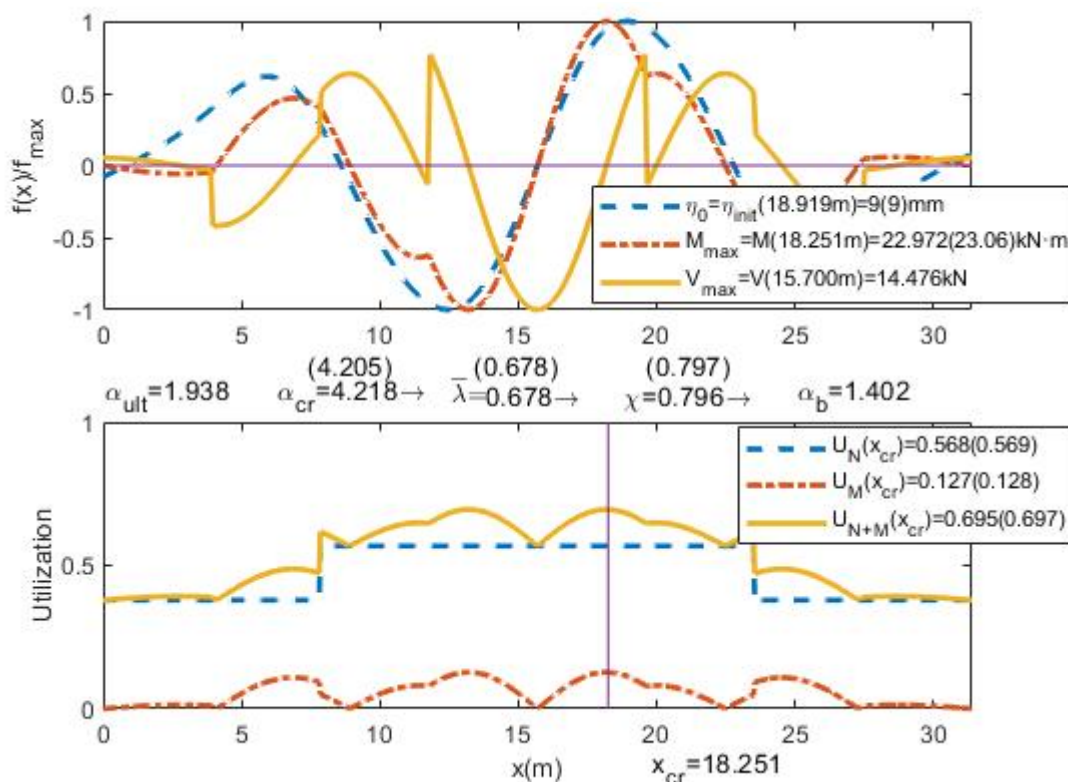


Figure 6. Example 2: UGLI imperfection η_{init} , bending moment M , shear force V , and utilization factor U . Values in brackets are from [7].

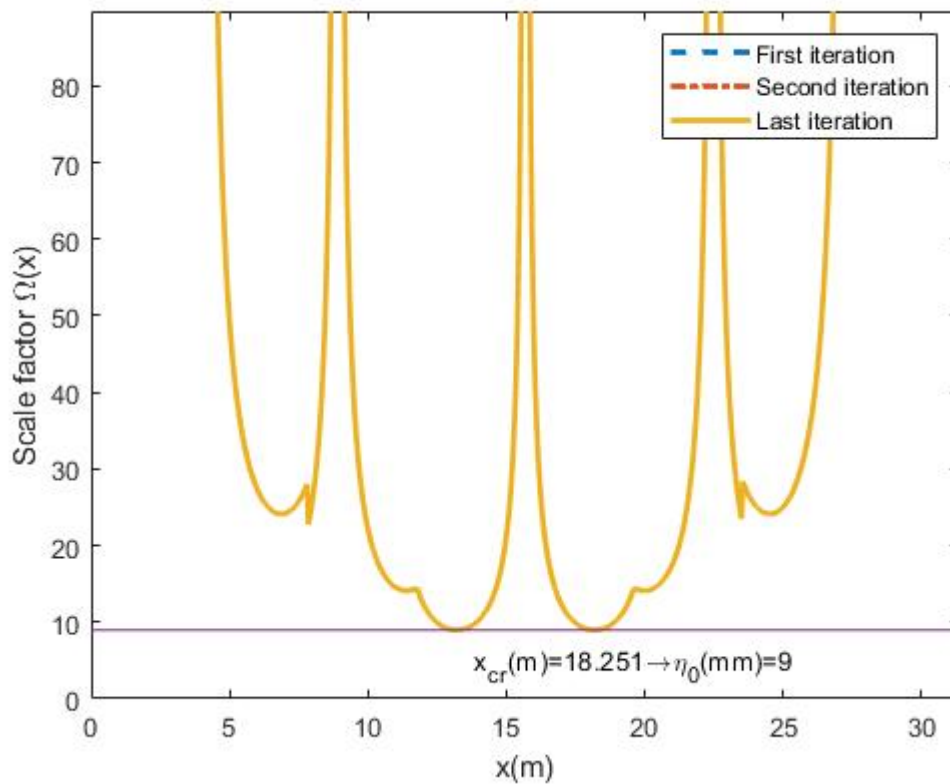


Figure 7. Example 2: the scale factor $\Omega(x)$. Utilization factor $U_{N+M} = 1.0$.

Table 2. Example 2: key values for each iteration.

Iteration	α_{ult}	$\bar{\lambda}$	χ	α_b	$x_{cr,i}$	$\frac{\eta_{0,i}}{\eta_0}$
1	1.9376	0.6777	0.7961	1.4023	13.2273	1

Chladný used the method also for more complicated cases, where the simplified method is not applicable, e.g., for the Maria Valeria Bridge across the Danube, Štúrovo–Esztergom. See Note 5 in [7] referring to [29]. According to the direct method, the same results are obtained.

Example 3. The Example 3 is described in Figure 8, with steel grade S355 and $\gamma_{M1} = 1.0$

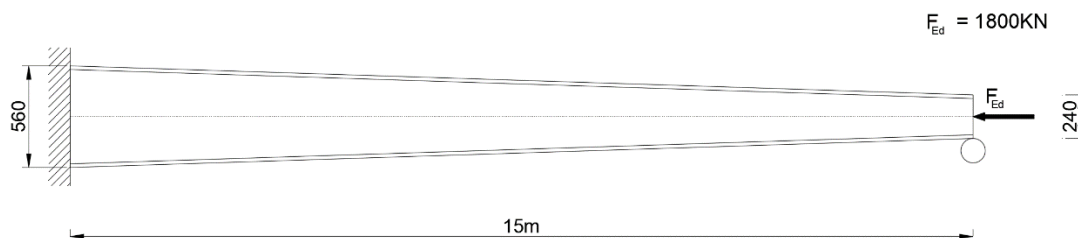


Figure 8. Example 3: description taken from [9,10] and [7] Section 5.2.

The member is fabricated from an IPE 400 section. It was cut in two pieces and consequently welded together in such a way that the depths of its cross-section at the ends are 560 and 240 mm.

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to the imperfection and the utilization factors $U(x)$ are plotted in Figure 9. The critical section is found at $x_{cr} = 12$ m. The scale factor minimum led to UGLI imperfection amplitude $\eta_0 = 36.31$ mm (Figure 10). The significant values for all iterations are shown in Table 3. Only two iterations were needed.

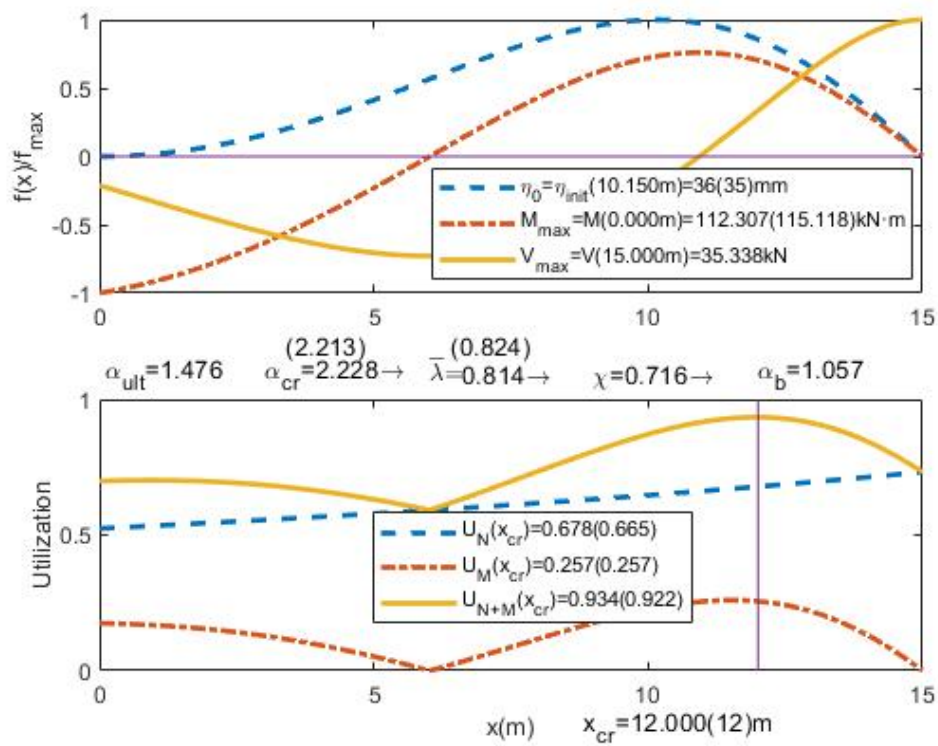


Figure 9. Example 3: UGLI imperfection η_{init} , bending moment M , shear force V , and utilization factor U . Values in brackets are from [7,10].

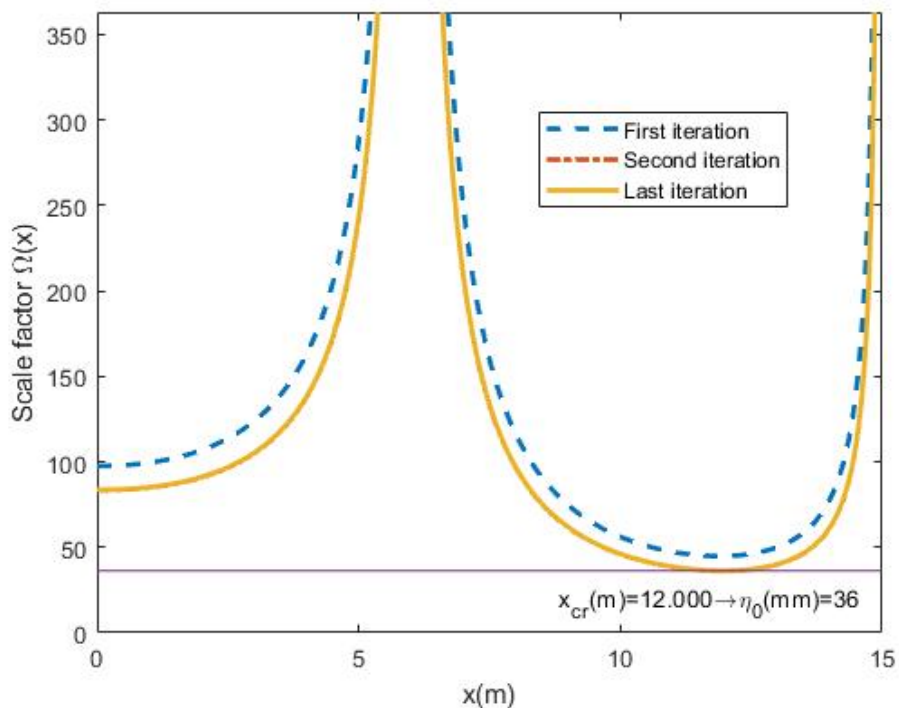


Figure 10. Example 3: the scale factor $\Omega(x)$. Utilization factor $U_{N+M} = 1.0$.

Table 3. Example 3: key values for each iteration.

Iteration	α_{ult}	$\bar{\lambda}$	χ	α_b	$x_{cr,i}$	$\frac{\eta_{0,i}}{\eta_0}$
1	1.3674	0.7834	0.7346	1.0045	11.95	1.2343
2	1.4777	0.8144	0.7155	1.0574	12	1

Chladný’s values are in brackets and they confirm the very good agreement of the results (Figure 9).

The direct method gives the value of imperfection amplitude $\eta_0 = 34.56$ mm which approximately equals value 36 mm (Figure 10).

Example 4. (a) The Example 4 was published by Marques et al. [28], with steel grade 235, and $\gamma_{M1} = 1.0$. It is a tapered column similar to that in Figure 8 but made from IPE 200, with different boundary conditions. The linearly varying height is defined by the taper ratio of $h_{max}/h_{min} = 3$. It is simply supported at both ends, $L = 12.9$ m. The member is restrained against out-of-plane buckling. The compression force $N_{Ed} = 500$ kN was applied. The computed load factor $\alpha_b = 1.004$ with GMNIA. The method proposed by the authors leads to $\alpha_b = 0.991$. The difference is -1.29% .

In Example 4(a), GMNIA was performed by Marques et al. [28] using ABAQUS software taking into account a geometrical imperfection with the shape of the buckling mode and an amplitude of $L/1000$, also considering the residual stresses of a welded cross-section.

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to the imperfection and the utilization factors $U(x)$ are plotted in Figure 11. The critical section is found at $x_{cr} = 10.2$ m. The scale factor minimum led to UGLI imperfection amplitude $\eta_0 = 28.04$ mm (Figure 12). The significant values for all iterations are shown in Table 4. Three iterations were needed.

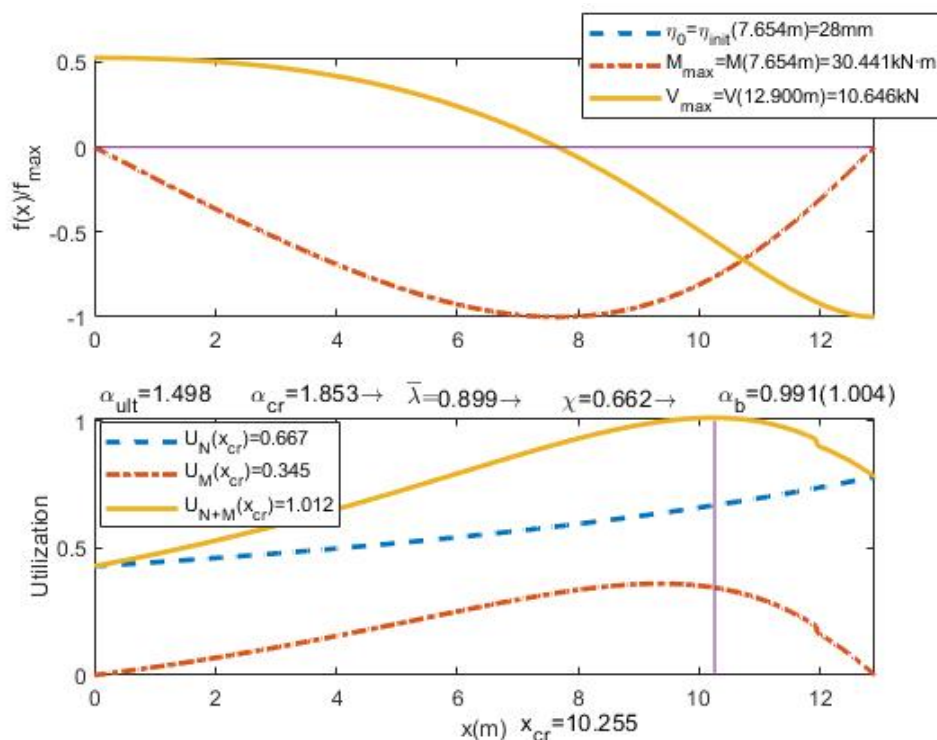


Figure 11. Example 4(a): UGLI imperfection η_{init} , bending moment M , shear force V , utilization factor U . Value (1.004) is obtained by Geometrically and Materially Nonlinear Analysis with Imperfections (GMNIA).

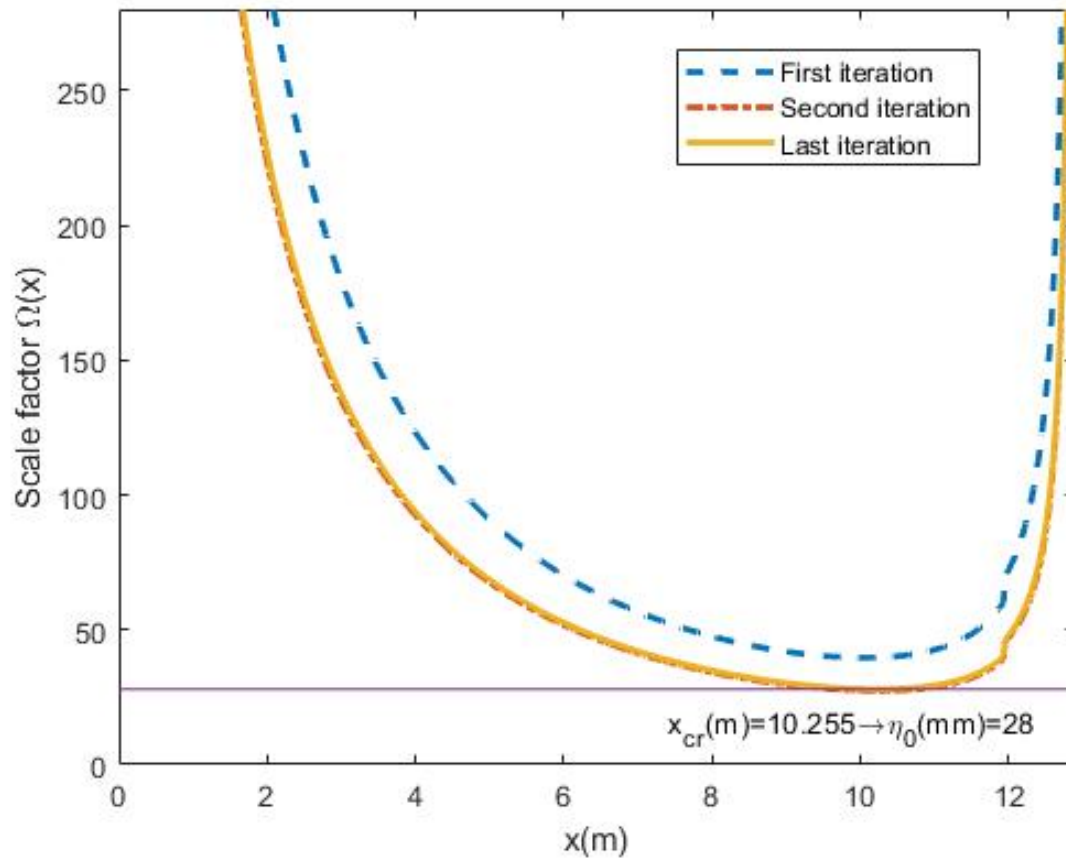


Figure 12. Example 4(a): the scale factor $\Omega(x)$. Utilization factor $U_{N+M} = 1.0$.

Table 4. Example 4: key values for each iteration.

Iteration	α_{ult}	$\bar{\lambda}$	χ	α_b	$x_{cr,i}$	$\frac{\eta_{0,i}}{\eta_0}$
1	1.2828	0.8319	0.7046	0.9038	10.0878	1.4135
2	1.5123	0.9033	0.6591	0.9967	10.268	0.9775
3	1.4975	0.8989	0.6619	0.9912	10.268	1

In the Example 4(a), the maximum moment $M_{max}(x = 7.654 \text{ m})$ can be obtained from Equation (25), which serves to verify the value $M_{max}(x = 7.654 \text{ m}) = 30.441 \text{ kNm}$ given in Figure 11:

$$M_{max} = \frac{F_{Ed} \cdot \eta_0}{\left(1 - \frac{1}{\alpha_{cr}}\right)} = \frac{500 \cdot 28 / 1000}{\left(1 - \frac{1}{1.852}\right)} \approx 30.441 \text{ kN} \cdot \text{m} \quad (25)$$

According to the direct method, the imperfection amplitude is $\eta_0 = 25.86 \text{ mm}$. It is a similar result.

(b) In Figure 13, the α_b values calculated for different slenderness at the critical section are compared with the GMNIA results of Marques et al. [28]. The maximum difference is -8% for relative slenderness $\bar{\lambda} = 0.773$ (buckling curve b). This analysis is performed for a hot-rolled tapered column $b = 100 \text{ mm}$, $t_f = t_w = 10 \text{ mm}$, $h_{min} = 100 \text{ mm}$ and $h_{max} = 400 \text{ mm}$, steel S235 with a linearly varying height, and a simply supported member. The uniform axial force $N_{Ed} = N_{pl,Rd,min} = 658 \text{ kN}$.

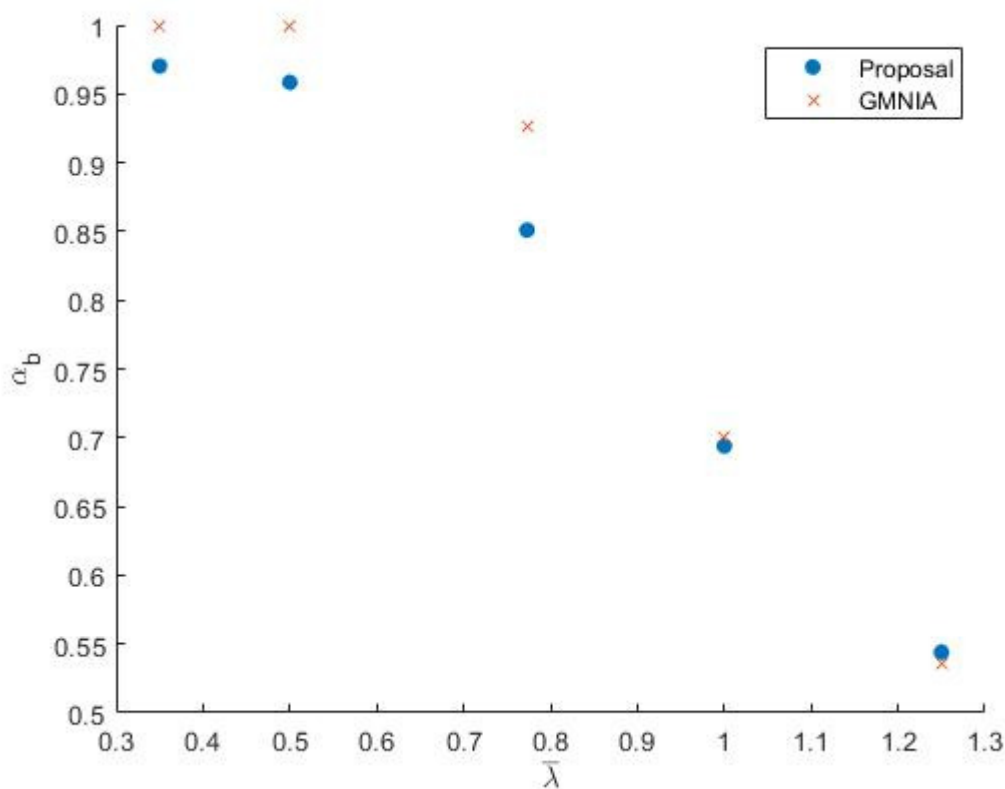


Figure 13. Example 4(b): comparison of α_b values calculated for the different slenderness obtained by GMNIA and by the proposed procedure of the authors.

In Example 4(b), GMNIA was performed by Marques et al. [28] using ABAQUS software taking into account a geometrical imperfection with the shape of the buckling mode and amplitude of $L/1000$, also considering the residual stresses of a hot-rolled cross-section.

4. Application to the Case Where Problems Were Found to Obtain the Critical Section and to the Member Made of Aluminum Alloy

Following the other examples, these are solved where the step changes in the cross-section and in the axial forces create numerical problems to find the critical section x_{cr} . These are pure academic problems found in the literature, however, there is the need to show that they may be solved.

Example 5. A Ph.D. student wrote in [24] about the obstacles of the method because he was not able to obtain the location of the critical section due to the never-ending cycle by going from finding the “critical section” to the other one and back. The Example 5 shows that proposed method has no such problem. A cantilever column with a step change in the cross-sectional parameters and axial forces is investigated (Figure 14). The material used in [24] is the steel grade S355. The safety factor $\gamma_{M1} = 1.0$.

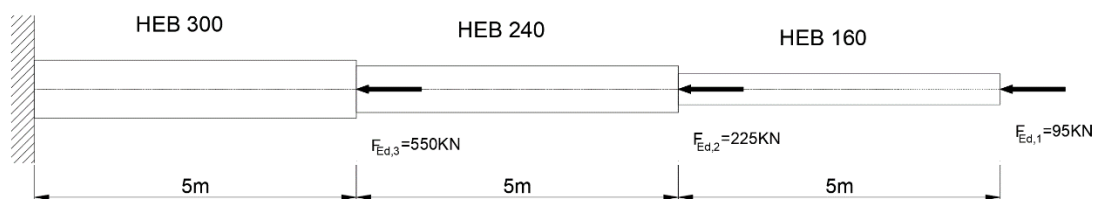


Figure 14. Example 5: description taken from [24].

The position of the section x_U where the greatest utilization factor $U_{N+M,max} < 1.0$ takes place may depend on the load level. The position of the section x_U may differ from the location of the critical section x_{cr} defined in the nomenclature. This is case in Figures 15 and 16. In the rare cases, as it is in Example 5 (Figure 14), the location of the critical section x_{cr} may be found in accordance with its definition given in nomenclature only if $U_{N+M,max} = 1.0$ (Figure 17). The cases in which the almost identical values of $U_{N+M,max}$ appear in the same time in more than one section are very rare. Example 5 is a such rare case. The comparison of the values $U_{N+M,max}(x_{cr} = 10\text{ m}) = 1.0$ with $U_{N+M}(x = 0\text{ m}) = 0.9378$ is shown in Figure 17. These values would be closer if steel S235 would be used instead of steel S355: 1.0 and 0.9965, and the value of η_0 would be decreased from 101 to 77 mm.

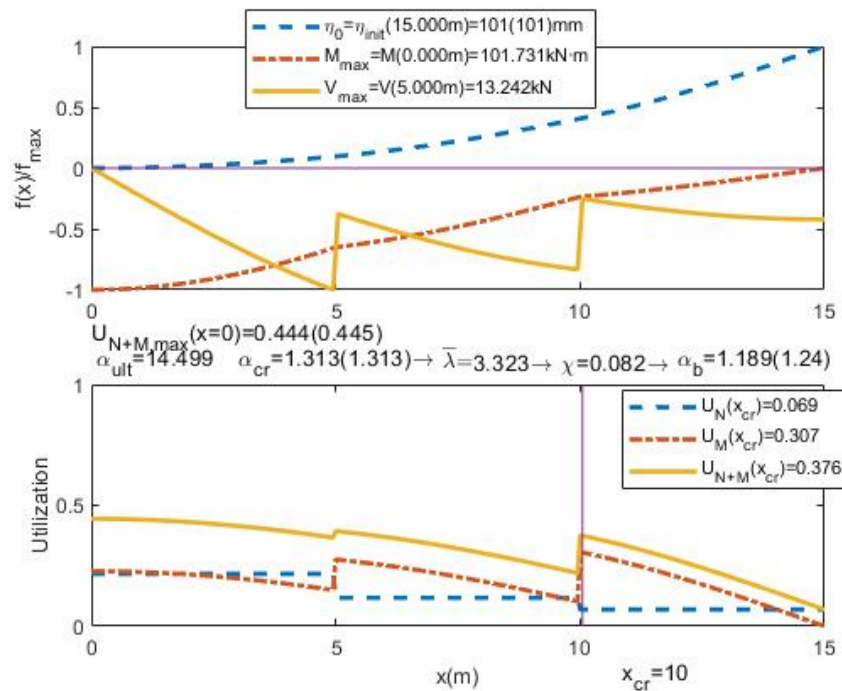


Figure 15. Example 5: UGLI imperfection η_{init} , bending moment M , shear force V , and utilization factor. Values in brackets are taken from [24]. Value (1.24) was obtained by GMNIA.

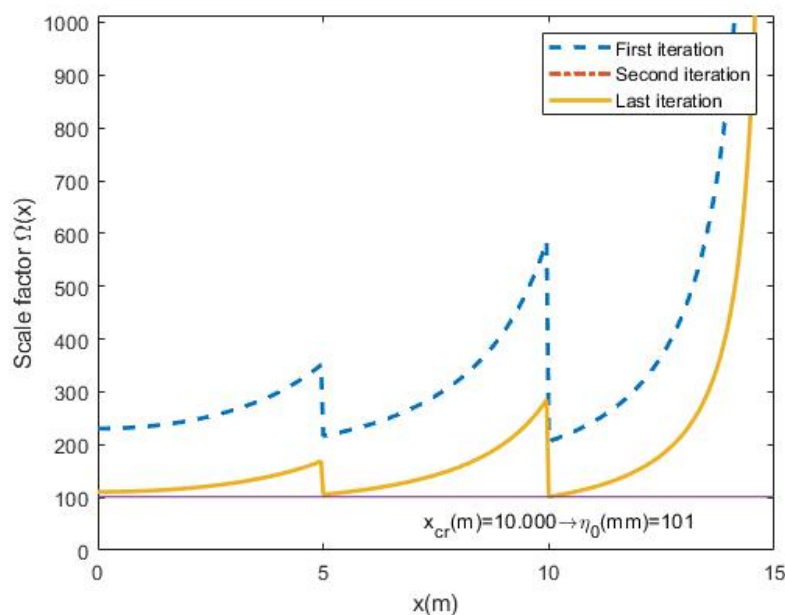


Figure 16. Example 5: the scale factor $\Omega(x)$. Utilization factor $U_{N+M} = 1.0$.

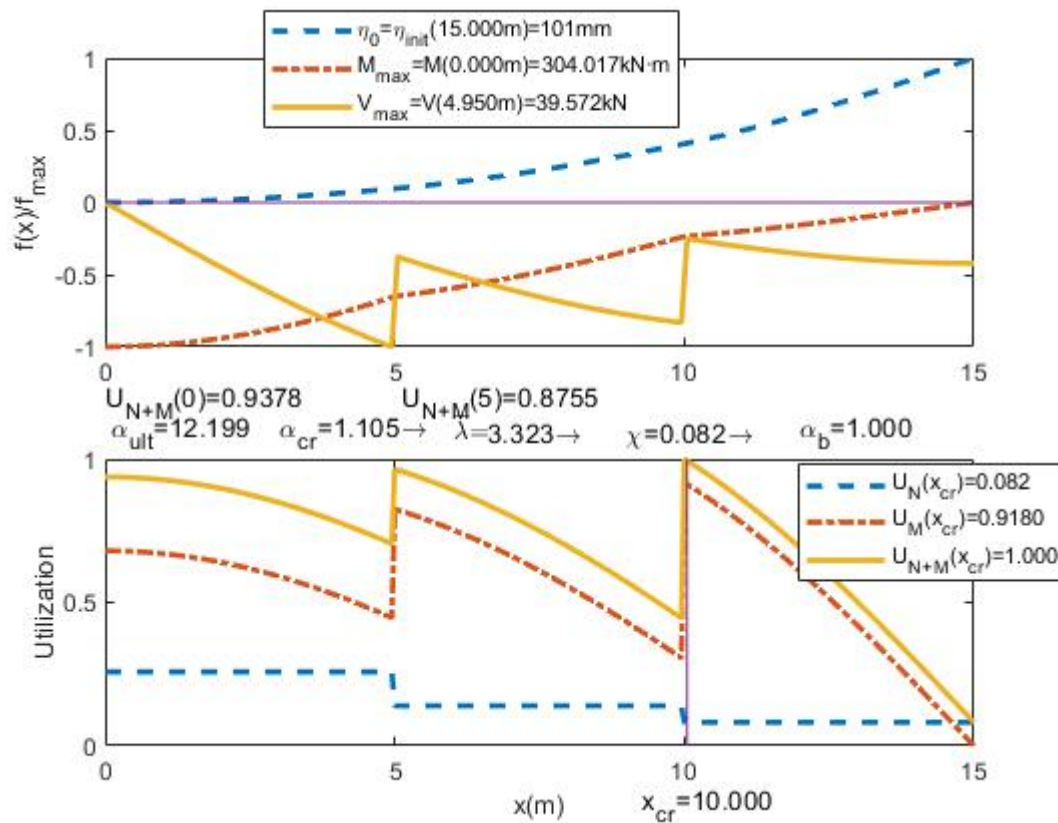


Figure 17. Example 5: UGLI imperfection η_{init} , bending moment M , shear force V , and utilization factor $U_{N+M} = 1.0$.

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to the imperfection and the utilization factors $U(x)$ are plotted in Figure 15. The resulting critical section is $x_{cr} = 10$ m. The minimum value of the scale factor led in the second and the last iteration to UGLI imperfection amplitude $\eta_0 = 101$ mm (Figure 16). The significant values for all iterations are shown in Table 5. With the procedure of the authors presented in this article, there is no problem for solving this example. The utilization at the buckling load level is shown in Figure 17. The buckling resistance $\alpha_b = 1.189$ was calculated according to authors' procedure and checked with GMNIA value $\alpha_b = 1.24$ (Figure 15).

Table 5. Example 5: key values for each iteration.

Iteration	α_{ult}	$\bar{\lambda}$	χ	α_b	$x_{cr,i}$	$\frac{\eta_{0,i}}{\eta_0}$
1	3.0523	1.5139	0.3372	1.0292	0.0	2.0485
2	9.5979	2.6845	0.1224	1.1749	10	1

In Example 5, GMNIA was performed by the authors (Figure 15) using the Open System for Earthquake Engineering Simulation (OPENSEES) software taking into account a geometrical imperfection with the shape of the buckling mode and amplitude of $L/500$ ($L_{cr}/1000$), also considering the residual stresses of a hot-rolled cross-section.

In Example 5, the maximum moment $M_{max}(x = 0$ m) can be obtained from Equation (26), which serves to verify the value $M_{max}(x = 0$ m) = 101.731 kNm given in Figure 15:

$$\begin{aligned}
 M_{max} &= \frac{F_{Ed1} \cdot \eta_{init}^x = 15}{\left(1 - \frac{1}{\alpha_{cr}}\right)} + \frac{F_{Ed2} \cdot \eta_{init}^x = 10}{\left(1 - \frac{1}{\alpha_{cr}}\right)} + \frac{F_{Ed3} \cdot \eta_{init}^x = 5}{\left(1 - \frac{1}{\alpha_{cr}}\right)} = \\
 &= \frac{101/1000}{\left(1 - \frac{1}{1.313}\right)} (95 + 0.41 \cdot 225 + 0.095 \cdot 550) \approx 101.731 \text{ kN} \cdot \text{m}
 \end{aligned}
 \tag{26}$$

If the value of the force of $F_{Ed,2} = 225$ kN (Figure 14) changes, the values of UGLI imperfection amplitude and the critical section position x_{cr} are given in Table 6. There is no problem to find the location of the critical section x_{cr} . From Table 6, it can be concluded that UGLI imperfection amplitude has the step change when the critical section position changes from $x_{cr} = 10$ to 5 m. It changes from $\eta_0 = 101$ to 145 mm for steel grade S355.

Table 6. Example 5: influence of the value of $F_{Ed,2}$ and the steel grade. Values in brackets are from [24].

Steel	$F_{Ed,2}$ (kN)	100	125	150	175	200	(225)	250	275	300	325
S 235	η_0 (mm)	68.8	71.1	73.4	75.7	78	77.7	113	115.5	115.5	115.3
S 275	η_0 (mm)	75	77.4	80	82.4	85	87.4	126.1	126	126	125.8
S 355	η_0 (mm)	86	88.8	91.6	94.5	97.4	101 (100.9)	145	145	144.9	144.7
	x_{cr} (m)	10	10	10	10	10	10 (10)	5	5	5	5

In Table 7, the buckling resistance α_b calculated according to the authors' procedure is checked with the GMNIA value for steel grades 235, 275 and 355.

Table 7. Example 5: buckling resistance α_b calculated according to the authors' procedure and with GMNIA.

Steel	$F_{Ed,2}$ (kN)	100	125	150	175	200	225	250	275	300	325
S 235	α_b (Authors)	1.381	1.333	1.287	1.242	1.2	1.159	1.065	1.023	0.985	0.949
	α_b (GMNIA)	1.455	1.396	1.346	1.294	1.246	1.193	1.138	1.09	1.07	1.03
S 275	α_b (Authors)	1.397	1.348	1.301	1.256	1.213	1.171	1.082	1.04	1.001	0.965
	α_b (GMNIA)	1.46	1.41	1.37	1.318	1.26	1.22	1.15	1.11	1.07	1.04
S 355	α_b (Authors)	1.42	1.37	1.322	1.275	1.231	1.189	1.148	1.064	1.025	0.988
	α_b (GMNIA)	1.48	1.45	1.39	1.34	1.29	1.24	1.21	1.169	1.121	1.08

In Brodniansky's work [24], where for UGLI imperfection the incorrect name EUGLI is used, the critical section x_{cr} could not be found using the utilization factor matrix due to numerical issues. According to the procedure proposed in this work, the critical section x_{cr} is possible to find. It may be concluded that Chladný's method has no obstacles.

Example 6. Höglund solved in [30] by equivalent member method given in [2] the member made of aluminum alloy EN AW-6005A-T6, buckling class A, with a yield strength $f_o = 215$ MPa, ultimate strength $f_u = 260$ MPa and $\gamma_{M1} = 1.1$. He also took into account welds located in the fixed end and in the place where a member changes his cross-section.

We took from example 6.11 [30] only the geometry $L = 5$ m and loading. The purpose was to show that also in example 6.11 [30], which is similar to Example 6, the presented procedure may obtain desired results without problems. Our member (Figure 18) is made of the aluminum alloy EN AW-6061-T6, buckling class A, $E = 70,000$ MPa, $f_o = 240$ MPa, $f_u = 260$ MPa (Table 3.2b [2]), $\gamma_{M1} = 1.1$. The section properties of EP (extruded profile) are $A_1 = 38.8$ cm², $I_1 = 1673$ cm⁴, $W_1 = 220$ cm³ and $A_2 = 91$ cm², $I_2 = 8091$ cm⁴, $W_2 = 736$ cm³.

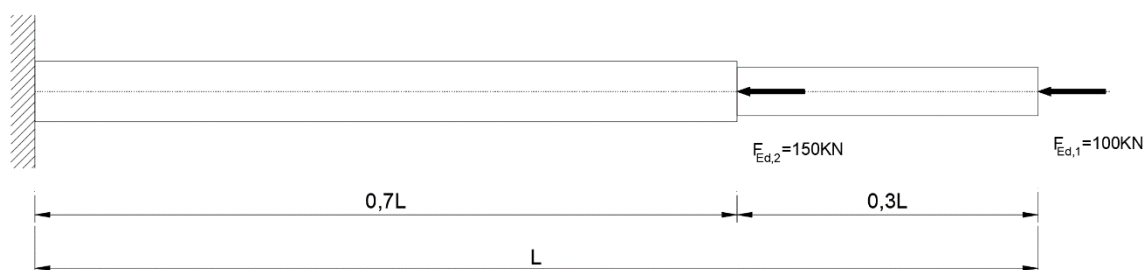


Figure 18. Example 6: description of the member made of aluminum alloy taken from [30] example 6.11.

Nothing may be compared with Höglund’s example 6.11, except the values of buckling lengths, which are the same.

The distributions of UGLI imperfection, bending moment $M(x)$ and shear force $V(x)$ due to the imperfection and the utilization factors $U(x)$ are plotted in Figure 19. The critical section takes place at $x_{cr} = 0$ m. The minimum value of scale factor led in the last iteration to UGLI imperfection amplitude $\eta_0 = 56$ mm (Figure 20). The influence of the welds is beyond the scope of this paper. The authors intend to take it into account in another paper.

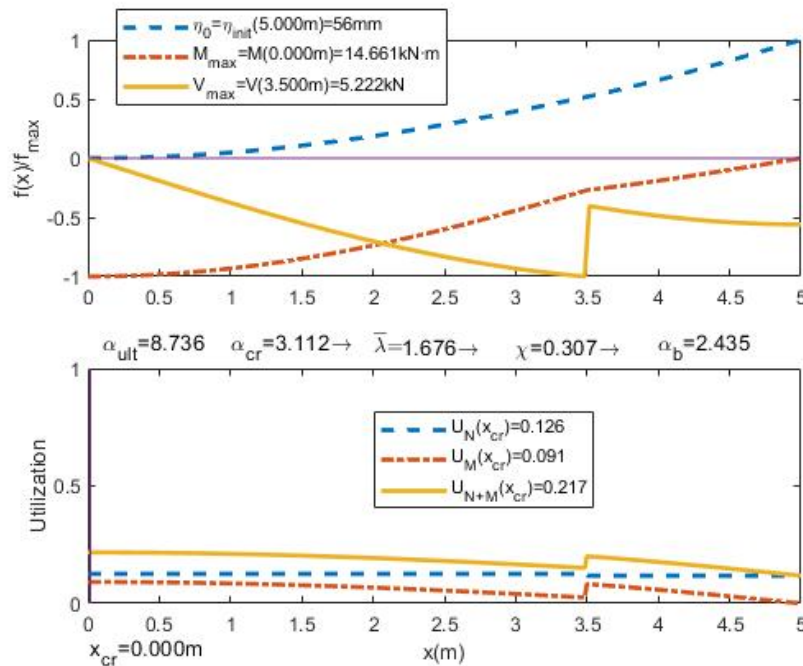


Figure 19. Example 6: UGLI imperfection η_{init} , bending moment M , shear force V , and utilization factor U .

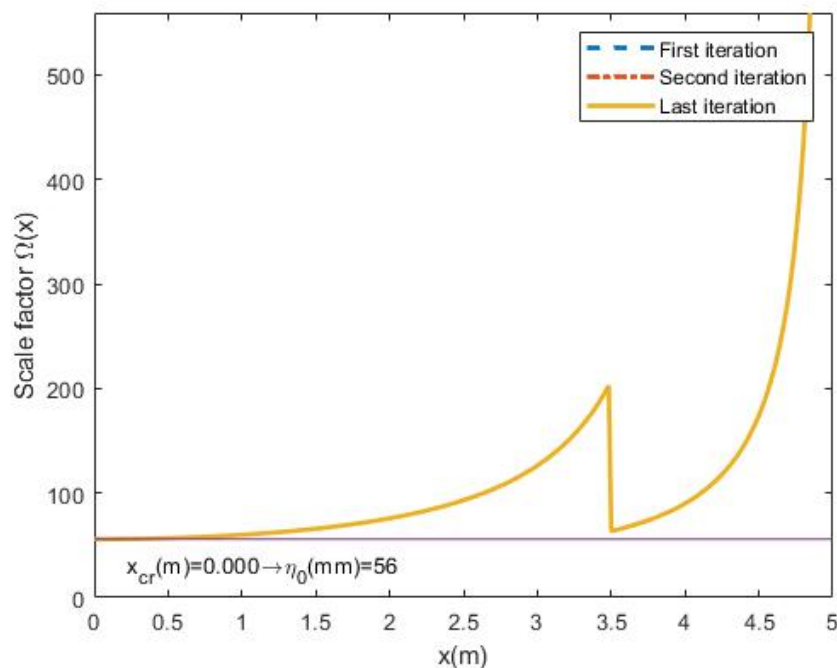


Figure 20. Example 6: the scale factor $\Omega(x)$. Utilization factor $U_{N+M} = 1.0$.

5. Conclusions

A new more general procedure is presented based on Chladný's method, which was accepted by CEN/TC 250 working groups for Eurocodes [1,2,4,5]. It enables to obtain the geometric equivalent UGLI imperfection $\eta_{init}(x)$, the critical section x_{cr} and UGLI imperfection amplitude η_0 for complex slender metal structures. This procedure has been validated by the recalculation of several examples of steel members published by Chladný [7], Papp [23] and Marques et al. [28]. The comparison of the results showed very good agreement with the results of other authors and results of GMNIA. It is shown that the presented procedure is also able to solve case [24] in which the critical section x_{cr} could not be found. The example of the member made of aluminum alloy investigated by Höglund [30] is also solved without problem.

The paper provides information about: (i) the history of method development used in Eurocodes and (ii) differences between the current Eurocodes [1,2] and their newest working drafts [4,5]. The method is described in detail in such a way that the users of Eurocodes will also be able to use it for complex frame structures made of steel or aluminum alloys.

The topics which are beyond scope of this paper are namely the influence of: (i) partial safety factor γ_{M1} ; (ii) local buckling at steel structures; and (iii) transverse or longitudinal welds at structures made of aluminum alloys, will be investigated in the next papers. The results of direct method are very close to the ones obtained by the authors' method, except the results in the Example 1. Further studies should be performed to determine if the direct method could be an alternative to the current method proposed in the Eurocodes.

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Nomenclature

α is the imperfection factor related to the flexural buckling curve (Tables 6.1 and 6.2 of EN 1993-1-1 [1]; Tables 3.2 and 6.6 in EN 1999-1-1 [2]). α_{LT} is the imperfection factor for lateral torsional buckling related to buckling curve (Tables 6.3, 6.4 and 6.5 in EN 1993-1-1 [1]; 6.3.2.2 in EN 1999-1-1 [2]). α_{cr} is the minimum load amplifier for the axial force configuration in members to reach the elastic critical buckling load (5.2.1(3) in [1]; 5.2.1(3) in [2]). α_{ult} is the amplifier for the load of members to reach the characteristic resistance of the critical cross-section (6.3.4(2) in [1], where more the convenient symbol $\alpha_{ult,k}$ is used; [2] does not know such quantity and symbol). α_b is the load amplifier of members to reach the ultimate buckling load (see Equations (12) and (16); the symbol is not used in [1,2]). γ_{M0} is the partial safety factor for the resistance of cross-section whatever the class is (6.1 in [1]; [2] does not know such quantity and symbol). γ_{M1} is the partial safety factor for the resistance of members to instability assessed by member checks (6.1 in [1] and 6.1.3; Table 6.1 in [2]). χ is the reduction factor for the relevant buckling curve (6.3.1.2 in [1]; 6.3.1.2 in [2]). $\bar{\lambda}$ is the relative slenderness (6.3.1.2 in [1]; 6.3.1.2 in [2]). $\bar{\lambda}_0$ is the plateau length of buckling curves (for steel [1]: 0.2; for the aluminum alloy [2]: 0.1 for buckling class A and 0.0 for buckling class B). $\bar{\lambda}_{LT}$ is the relative slenderness for lateral torsional buckling (6.3.2.2 and 6.3.2.3 in [1]; 6.3.2.2 in [2]). $\bar{\lambda}_{TF}$ is the relative slenderness for torsional flexural buckling. $\bar{\lambda}_{LT,0}$ is the plateau length of lateral torsional buckling curves. β is the correction factor for lateral torsional buckling curves (6.3.2.3 in [1]; [2] does not know such quantity and symbol). η_0 is the amplitude of UGLI imperfection. $\{\eta_{init}\}$ is UGLI imperfection in the shape of elastic critical buckling mode. $\{\eta_{cr}\}$ is the shape of elastic critical buckling mode. $\eta_{cr,w}$ is the displacement of the buckling shape component perpendicular to axis y along the x axis. $\eta_{cr,v}$ is the displacement of the buckling shape component perpendicular to axis z along the x axis. $\eta_{cr,\theta x}$ is the torsional rotation of the buckling shape component about shear center axis along the x axis. A is the cross-sectional area. E is the modulus of elasticity (210,000 MPa for steel [1]; 70,000 MPa for aluminum alloy [2]). $e_{0,d}$ ($e_{0,k}$) is the design (characteristic) value of the initial imperfection amplitude of the equivalent member used in the calculation of UGLI imperfection amplitude. It is given in 5.3.2(11) [1,2] or in 7.3.6(1) [4] or in 7.3.2(11) in [5]. The indices d and k are sometimes omitted.

e_0 is the local bow imperfection given in Table 5.1 [1,2] or in 7.3.3.1(1) [4] or in 7.3.2(3)b) [5]. It is used when performing second order analysis including member imperfections related to flexural buckling. It is not used in UGLI imperfection method and must not be commuted with the above amplitude. I_y, I_z are the second moments of area with respect to the y, z axes. I_w is the warping constant. N_{cr} is the elastic critical force of the relevant buckling mode based on the gross cross-section properties. N_{Ed} is the design value of normal force. N_{Rk} is the characteristic resistance of normal force in the critical section (depending on Class 1, 2 and 3 cross-section). M_{Rk} is the characteristic resistance of bending moment in the critical section (depending on Class 1, 2 and 3 cross-section). W_z is the section modulus about z axis (depending on Class 1, 2 and 3 cross-section). W_y is the section modulus about y axis (depending on Class 1, 2 and 3 cross-section). W_B is the warping section modulus. x_{cr} is the critical section where the utilization factor is greater than at all the other sections.

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